



SMR.1771 - 9

**Conference and Euromech Colloquium #480**

**on**

**High Rayleigh Number Convection**

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**Local heat transport analysis based  
on DNS/LES of turbulent RBC  
in wide containers**

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DLR-Institute of Aerodynamics & Flow Technology  
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Germany

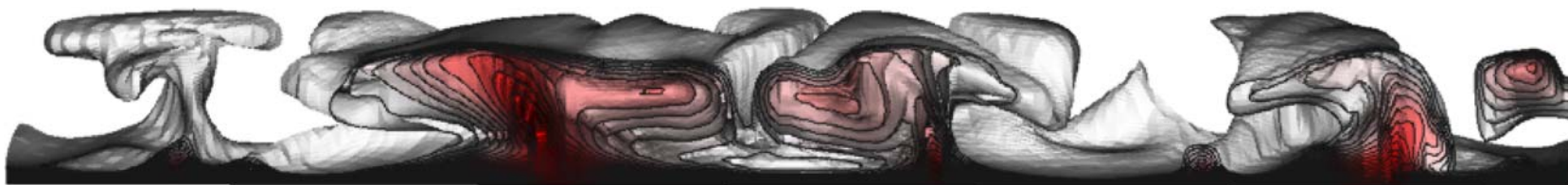
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These are preliminary lecture notes, intended only for distribution to participants

# Local heat transport analysis based on DNS/LES of turbulent RBC in wide cylindrical containers

Olga Shishkina & Claus Wagner

German Aerospace Center  
Institute for Aerodynamics and Flow Technology  
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# Outline

- 1 DNS and LES of turbulent Rayleigh–Bénard convection in wide cylindrical containers
- 2 Dependences of the flow patterns on  $Ra$  and  $\Gamma$
- 3 Thermal plume extraction
- 4 Thermal dissipation rate analysis
- 5 Spatial distribution of the local heat fluxes

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# Conducted DNS and LES of turbulent RBC in wide cylindrical containers

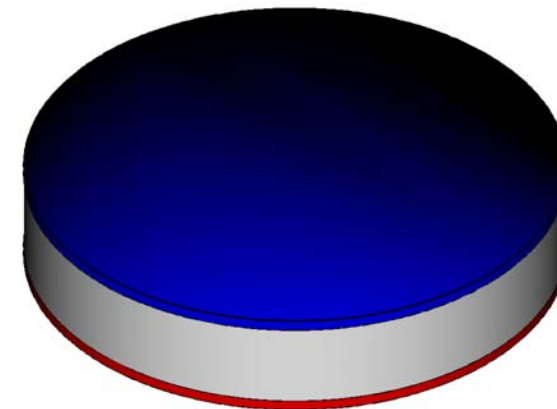
## Main parameters

- Rayleigh number  $Ra = \alpha g H^3 \Delta\theta / (\kappa \nu)$
- Prandtl number  $Pr = \nu / \kappa$
- Aspect ratio  $\Gamma = D / H$

$H$  height of the cylinder  
 $D$  diameter of the cylinder  
 $\kappa$  thermal diffusivity  
 $\nu$  kinematic viscosity  
 $\Delta\theta$  temperature difference  
 $\alpha$  thermal expansion coefficient  
 $g$  gravitational acceleration

## Conducted simulations

- $Ra = 10^5, \dots, 10^9$
- $Pr = 0.7$
- $\Gamma = 10, \Gamma = 5$



# Conducted DNS and LES of turbulent RBC in wide cylindrical containers

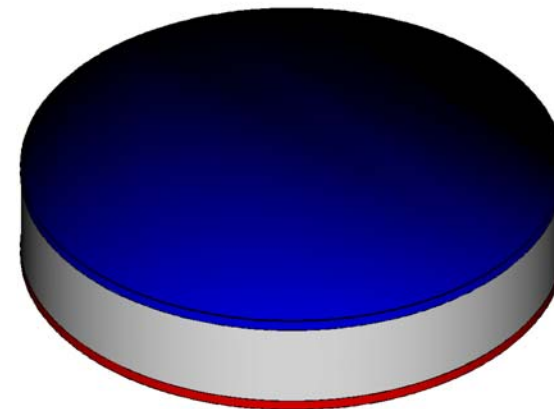
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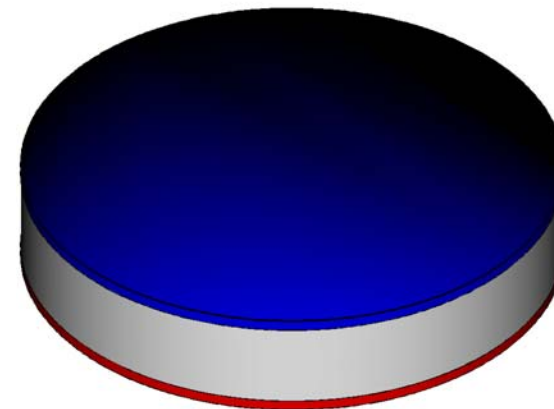
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# Finite volume approach to simulation of RBC

## Filtered governing dimensionless equations

$$\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial(\bar{u}_i \bar{u}_j)}{\partial x_j} + \frac{\partial \tau_{ij}}{\partial x_j} + \frac{\partial \bar{p}}{\partial x_i} = 2 \left( \frac{Pr}{\Gamma^3 Ra} \right)^{1/2} \frac{\partial \bar{S}_{ij}}{\partial x_j} + \bar{T} \delta_{i3}$$

$$\frac{\partial \bar{u}_i}{\partial x_i} = 0$$

$$\frac{\partial \bar{T}}{\partial t} + \frac{\partial(\bar{T} \bar{u}_j)}{\partial x_j} + \frac{\partial \tau_{Tj}}{\partial x_j} = \left( \frac{1}{\Gamma^3 Ra Pr} \right)^{1/2} \frac{\partial^2 \bar{T}}{\partial x_j^2}$$

## Filtered strain tensor

$$\bar{S}_{ij} = \frac{1}{2} \left( \frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right)$$

## Subgrid scale stress tensors

$$\tau_{ij} = \bar{u}_i \bar{u}_j - \bar{u}_i \bar{u}_j$$

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# LES: Tensor diffusivity subgrid-scale model

## Exact series expansions for subgrid scale stress tensors

$$\overline{uv} - \bar{u}\bar{v} = \frac{1}{12} \sum_{\beta=z,r,\varphi} \Delta\beta^2 \frac{\partial\bar{u}}{\partial\beta} \frac{\partial\bar{v}}{\partial\beta} + \mathcal{O} \left( \sum_{\beta=z,r,\varphi} \Delta\beta^4 \right)$$



Leonard, Adv. Geophys. **18** (1974) 237

DNS:  $10^5 \leq Ra \leq 10^7$

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# Numerical method

- fourth order spatial discretization schemes
- calculation of the velocity field at the cylinder axis
- hybrid explicit/semi-implicit time stepping
- numerical von Neumann stability
- adaptive mesh generation



Shishkina & Wagner *Computers & Fluids* (2006)

A fourth order finite volume scheme for turbulent flow simulations in cylindrical domains

# Resolution check

$$Nu_{S_z} = \Gamma^{1/2} Ra^{1/2} Pr^{1/2} \langle u_z T \rangle_{t, S_z} - \Gamma^{-1} \langle \partial T / \partial z \rangle_{t, S_z}$$

$$Nu_V = \Gamma^{1/2} Ra^{1/2} Pr^{1/2} \langle u_z T \rangle_{t, V} + 1$$

Nusselt numbers calculated at different locations,  $\Gamma = 5$

$Ra$	$10^6$	$10^7$	$10^8$
$Nu_{S_z}$ for $z = 0$	8.06	15.54	33.0
$Nu_{S_z}$ for $z = H/2$	8.27	15.56	33.1
$Nu_{S_z}$ for $z = H$	8.09	15.54	32.9
$Nu_V$	8.22	15.55	32.8
Mean $Nu$	8.2	15.55	32.9
Error, less than	1.7%	0.1%	0.7%
Time	42	107	43

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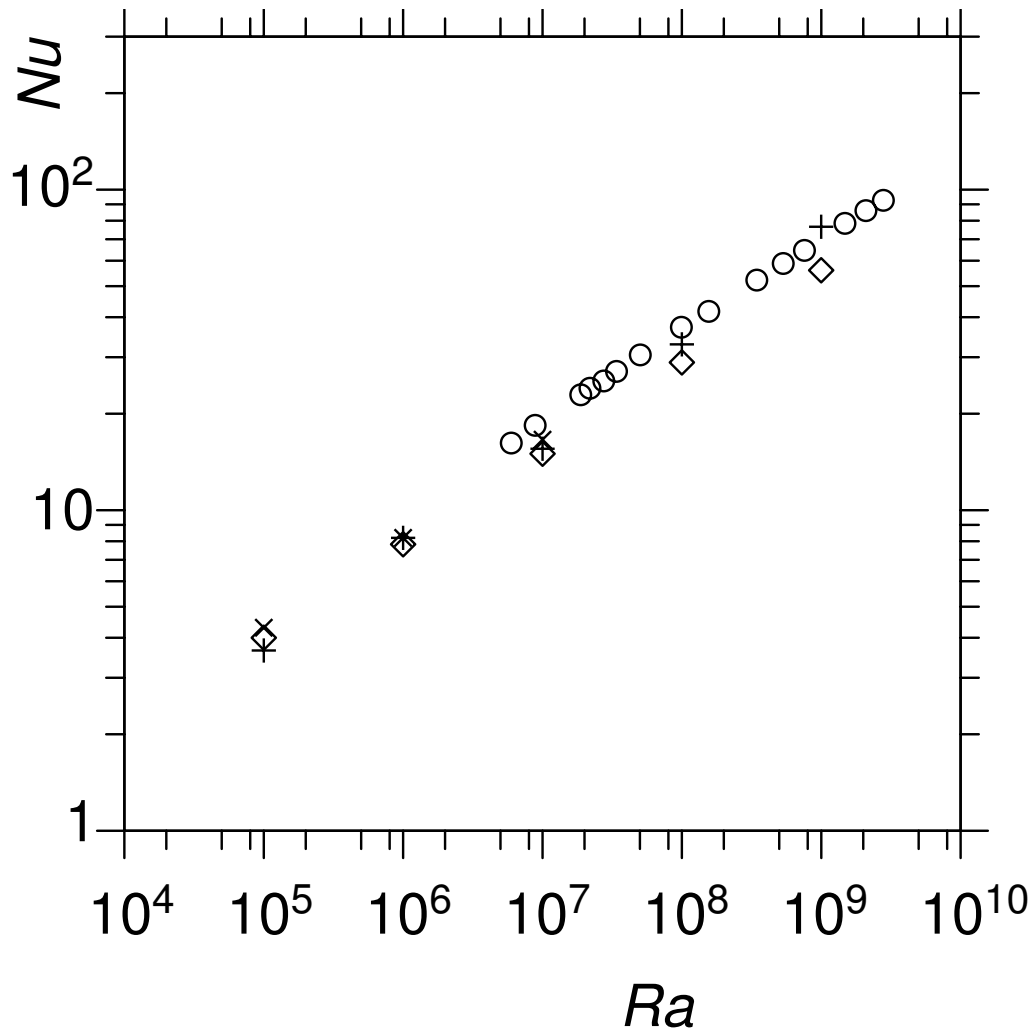
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# Nusselt number vs. Rayleigh number



$Pr = 0.7, \Gamma = 5$  (pluses)

$Pr = 0.7, \Gamma = 10$  (crosses)



Shishkina & Wagner *J. Fluid Mech.* **546** (2006)

$Pr = 0.7, \Gamma = 6.7$  (diamonds)



Wu & Libchaber *Phys. Rev. A* **45** (1992)

$Pr = 0.68, \Gamma = 1$  (circles)

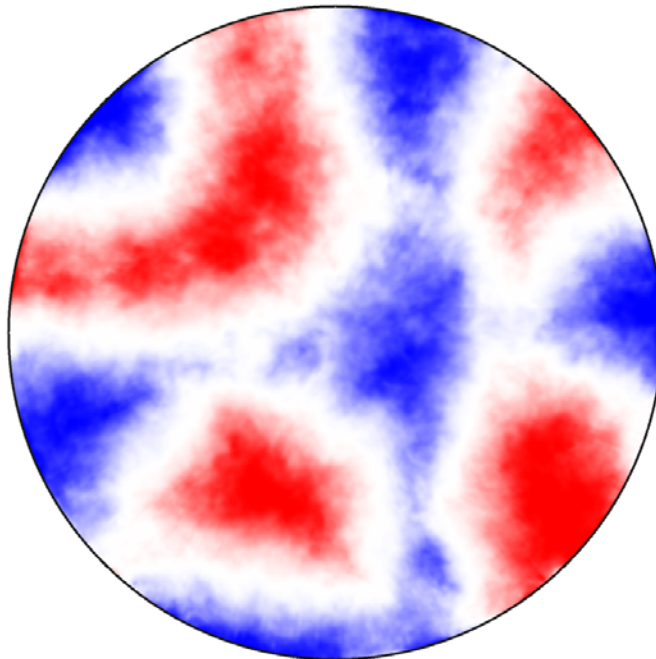
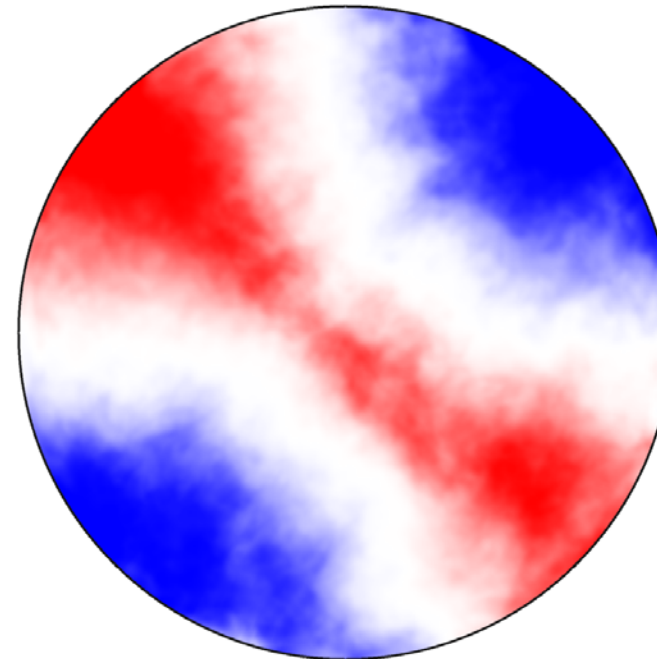


Niemela & Sreenivasan *J. Fluid Mech.* **481** (2003)

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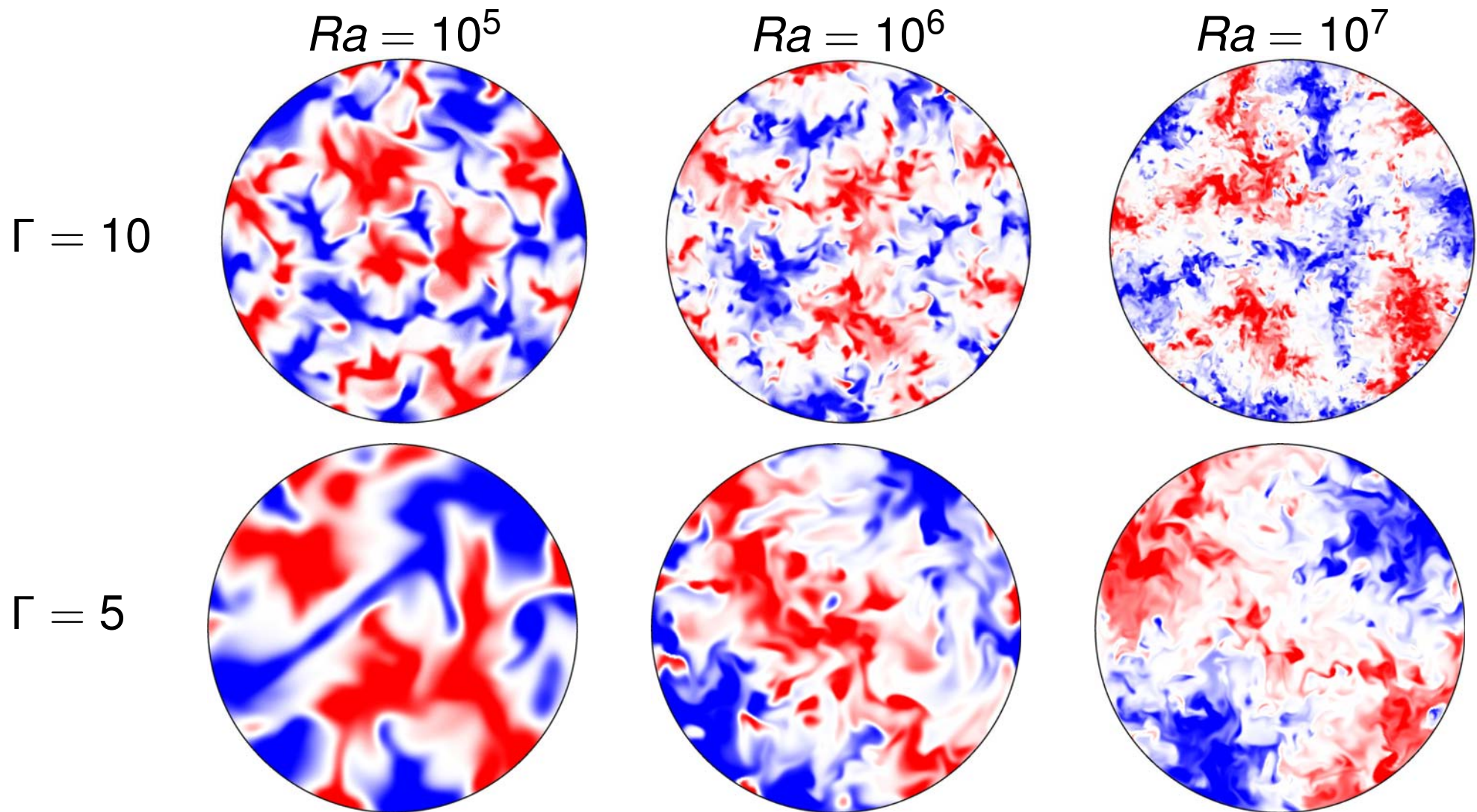
# Mean temperature patterns at $z = H/2$ , $Ra = 10^7$

 $\Gamma = 10$  $\Gamma = 5$ 

The colour scale ranges from blue (negative values) through white (zero) to red (positive values)

Mean flow structures depend strongly on the aspect ratio

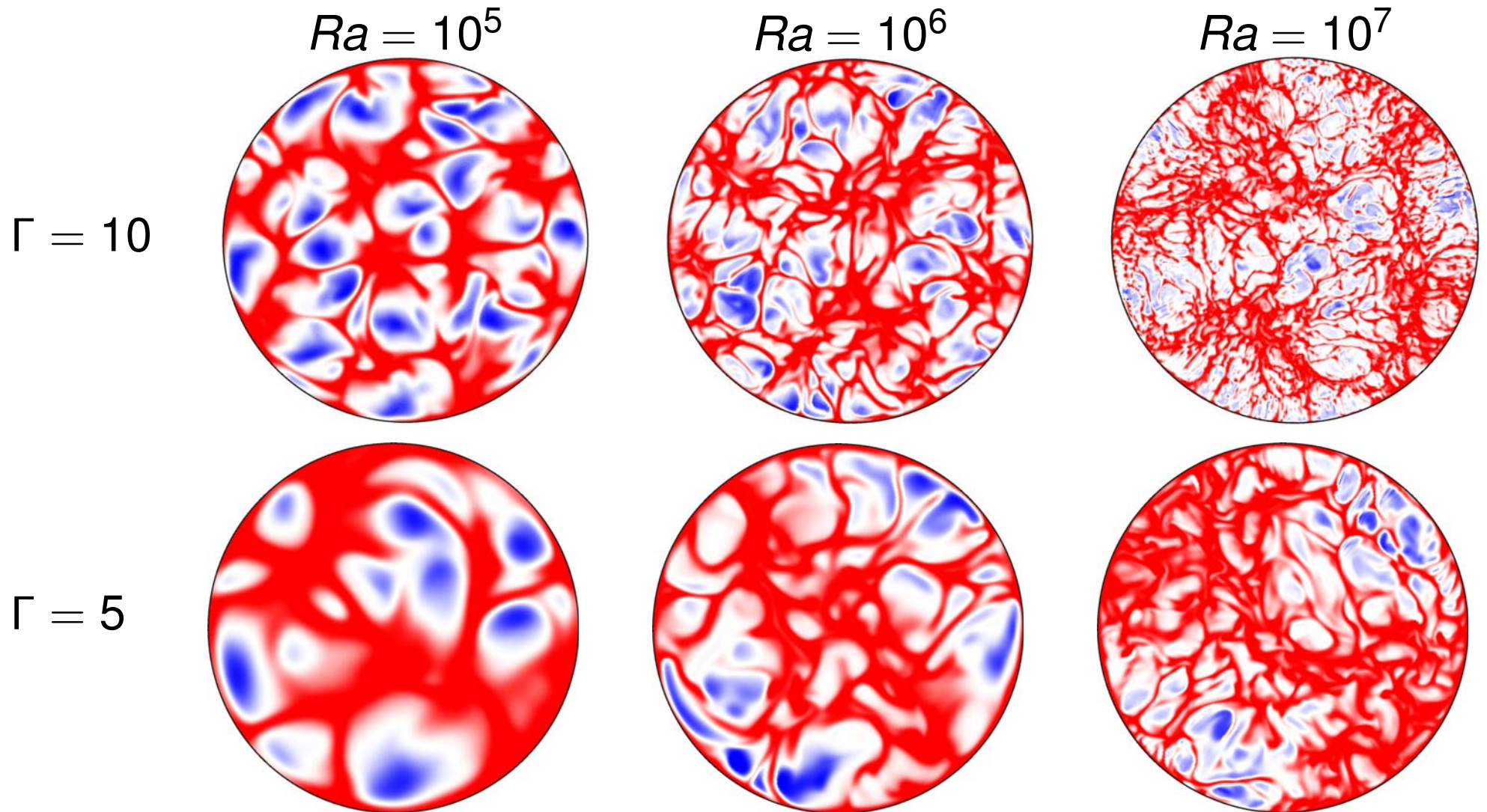
# Temperature patterns at $z = H/2$



The number of visible large flow structures increases with  $\Gamma$  and decreases with growing  $Ra$



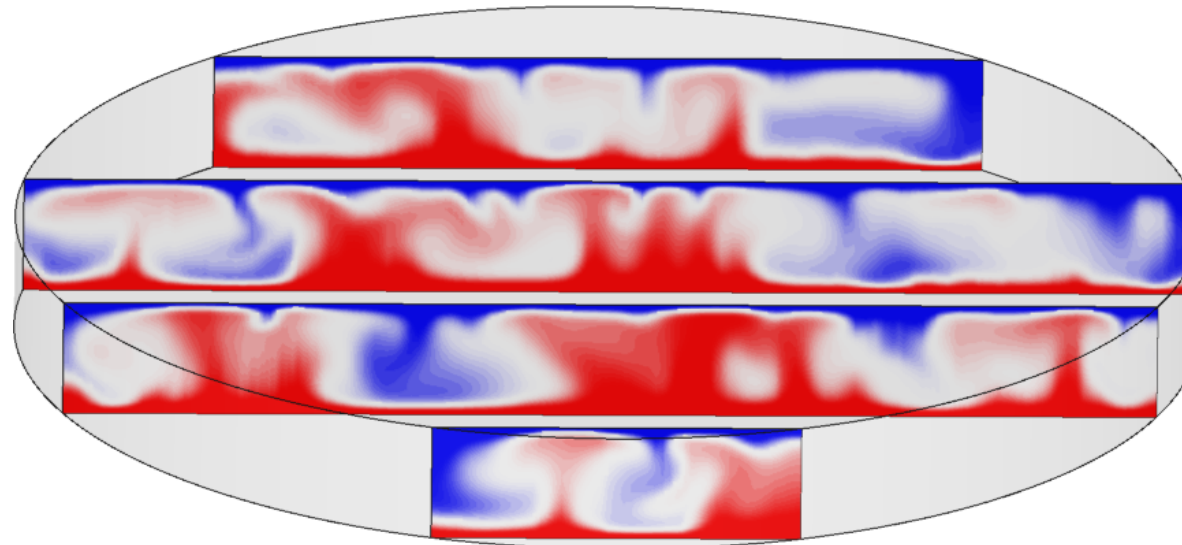
# Temperature patterns at $z = H/(2Nu)$



The plume roots become thinner with growing  $Ra$  and  $\Gamma$

# Vertical cross-sections of the temperature

for  $Ra = 10^5$ ,  $Pr = 0.7$ ,  $\Gamma = 10$



The colour scale ranges from blue (negative values) through white (zero) to red (positive values)

Warm and cold thermal plumes are seen in a mushroom-like form

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# Snapshots of $T\varepsilon_\theta$ in horizontal cross-sections

with superimposed velocity vectors for  $Ra = 10^5$

## Definition of $C(T, \varepsilon_\theta)$

$$C(T, \varepsilon_\theta) \equiv T\varepsilon_\theta$$

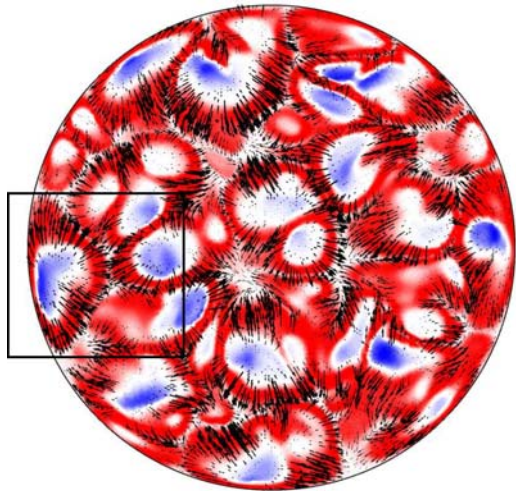
with the thermal dissipation rate

$$\varepsilon_\theta = \Gamma^{-3/2} Ra^{-1/2} Pr^{-1/2} (\nabla T)^2$$



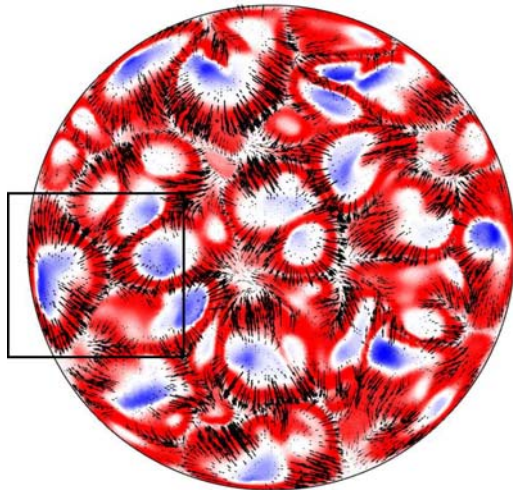
# Snapshots of $T_{\varepsilon\theta}$ in horizontal cross-sections with superimposed velocity vectors for $Ra = 10^5$

$$z = 10^{-2}H$$

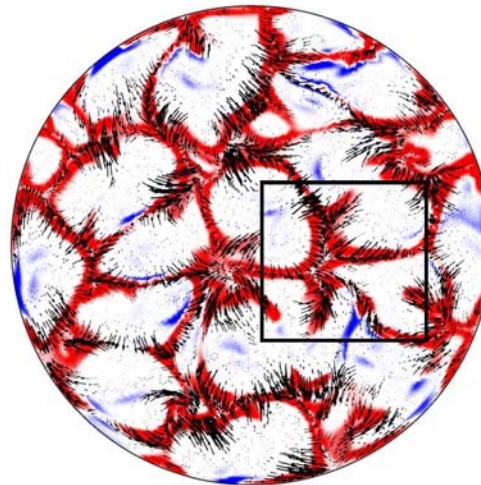


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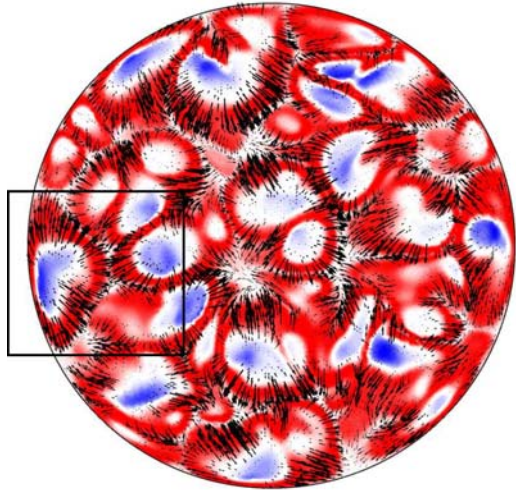


$$z = H/(2Nu)$$

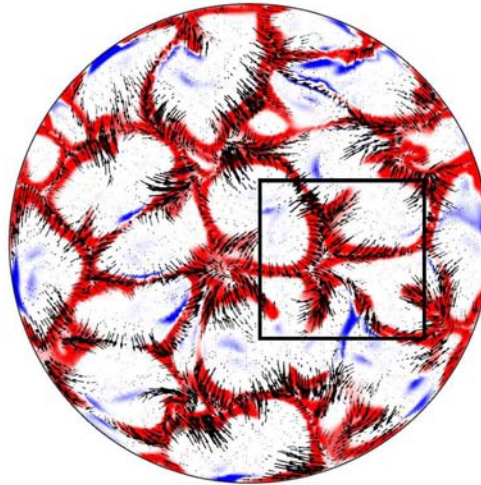


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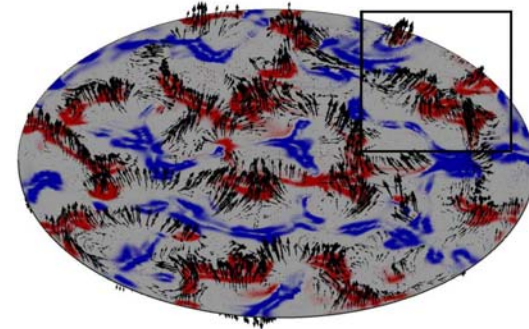
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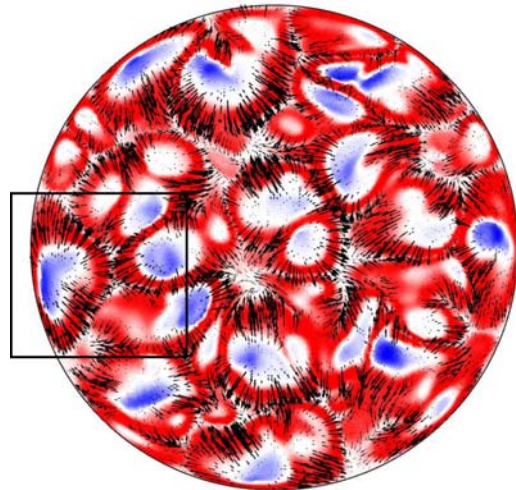
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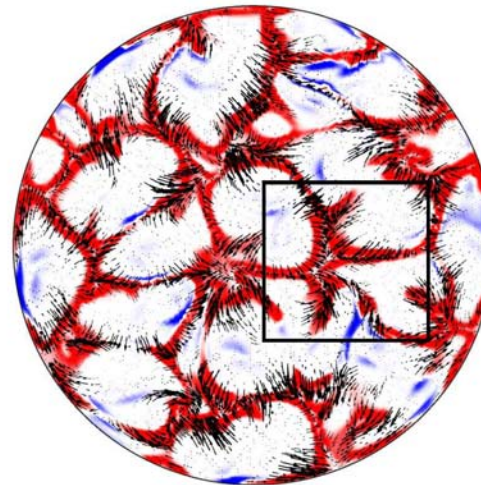


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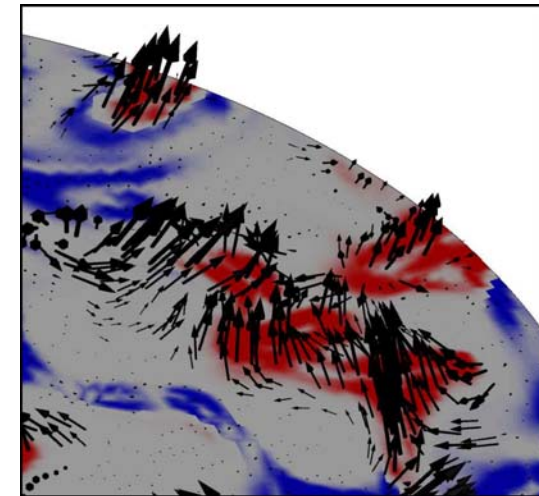
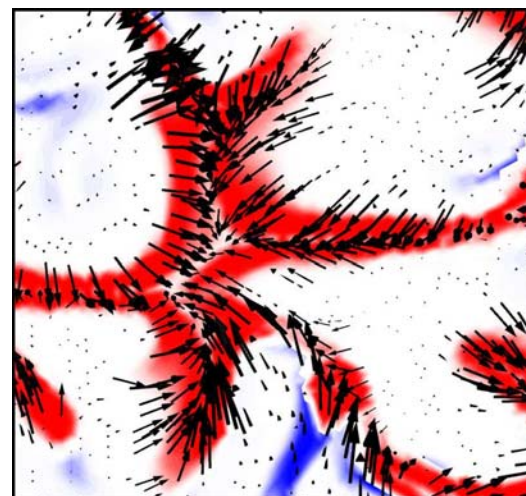
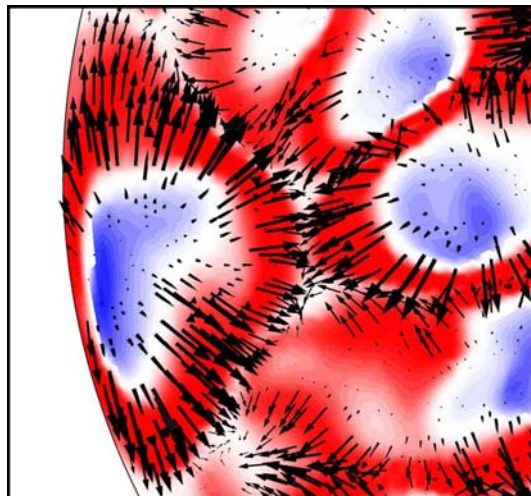
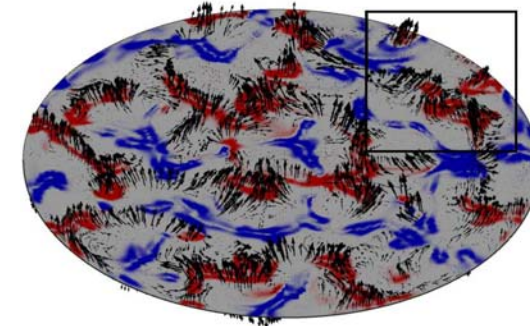
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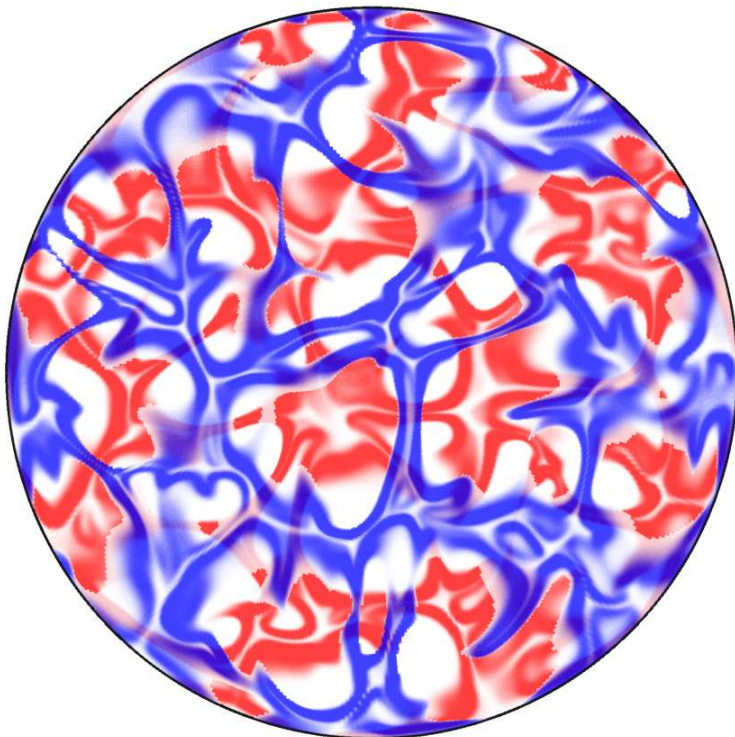


highlight the three-dimensional nature of the plumes



# Superimposed instantaneous fields of $T\varepsilon_\theta$ ,

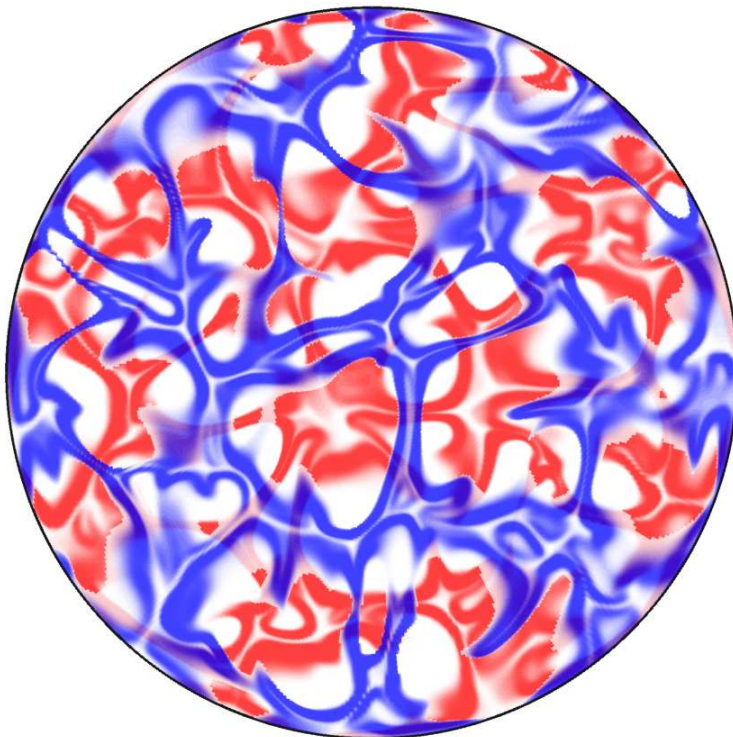
$Ra = 10^5$ ,  $z = \lambda_\theta$  (red) and  $z = H - \lambda_\theta$  (blue)





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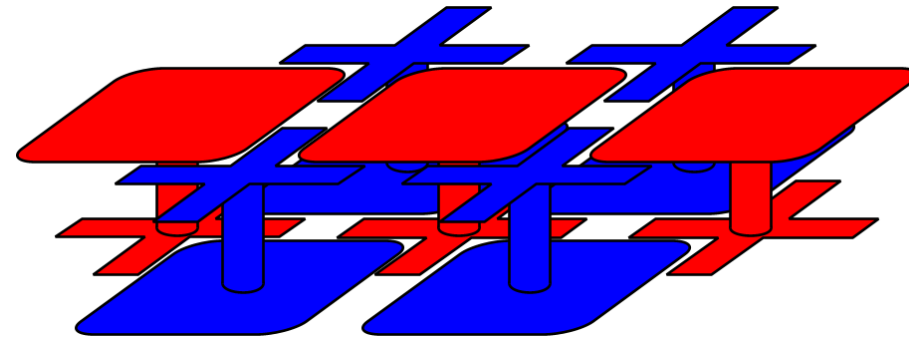
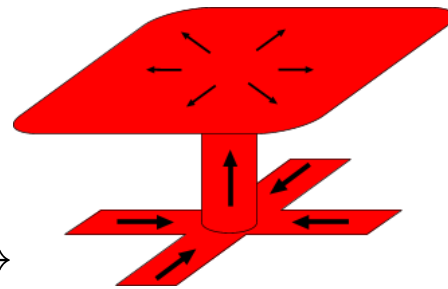
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In moderate-Rayleigh-number case the roots of the warm and cold thermal plumes have a tendency to intersect at right angles.

# Formation of warm and cold plumes in wide containers in moderate-Rayleigh-number regime

Cap →  
Stem →  
Roots →



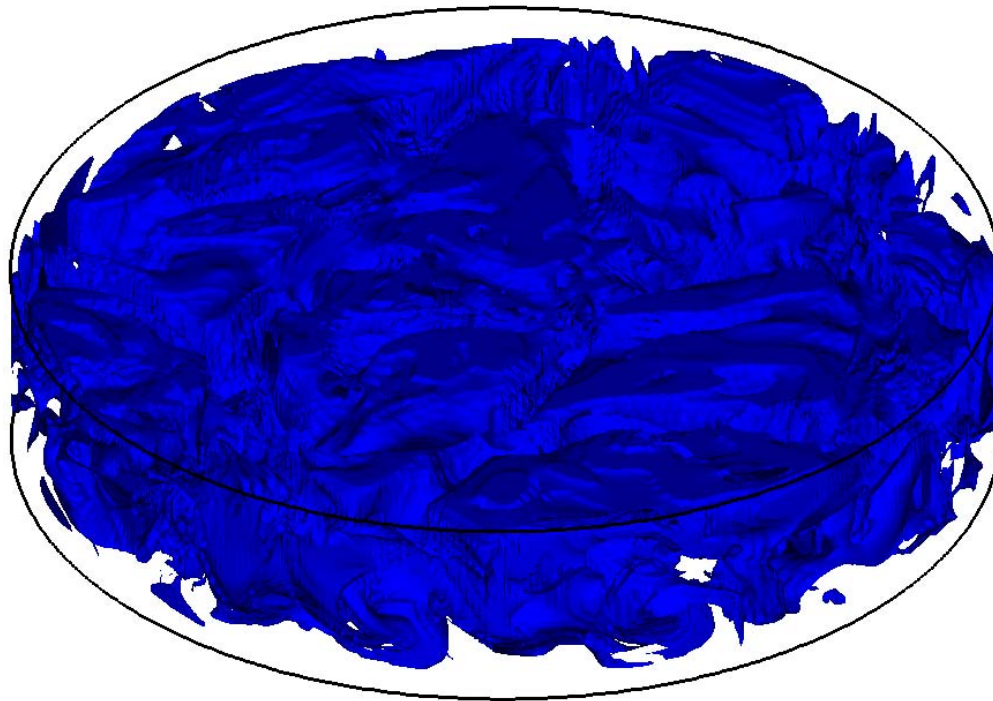


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# Spatial distribution of the thermal dissipation rate

$$\varepsilon_{\theta} = \Gamma^{-3/2} Ra^{-1/2} Pr^{-1/2} (\nabla T)^2$$



$$Ra = 10^6, \Gamma = 5$$

Deep-blue  $0 \leq \frac{\varepsilon_{\theta}}{\varepsilon_{\theta, \max}} \leq 5 \cdot 10^{-4}$

Blue  $5 \cdot 10^{-4} < \frac{\varepsilon_{\theta}}{\varepsilon_{\theta, \max}} \leq 2 \cdot 10^{-3}$

Green  $2 \cdot 10^{-3} < \frac{\varepsilon_{\theta}}{\varepsilon_{\theta, \max}} \leq 3 \cdot 10^{-3}$

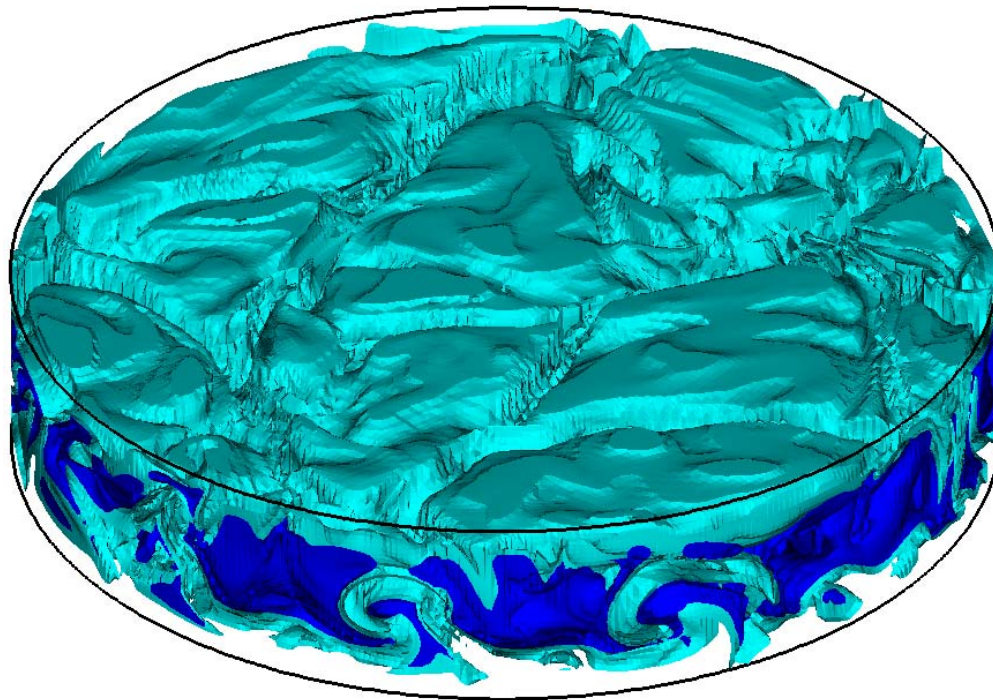
Yellow  $3 \cdot 10^{-3} < \frac{\varepsilon_{\theta}}{\varepsilon_{\theta, \max}} \leq 5 \cdot 10^{-3}$

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with  $\varepsilon_{\theta, \max} = \max_V \varepsilon_{\theta}$

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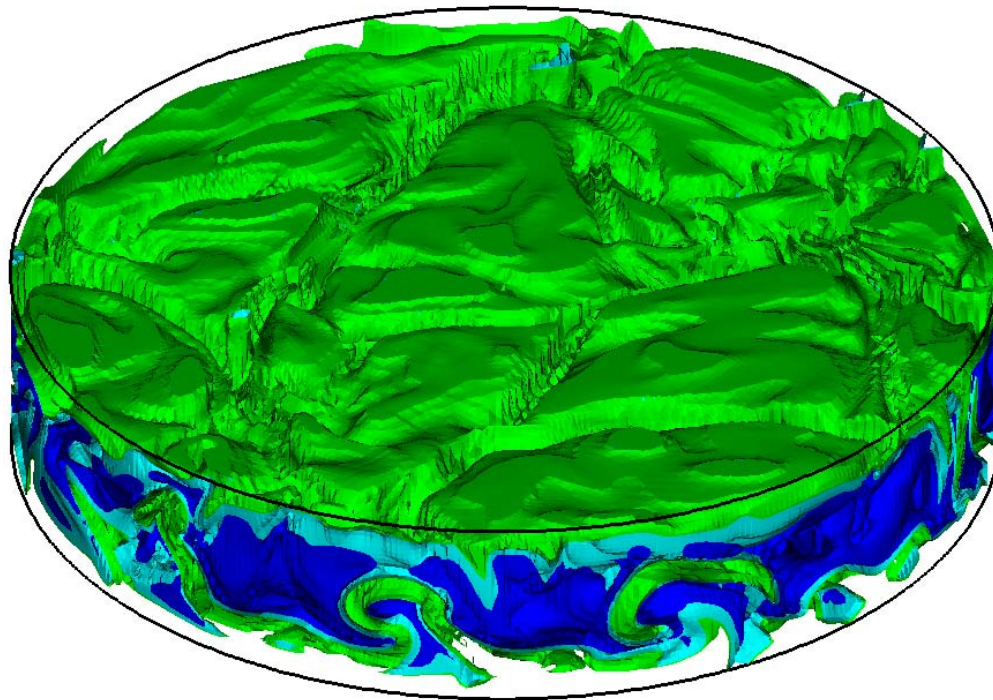
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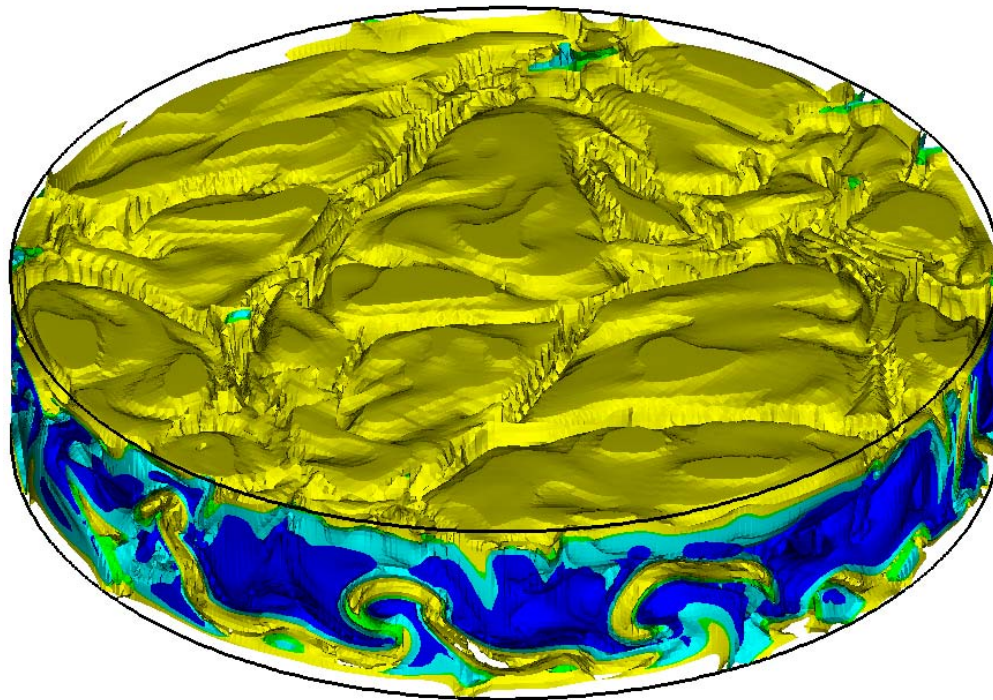
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Green  $2 \cdot 10^{-3} < \frac{\varepsilon_{\theta}}{\varepsilon_{\theta, \max}} \leq 3 \cdot 10^{-3}$

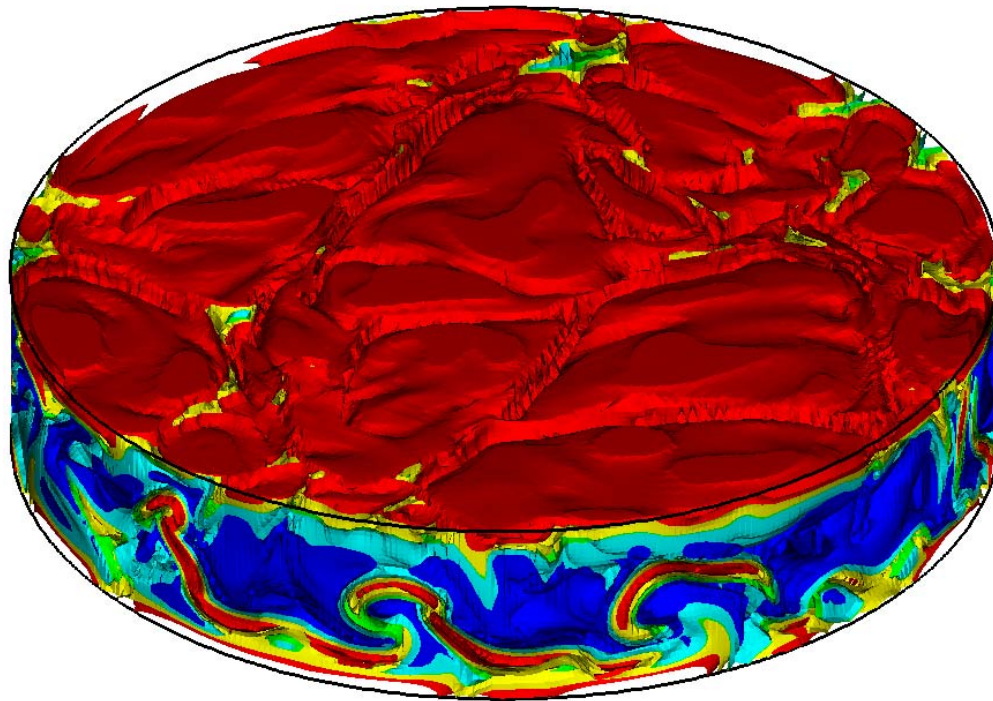
Yellow  $3 \cdot 10^{-3} < \frac{\varepsilon_{\theta}}{\varepsilon_{\theta, \max}} \leq 5 \cdot 10^{-3}$

Red  $5 \cdot 10^{-3} < \frac{\varepsilon_{\theta}}{\varepsilon_{\theta, \max}} \leq 10^{-2}$

with  $\varepsilon_{\theta, \max} = \max_V \varepsilon_{\theta}$

# Spatial distribution of the thermal dissipation rate

$$\varepsilon_\theta = \Gamma^{-3/2} Ra^{-1/2} Pr^{-1/2} (\nabla T)^2$$



$$Ra = 10^6, \Gamma = 5$$

Deep-blue  $0 \leq \frac{\varepsilon_\theta}{\varepsilon_{\theta, \max}} \leq 5 \cdot 10^{-4}$

Blue  $5 \cdot 10^{-4} < \frac{\varepsilon_\theta}{\varepsilon_{\theta, \max}} \leq 2 \cdot 10^{-3}$

Green  $2 \cdot 10^{-3} < \frac{\varepsilon_\theta}{\varepsilon_{\theta, \max}} \leq 3 \cdot 10^{-3}$

Yellow  $3 \cdot 10^{-3} < \frac{\varepsilon_\theta}{\varepsilon_{\theta, \max}} \leq 5 \cdot 10^{-3}$

Red  $5 \cdot 10^{-3} < \frac{\varepsilon_\theta}{\varepsilon_{\theta, \max}} \leq 10^{-2}$

with  $\varepsilon_{\theta, \max} = \max_V \varepsilon_\theta$

# Contribution to $\langle \varepsilon_\theta \rangle_V$ from the subdomain $\varepsilon_\theta \leq \xi \varepsilon_{\theta, \max}$

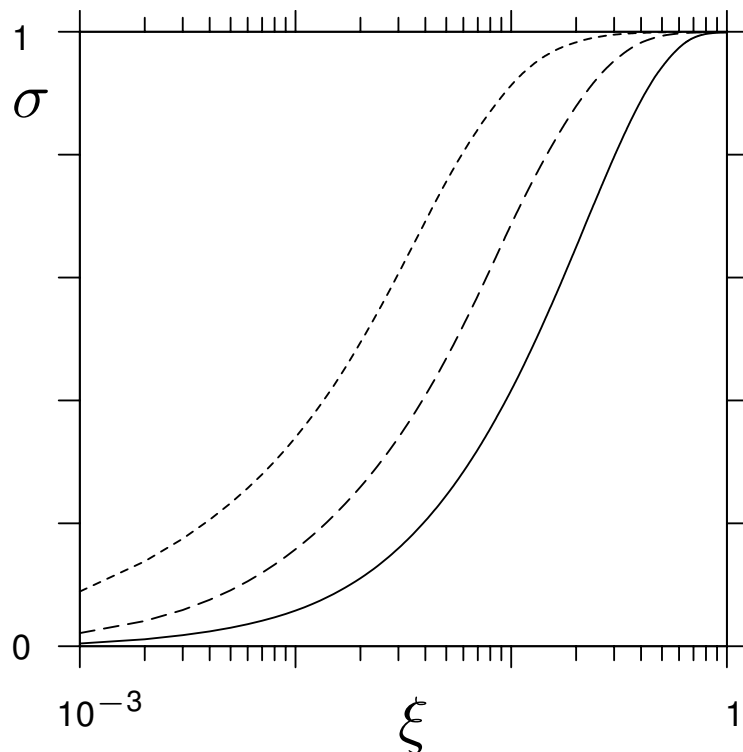
$$\sigma(\xi) = \frac{\langle \varepsilon_\theta \vartheta(\xi \varepsilon_{\theta, \max} - \varepsilon_\theta) \rangle_V}{\langle \varepsilon_\theta \rangle_V},$$

with  $\vartheta(x)$  – the Heaviside function

# Contribution to $\langle \varepsilon_\theta \rangle_V$ from the subdomain $\varepsilon_\theta \leq \xi \varepsilon_{\theta, \max}$

$$\sigma(\xi) = \frac{\langle \varepsilon_\theta \vartheta(\xi \varepsilon_{\theta, \max} - \varepsilon_\theta) \rangle_V}{\langle \varepsilon_\theta \rangle_V},$$

with  $\vartheta(x)$  – the Heaviside function



For  $\Gamma = 10$  the parts of the domain, where  $\varepsilon_\theta$  is less than 0.1% of its maximum, contribute

- 0.4% ( $Ra = 10^5$ , —)
- 2.1% ( $Ra = 10^6$ , ---)
- 8.9% ( $Ra = 10^7$ , - - - -)

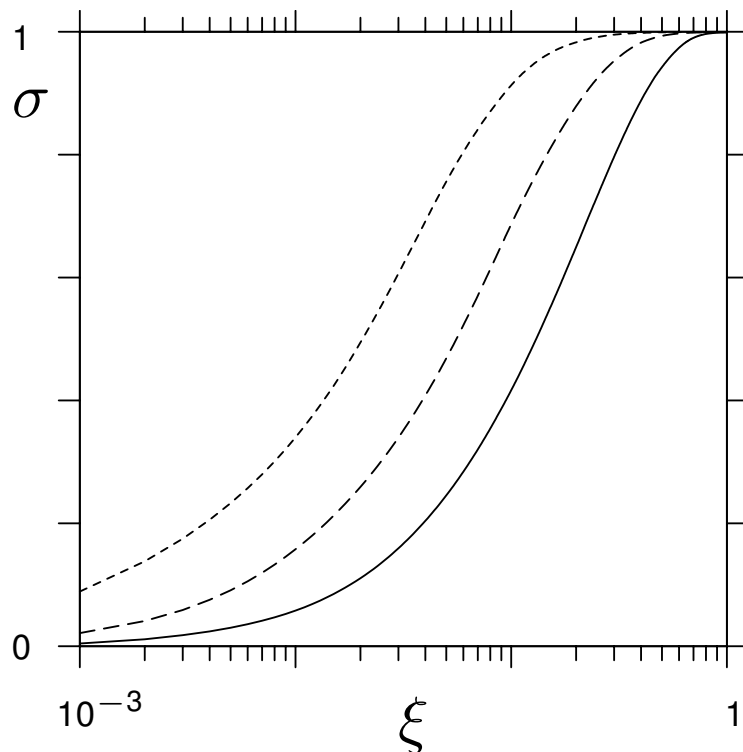
to the volume averaged  $\langle \varepsilon_\theta \rangle_V$ .



# Contribution to $\langle \varepsilon_\theta \rangle_V$ from the subdomain $\varepsilon_\theta \leq \xi \varepsilon_{\theta, \max}$

$$\sigma(\xi) = \frac{\langle \varepsilon_\theta \vartheta(\xi \varepsilon_{\theta, \max} - \varepsilon_\theta) \rangle_V}{\langle \varepsilon_\theta \rangle_V},$$

with  $\vartheta(x)$  – the Heaviside function



The turbulent background part of the volume averaged thermal dissipation rate increases with the Rayleigh number.

Contribution to  $\langle \varepsilon_\theta \rangle_V = \Gamma^{1/2} Ra^{-1/2} Pr^{-1/2} Nu$   
with growing  $Ra$

Turbulent background



Grossmann & Lohse **407** *JFM* (2000)



Shishkina & Wagner **546** *JFM* (2006)

Contribution to  $\langle \varepsilon_\theta \rangle_V = \Gamma^{1/2} Ra^{-1/2} Pr^{-1/2} Nu$   
with growing  $Ra$

Boundary layers



Turbulent background



Grossmann & Lohse **407** *JFM* (2000)



Shishkina & Wagner **546** *JFM* (2006)



Verzicco & Camussi **477** *JFM* (2003)

Contribution to  $\langle \varepsilon_\theta \rangle_V = \Gamma^{1/2} Ra^{-1/2} Pr^{-1/2} Nu$   
with growing  $Ra$

Boundary layers



Thermal plumes



Turbulent background

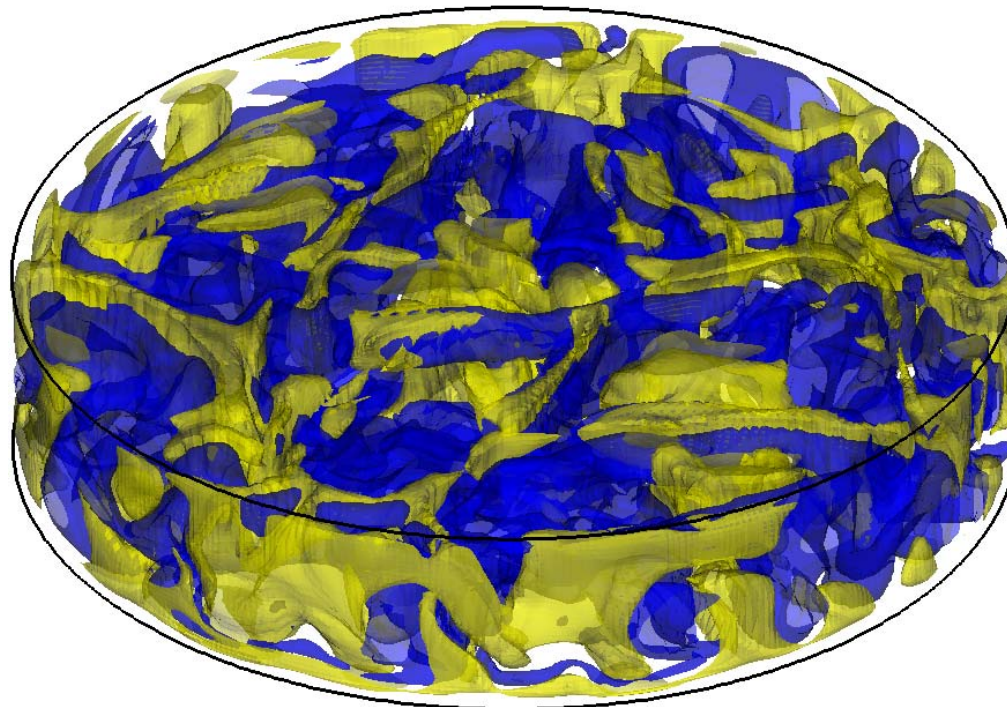


Contribution of the thermal plumes decreases with growing  $Ra$

# Outline

- 1 DNS and LES of turbulent Rayleigh–Bénard convection in wide cylindrical containers
- 2 Dependences of the flow patterns on  $Ra$  and  $\Gamma$
- 3 Thermal plume extraction
- 4 Thermal dissipation rate analysis
- 5 Spatial distribution of the local heat fluxes

# Local heat flux $\Omega$ for $Ra = 10^6$ , $\Gamma = 5$



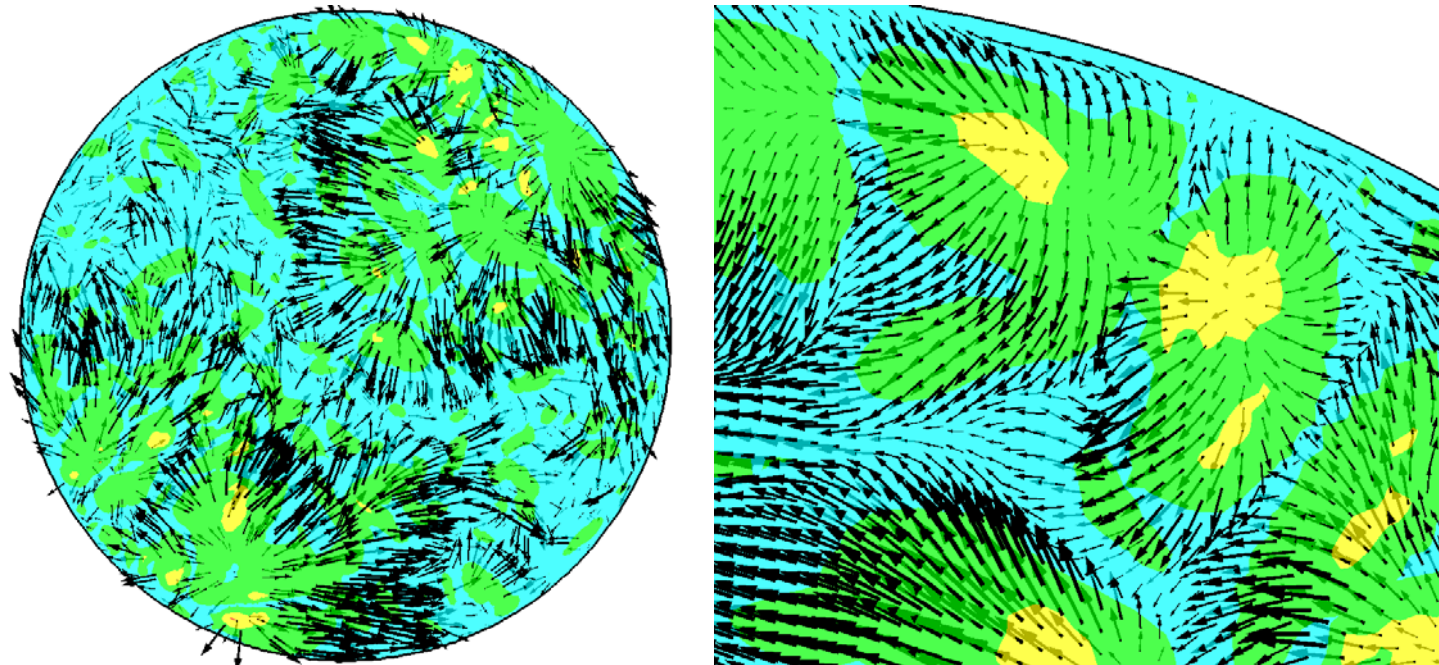
Blue  $\Omega < 0$

Yellow  $\Omega \geq 2Nu$

Up to 1/3 of the Rayleigh cell corresponds to negative  $\Omega$

# Local heat flux $\Omega$ for $Ra = 10^8, \Gamma = 5$

$z = 10^{-3}H$

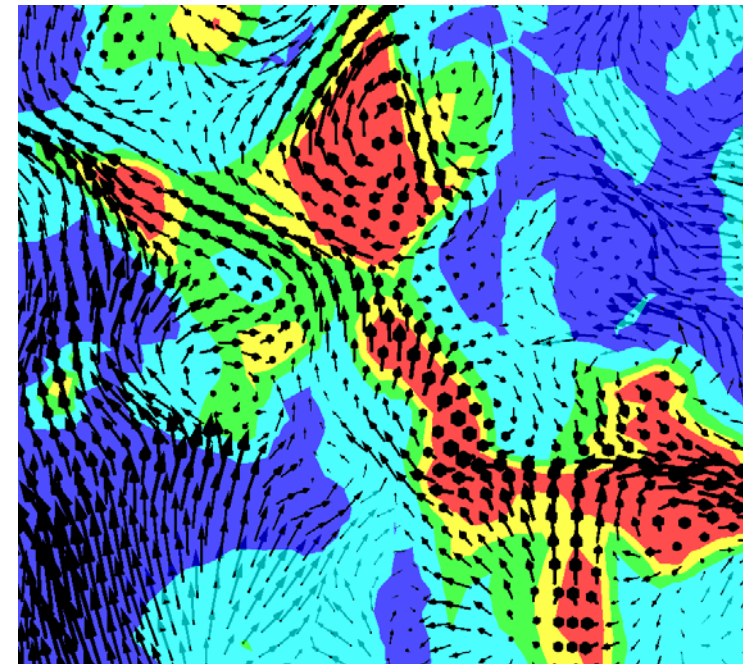
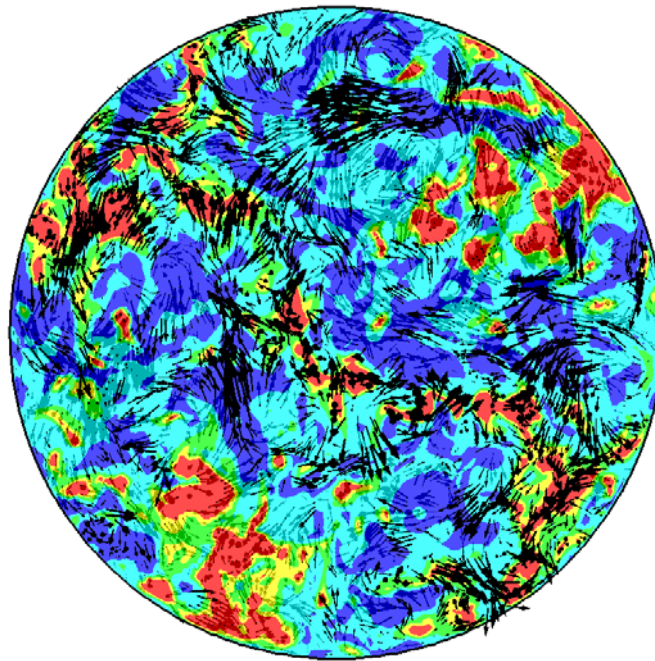


Regions of high  $\Omega$ -values correspond to the thermal plumes



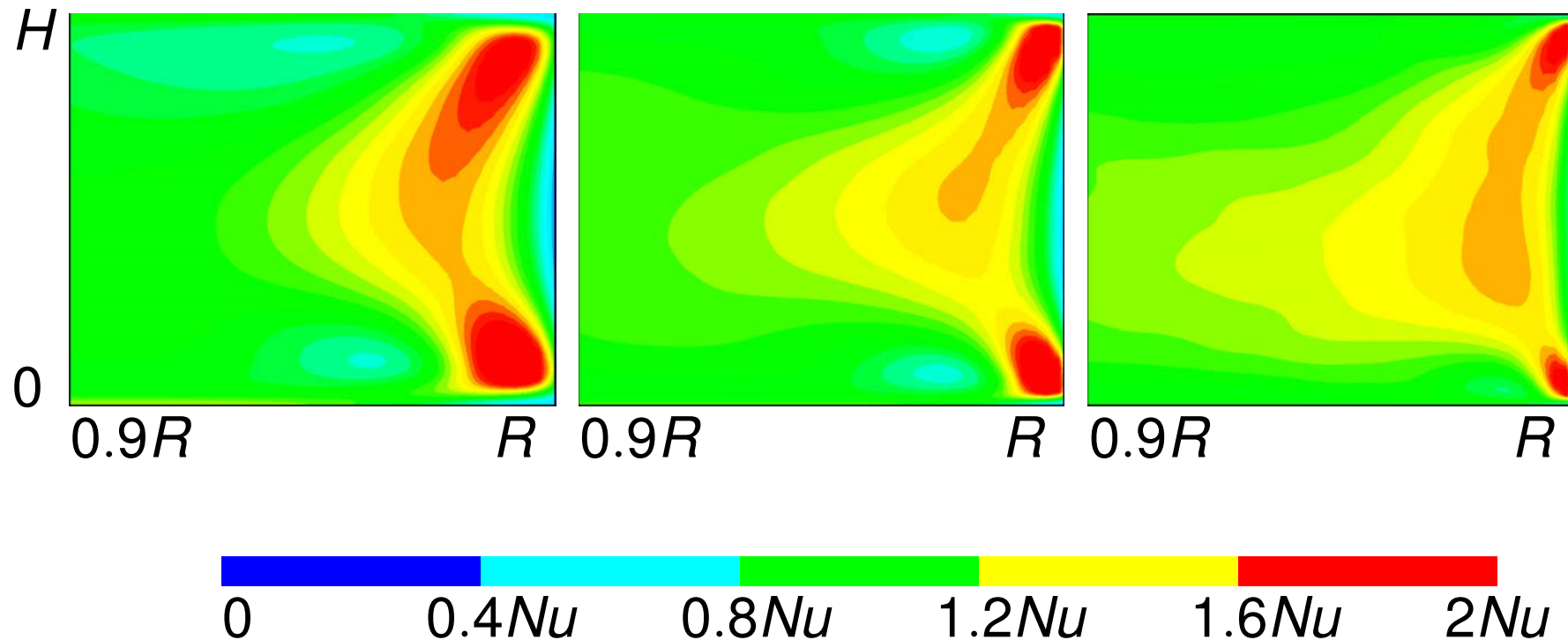
# Local heat flux $\Omega$ for $Ra = 10^8$ , $\Gamma = 5$

$z = H/2$



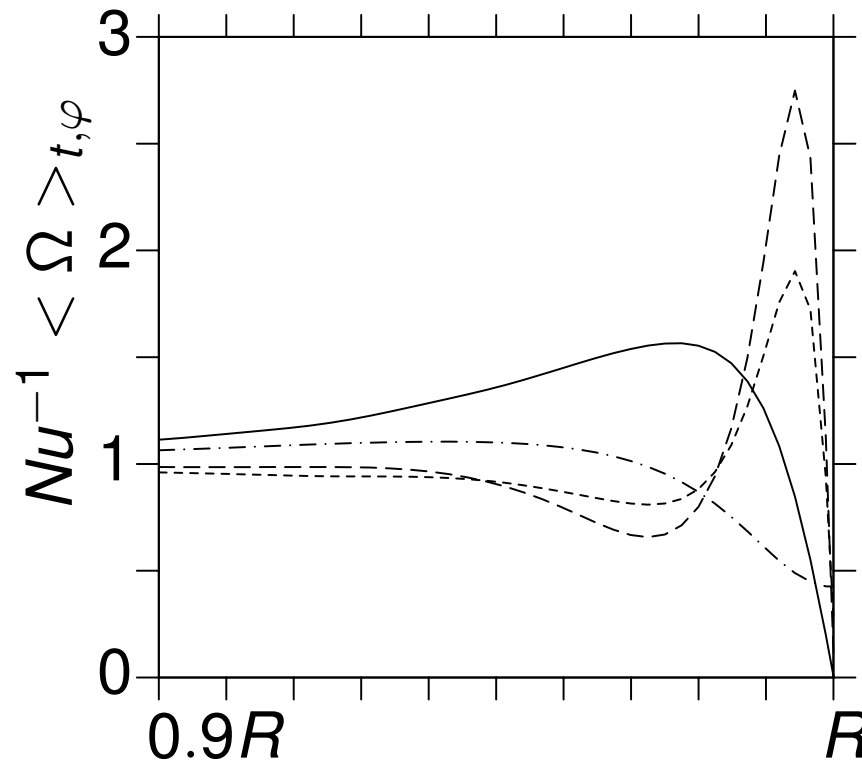
Spectrum of  $\Omega$ -values widens while moving away from the plates

# Mean heat flux $\langle \Omega \rangle_{t,\varphi}$ near the vertical wall

 $Ra = 10^6$  $Ra = 10^7$  $Ra = 10^8$ 

With growing  $Ra$  maximum points move closer to the corners

# Mean heat flux $\langle \Omega \rangle_{t,\varphi}$ near the vertical wall



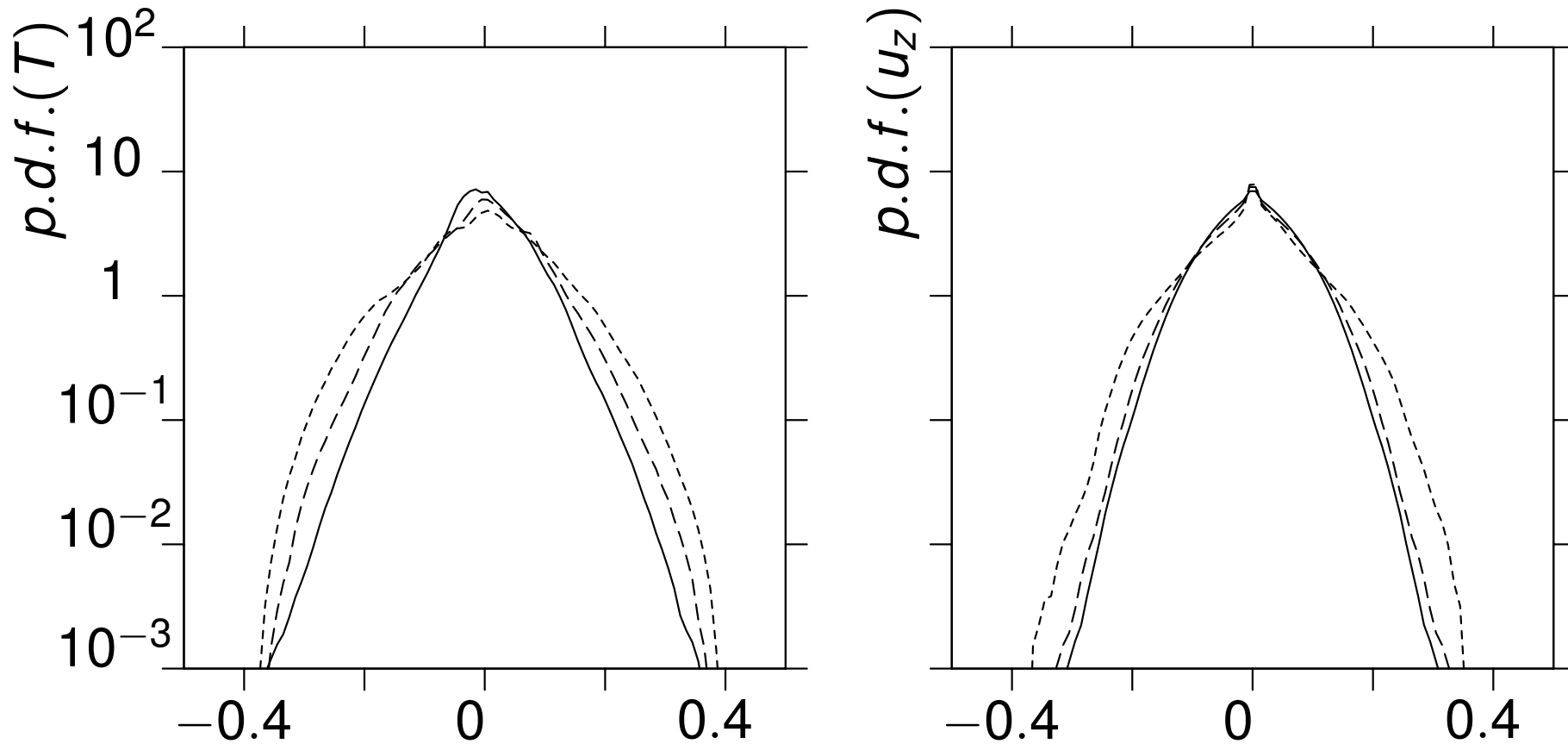
The case  $Ra = 10^7$

- $z = H/2$
- - -  $z = H/(Nu)$
- · -  $z = H/(2Nu)$
- · ·  $z = 10^{-3}H$

Highest values of  $\langle \Omega \rangle_{t,\varphi}$  are approached in the bulk at distance  $z \approx Nu^{-1}H$  from the horizontal plates

# Density functions of the temperature and axial velocity

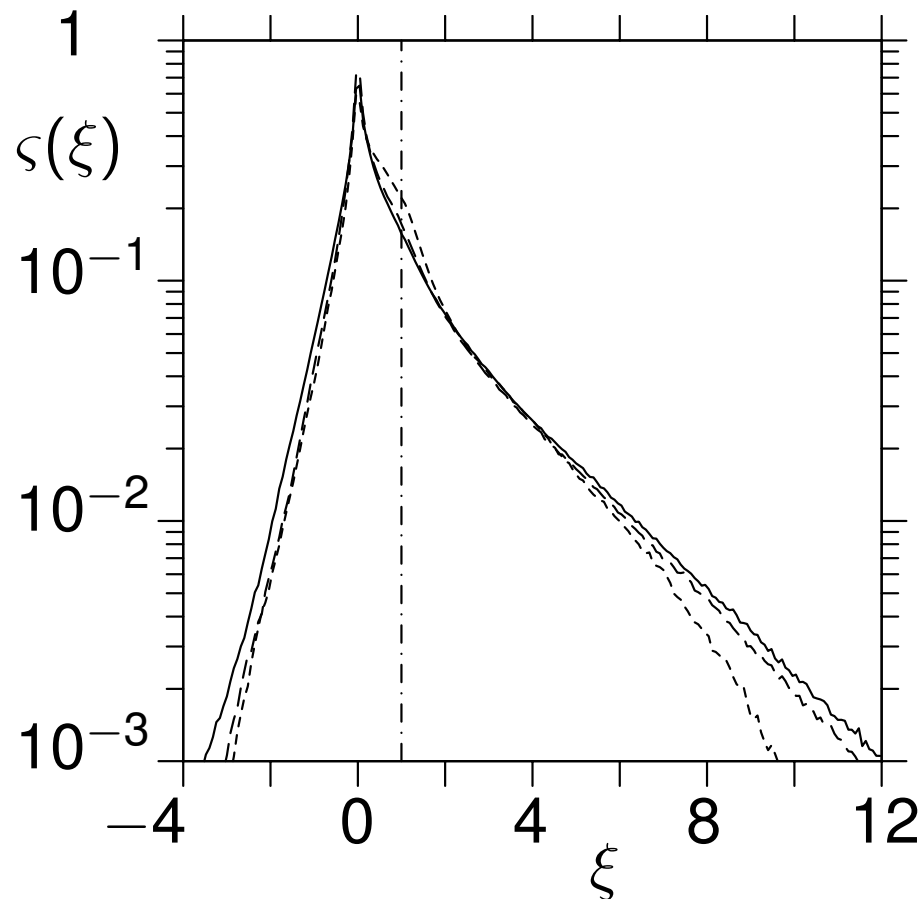
for  $Ra = 10^8$  (—),  $Ra = 10^7$  (---) and  $Ra = 10^6$  (- - - -)



p.d.f. of  $T$  and  $u_z$  are symmetric

# Volume distribution density function of the heat flux $\Omega$

for  $Ra = 10^8$  (—),  $Ra = 10^7$  (---) and  $Ra = 10^6$  (- - - -),  $\Gamma = 5$ ,  $\delta = 0.08$



$$\varsigma(\xi) = \frac{1}{\delta} \left\langle \vartheta \left( \Omega - \left( \xi_i - \frac{\delta}{2} \right) Nu \right) \right\rangle_V - \frac{1}{\delta} \left\langle \vartheta \left( \Omega - \left( \xi_i + \frac{\delta}{2} \right) Nu \right) \right\rangle_V$$

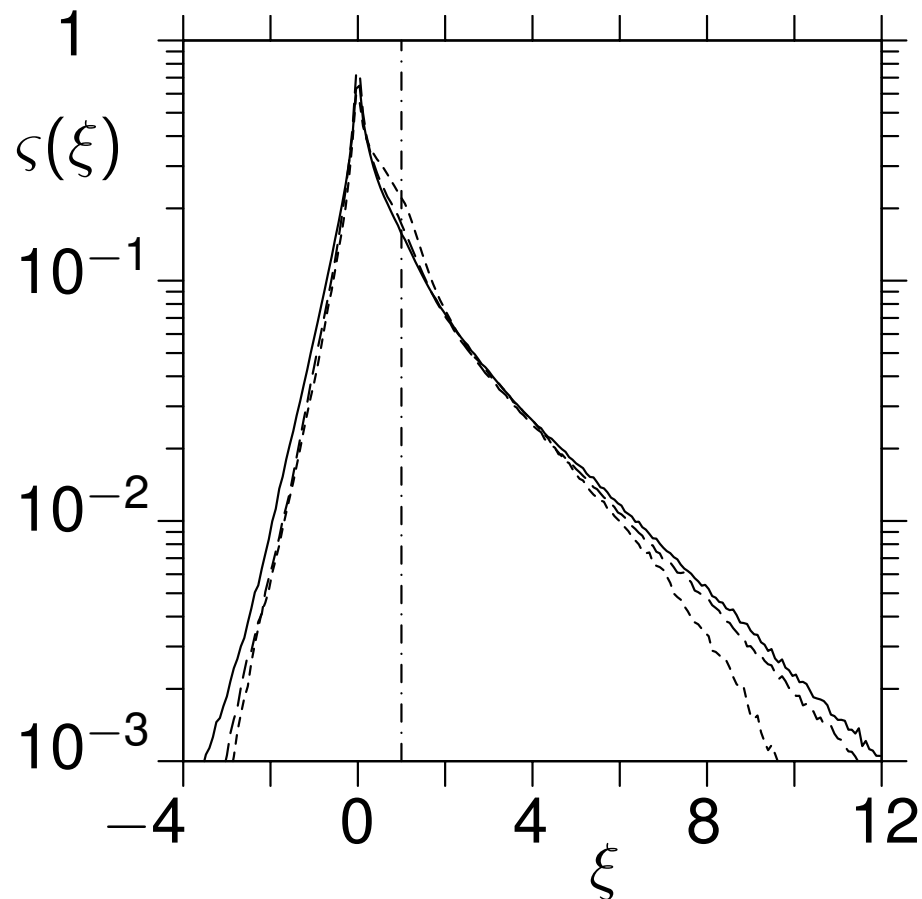
with the Heaviside function

$$\vartheta(x) = \begin{cases} 1, & \text{if } x \geq 0, \\ 0, & \text{otherwise,} \end{cases}$$

and integer  $(\xi_i/\delta + 1/2)$

# Volume distribution density function of the heat flux $\Omega$

for  $Ra = 10^8$  (—),  $Ra = 10^7$  (---) and  $Ra = 10^6$  (- - - -),  $\Gamma = 5$ ,  $\delta = 0.08$

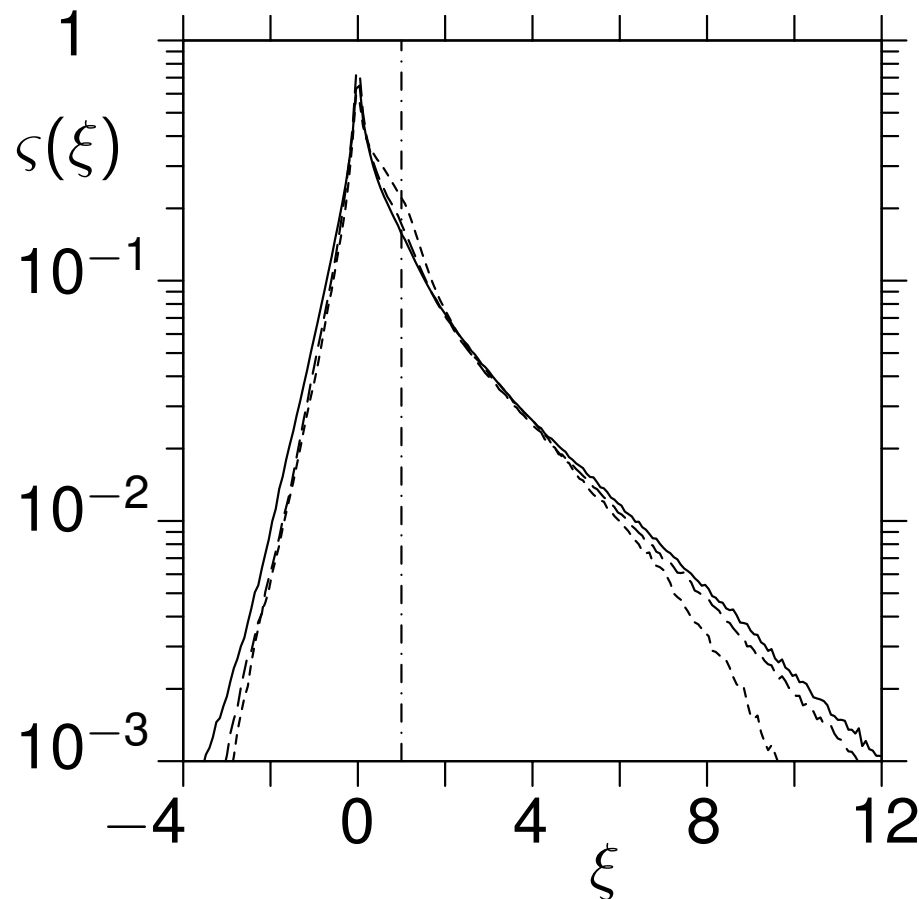


$s(\xi)$  for  $\xi \in [\xi_i - \frac{\delta}{2}, \xi_i + \frac{\delta}{2}[$   
indicates the percentage  
of the fluid volume with  
 $\Omega \in [(\xi_i - \frac{\delta}{2})Nu, (\xi_i + \frac{\delta}{2})Nu[$



# Volume distribution density function of the heat flux $\Omega$

for  $Ra = 10^8$  (—),  $Ra = 10^7$  (---) and  $Ra = 10^6$  (- - - -),  $\Gamma = 5$ ,  $\delta = 0.08$

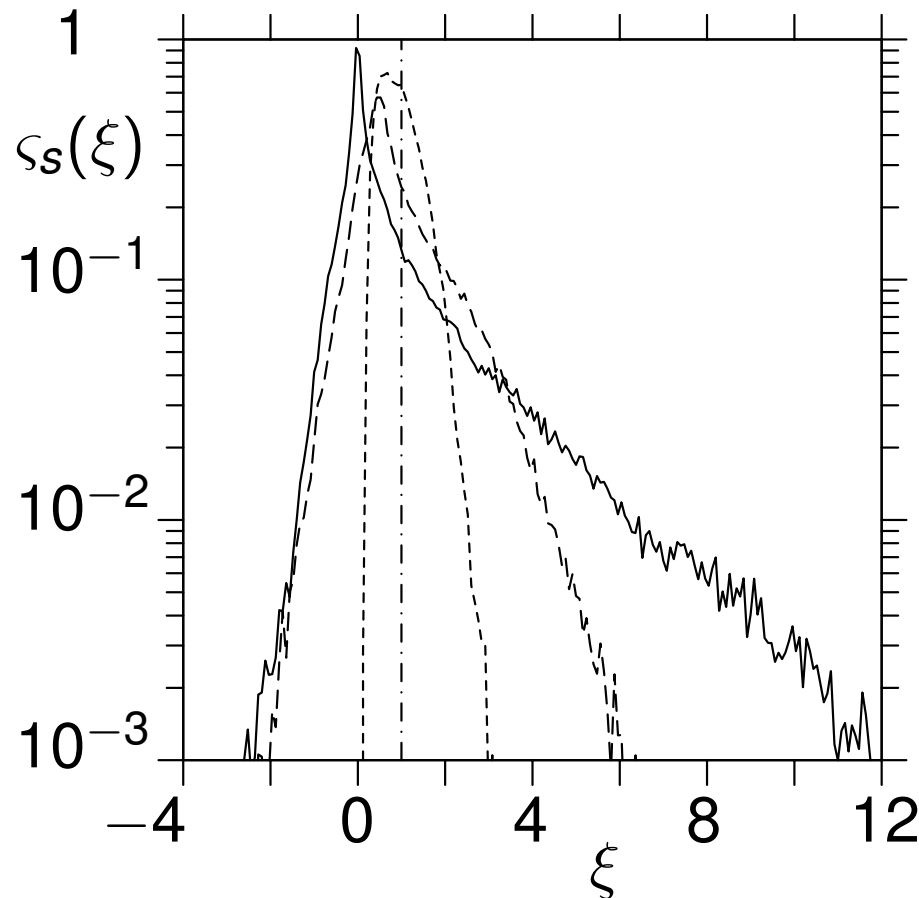


## Spatial distribution of $\Omega$

%% vs. $Ra$	$10^6$	$10^7$	$10^8$
$\Omega < 0$	25	28	31
$0 \leq \Omega < 2Nu$	58	54	51
$\Omega \geq 2Nu$	17	18	18

# Areal distribution density function of the heat flux $\Omega$

for  $Ra = 10^7$ ,  $z = 0.5H$  (—),  $z = H/(2Nu)$  (---) and  $z = 10^{-3}H$  (- - - -)

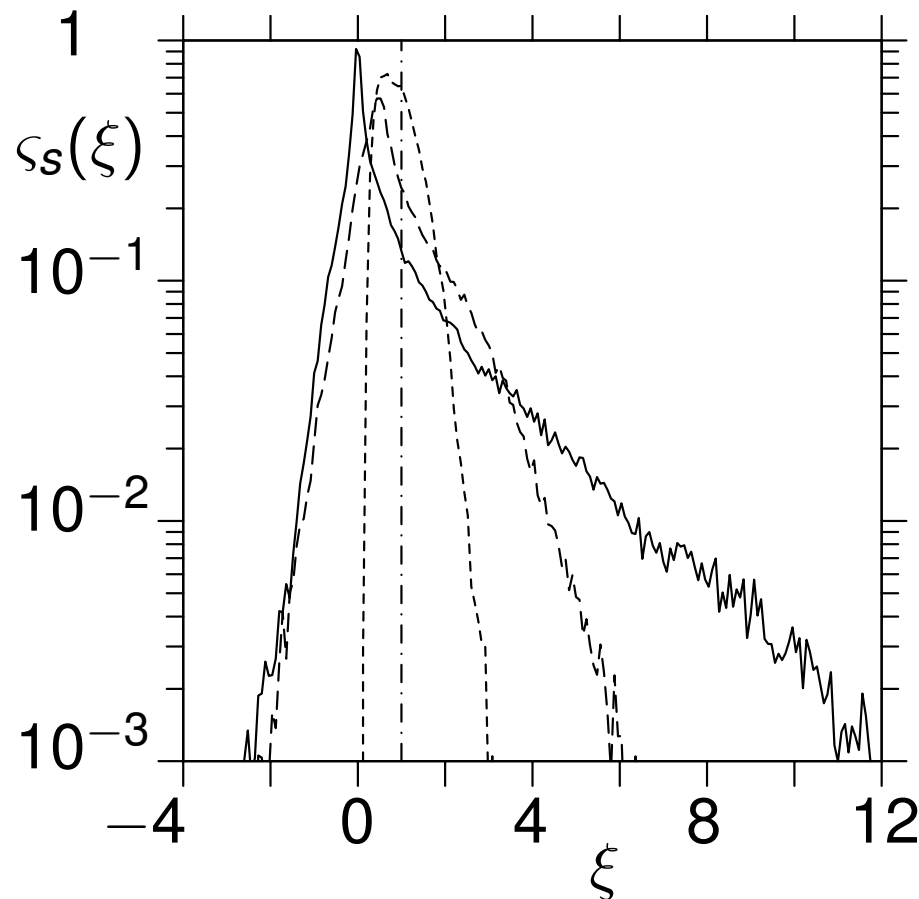


$$\varsigma_s(\xi) = \frac{1}{\delta} \left\langle \vartheta \left( \Omega - \left( \xi_i - \frac{\delta}{2} \right) Nu \right) \right\rangle_{S_z} - \frac{1}{\delta} \left\langle \vartheta \left( \Omega - \left( \xi_i + \frac{\delta}{2} \right) Nu \right) \right\rangle_{S_z}$$

indicates the portion of  $S_z$  with  
 $\Omega \in \left[ \left( \xi_i - \frac{\delta}{2} \right) Nu, \left( \xi_i + \frac{\delta}{2} \right) Nu \right]$

# Areal distribution density function of the heat flux $\Omega$

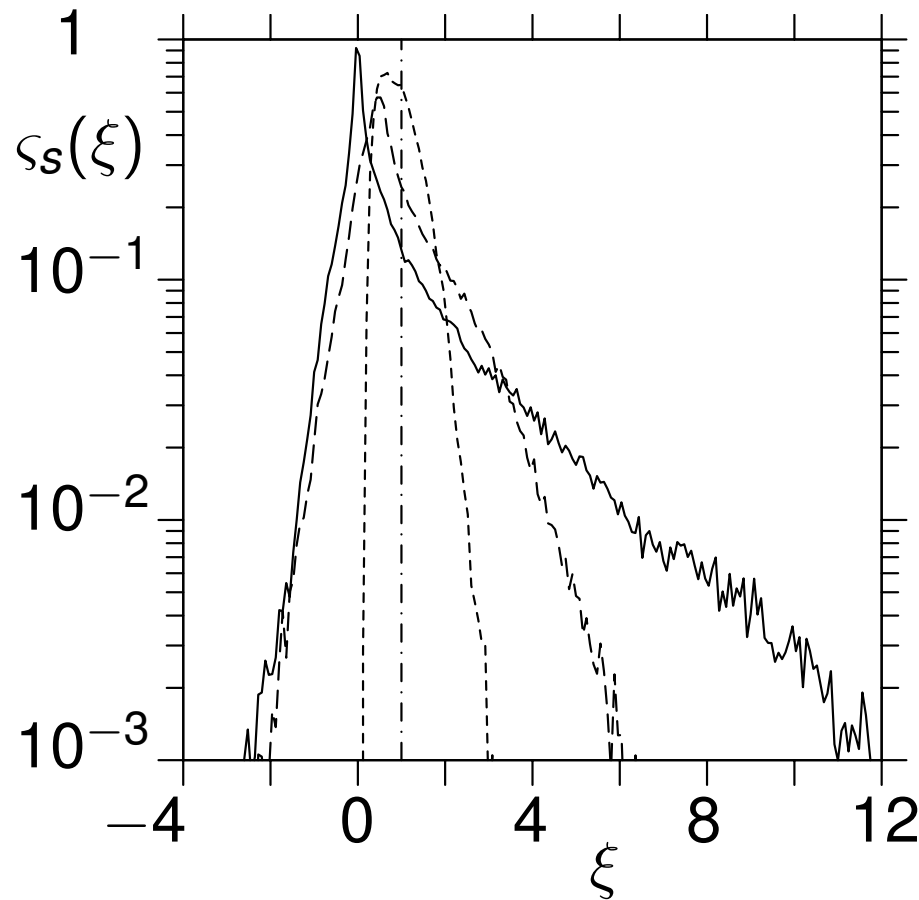
for  $Ra = 10^7$ ,  $z = 0.5H$  (—),  $z = H/(2Nu)$  (---) and  $z = 10^{-3}H$  (- - - -)



The spread of the tails in  $\Omega$ -distributions widens while moving away from the plates towards the bulk

# Areal distribution density function of the heat flux $\Omega$

for  $Ra = 10^7$ ,  $z = 0.5H$  (—),  $z = H/(2Nu)$  (---) and  $z = 10^{-3}H$  (- - - -)



## Areal distribution of $\Omega$

%% vs. $z$	0	$\frac{H}{(2Nu)}$	$\frac{H}{2}$
$\Omega < 0$	0	13	31
$0 \leq \Omega < 2Nu$	98	71	50
$\Omega \geq 2Nu$	2	16	19

# Main conclusions

## Local heat flux $\Omega$

The spread of the tails of the local heat flux distributions increases with the Rayleigh number and while moving away from the plates towards the bulk. Up to 1/3 of the fluid volume corresponds  $\Omega < 0$ . With growing  $Ra$  the zones of the highest values of the time-averaged local heat flux nestle closer to the corners.

## Turbulent background

Both, the portion of the whole domain which corresponds to the turbulent background and the contribution to the volume averaged thermal dissipation rate from the turbulent background, increase with the Rayleigh number.