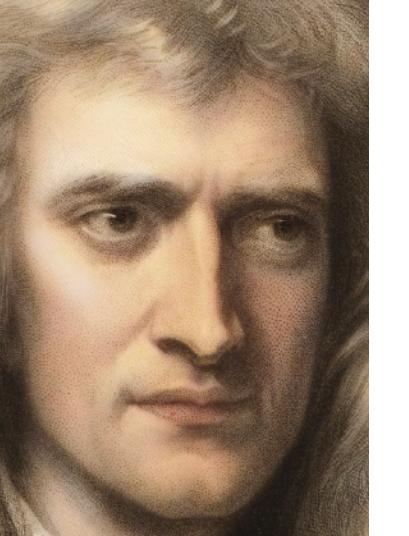
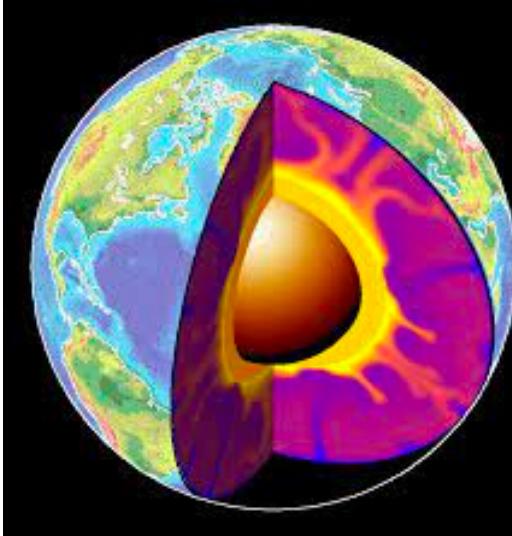


Joint ICTP-EAIFR-IUGG Workshop on Computational Geodynamics: Towards Building a New Expertise Across Africa



Theory from  to  in 3 hours or less

Richard Katz
University of Oxford

How to learn continuum mechanics

1. Study continuum mechanics;
2. Do problems with continuum mechanics;
3. Go back to 1.

[Caution: this may take your whole life.]

Outline

1. Review of Newton's laws,
displacement & velocity fields
2. Inertia in a continuum
3. Forces in a continuum (stress)
4. Rheology & deformation in 1D
5. Rheology & deformation in 3D
6. Conservation of mass
7. Full equations of motion
8. Scaling analysis & simplification
9. Work, energy & power
10. Plasticity

Newton's laws of mechanics

1. Every body continues in a state of rest or uniform motion in a straight line unless acted upon by a force.
2. **The rate of change of momentum of a body is given by the applied force.**
3. Action and reaction are equal and opposite at all times.

Applies to

- Objects more massive than atoms;
- Speeds slower than the speed of light.

Can be expressed as

$$\sum \mathbf{F} = M\mathbf{a}$$

(And usually, $\frac{dM}{dt} = 0$)

A note about notation

- *Tensors* (of second order or higher) are expressed with italic bold.
- *Vectors* (tensors of first order) are expressed with non-italic bold.
- *Scalars* are expressed with italic and have only a magnitude.
- *Indices* are integers and are expressed as subscripts.

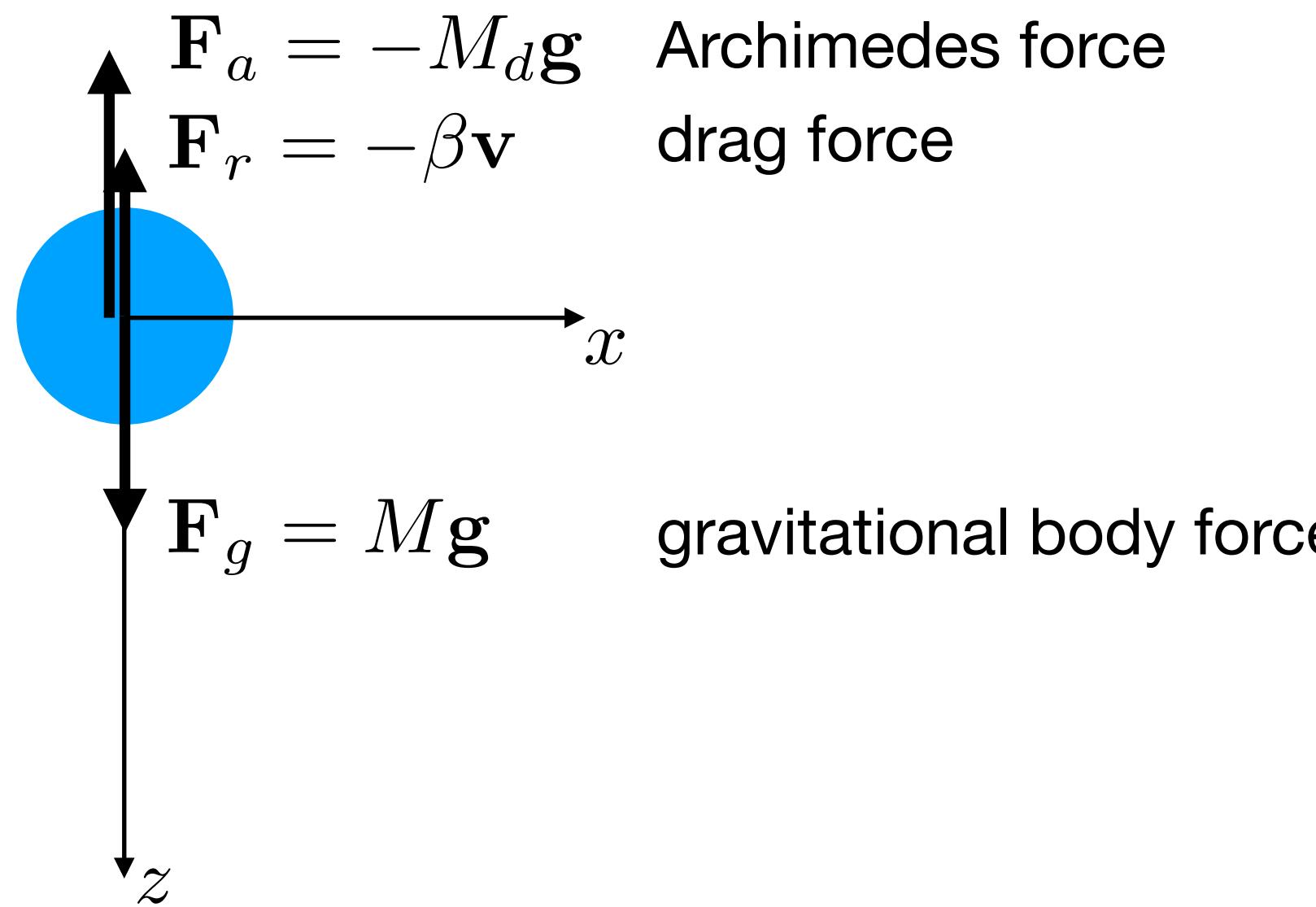
A

\mathbf{F}

F

F_i, A_{ij}

Example: ball falling through fluid



$$\sum \mathbf{F} = M \mathbf{a}$$

$$\mathbf{F}_g + \mathbf{F}_a + \mathbf{F}_r = M \mathbf{a},$$

$$\Delta_M \mathbf{g} - \beta \dot{\mathbf{x}} = M \ddot{\mathbf{x}}$$

$$\Delta_M g - \beta \dot{z} = M \ddot{z}$$

where $\Delta_M = M - M_d$

if inertia is negligible,

$$\text{Case 1: } \ddot{z} \sim 0 \Rightarrow \Delta_M g \sim \beta \dot{z} \Rightarrow \dot{z} = \frac{\Delta_M g}{\beta} \Rightarrow z(t) = \frac{\Delta_M g}{\beta} t + z_0$$

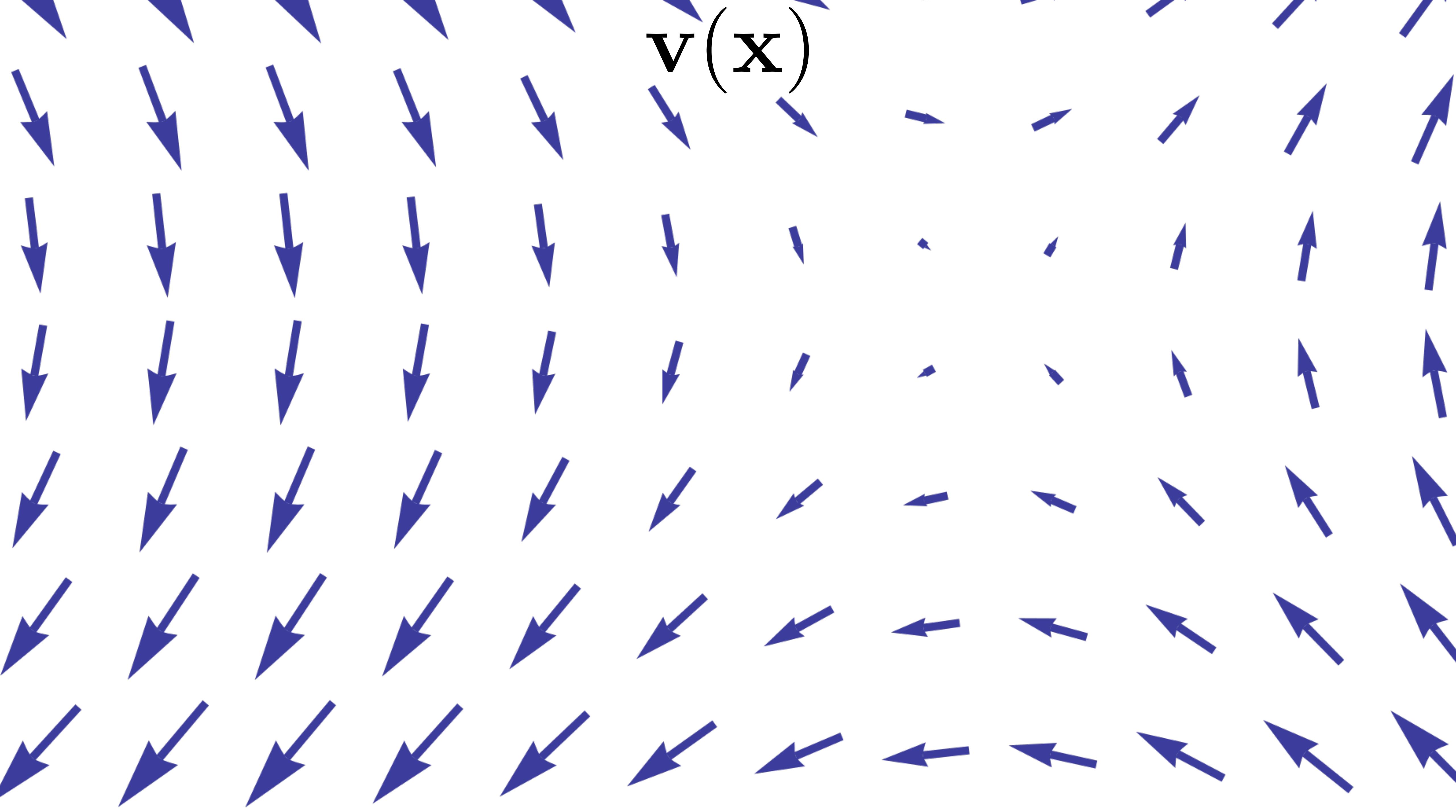
if drag is negligible,

$$\text{Case 2: } \beta \sim 0 \Rightarrow \Delta_M g \sim M \ddot{z} \Rightarrow \ddot{z} = \frac{\Delta_M}{M} g \Rightarrow z(t) = \frac{\Delta_M}{M} \frac{g}{2} t^2 + v_0 t + z_0$$

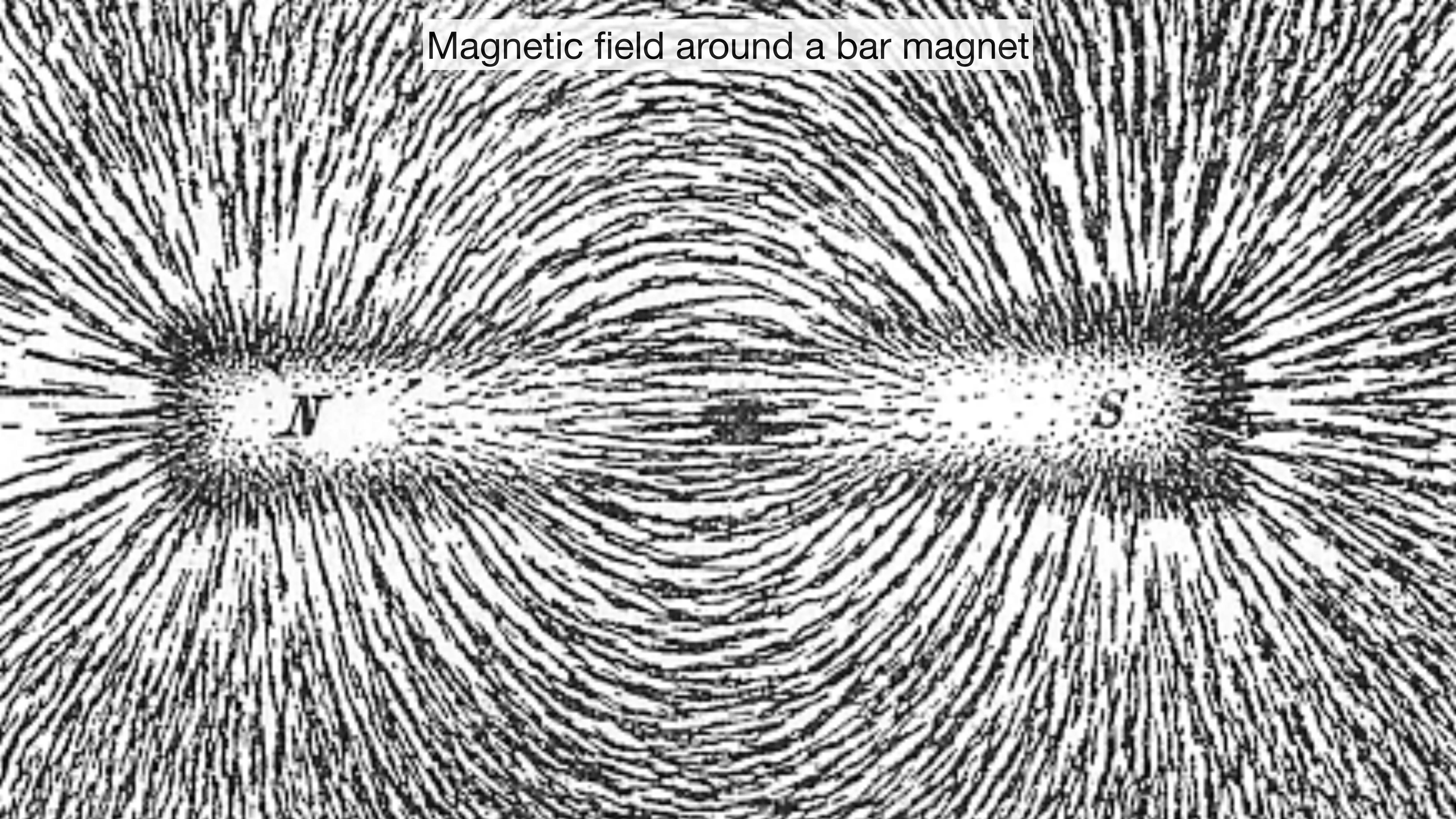
The forces, speed and acceleration “move” with the ball (obviously)

Another note about notation

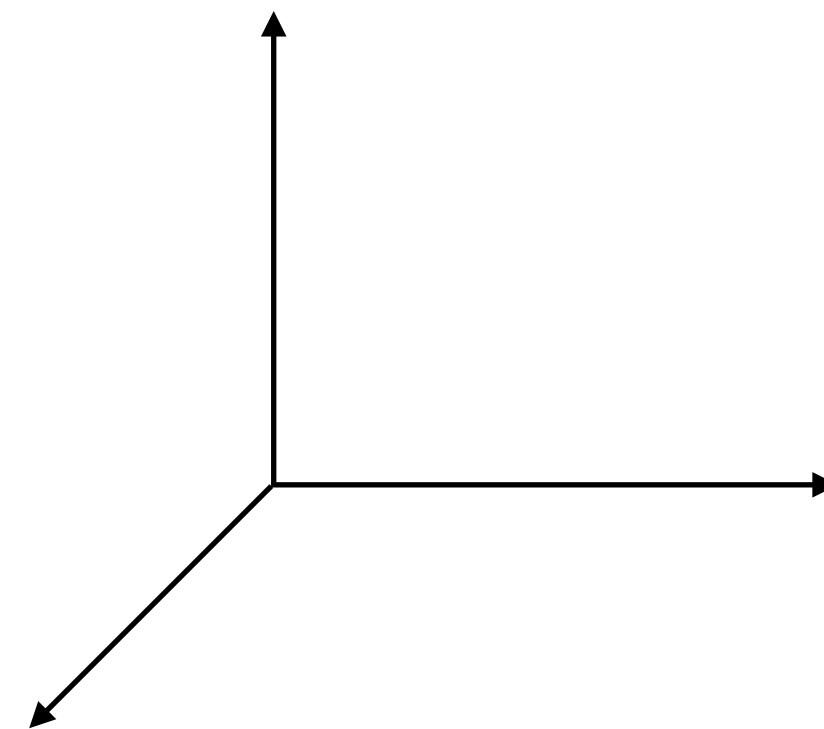
- *Tensors fields* (of second order or higher) are expressed with italic bold. $A(\mathbf{x}, t)$
- *Vectors fields* (tensors of first order) are expressed with non-italic bold. $\mathbf{F}(\mathbf{x}, t)$
- *Scalars fields* are expressed with italic and have only a magnitude. $F(\mathbf{x}, t)$
- *Indices* are integers and are expressed as subscripts. $F_i(\mathbf{x}, t), A_{ij}(\mathbf{x}, t)$

$v(x)$ 

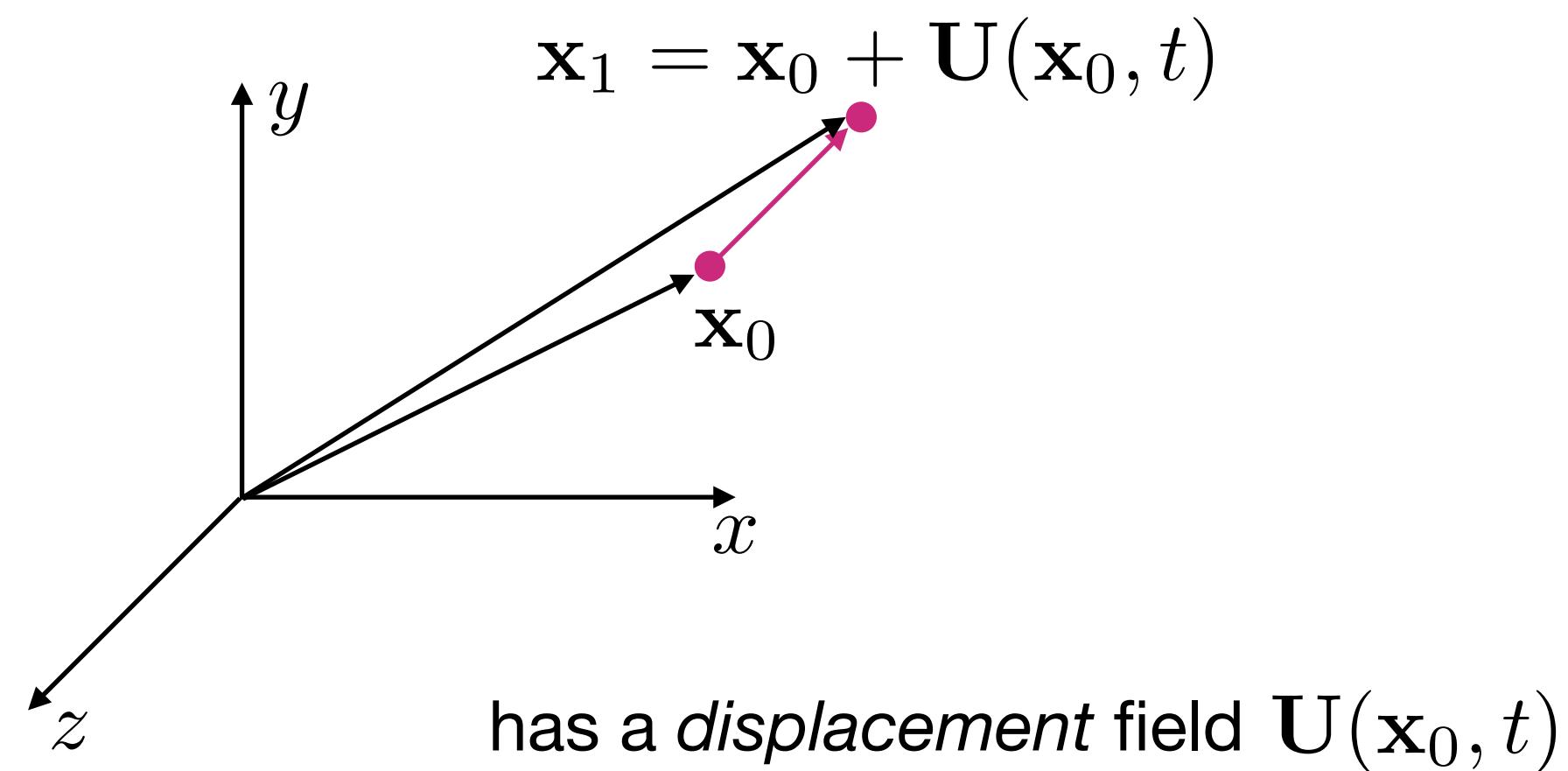
Magnetic field around a bar magnet



a continuum

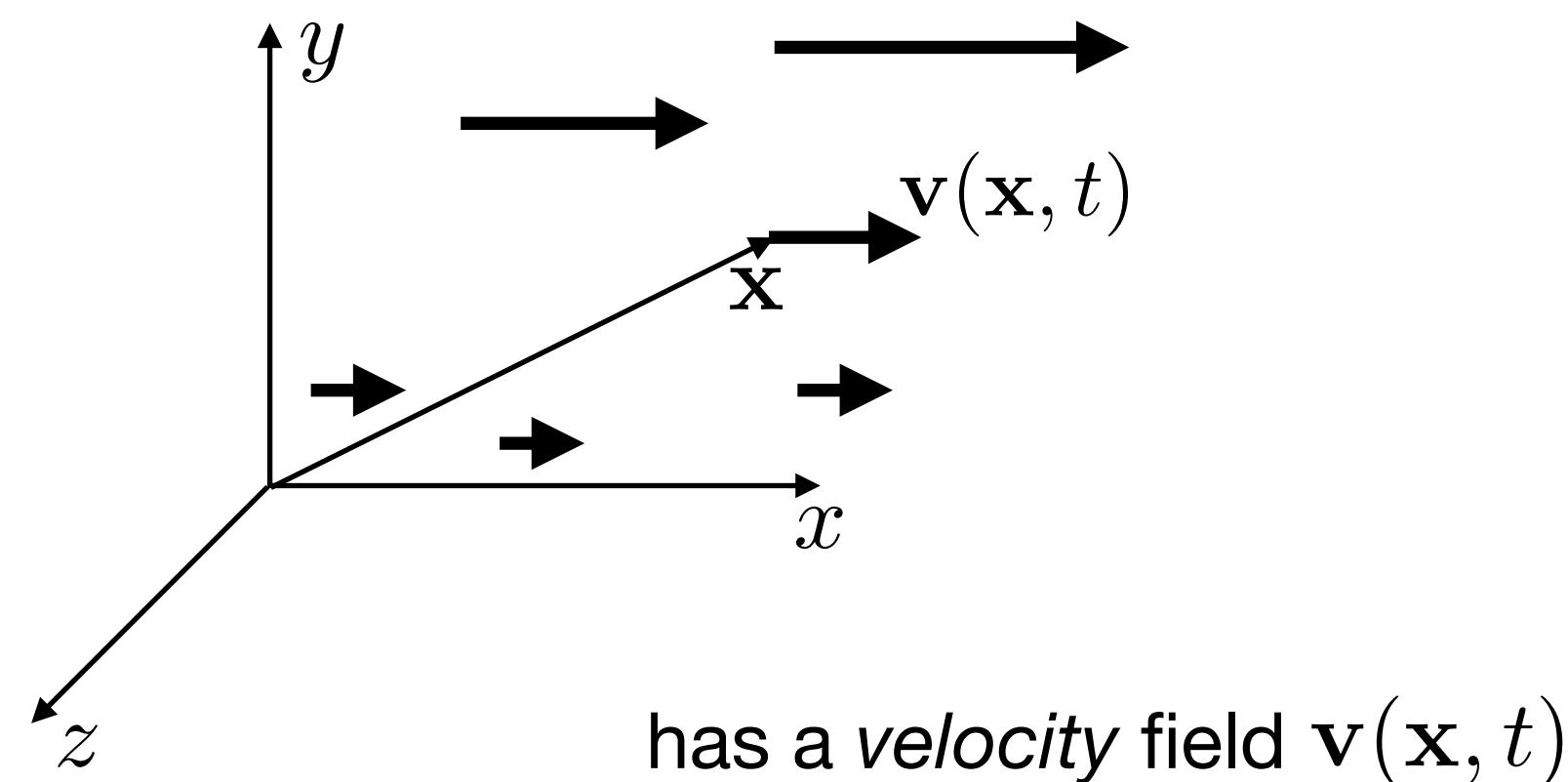


a solid continuum:



$$\mathbf{v} = \frac{d\mathbf{x}}{dt} = \frac{\partial \mathbf{U}}{\partial t}$$

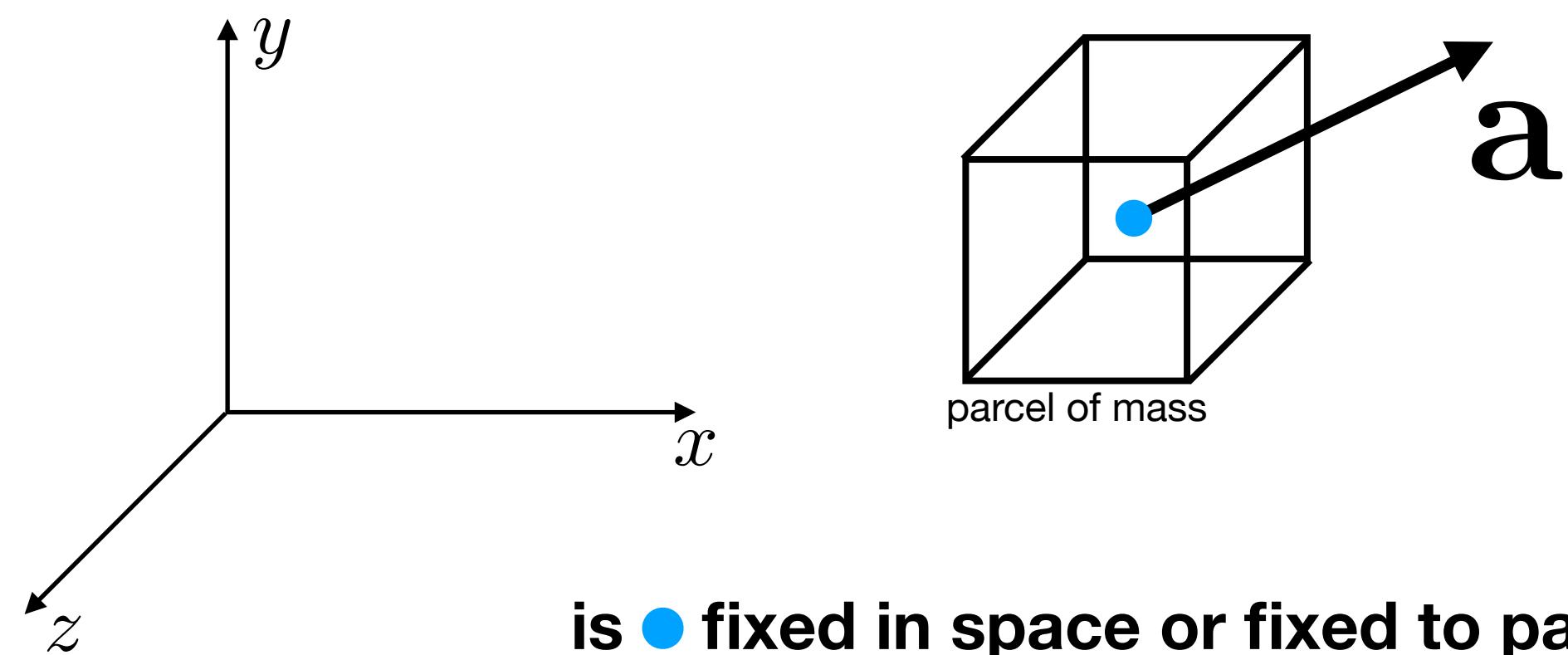
a fluid continuum:



$$\mathbf{v} = \frac{d\mathbf{x}}{dt}$$

Inertia in a continuum

(very small) parcel of mass in a continuum



$$\sum \mathbf{F} = M\mathbf{a}$$

acceleration of what??

$$\sum \mathbf{F} = M \frac{D\mathbf{v}}{Dt}$$

Lagrangian derivative following the parcel

$$\begin{aligned} \frac{D\mathbf{v}}{Dt} &= \frac{d}{dt}\mathbf{v}(\mathbf{x}(t), t) = \frac{\partial \mathbf{v}}{\partial x} \frac{dx}{dt} + \frac{\partial \mathbf{v}}{\partial y} \frac{dy}{dt} + \frac{\partial \mathbf{v}}{\partial z} \frac{dz}{dt} + \frac{\partial \mathbf{v}}{\partial t} \\ &= \left(\frac{dx}{dt}, \frac{dy}{dt}, \frac{dz}{dt} \right) \cdot \left(\frac{\partial \mathbf{v}}{\partial x}, \frac{\partial \mathbf{v}}{\partial y}, \frac{\partial \mathbf{v}}{\partial z} \right) + \frac{\partial \mathbf{v}}{\partial t} \\ &= \mathbf{v} \cdot \nabla \mathbf{v} + \frac{\partial \mathbf{v}}{\partial t} \end{aligned}$$

Material derivative + Eulerian derivative at a point in space

The Lagrangian & Eulerian derivatives

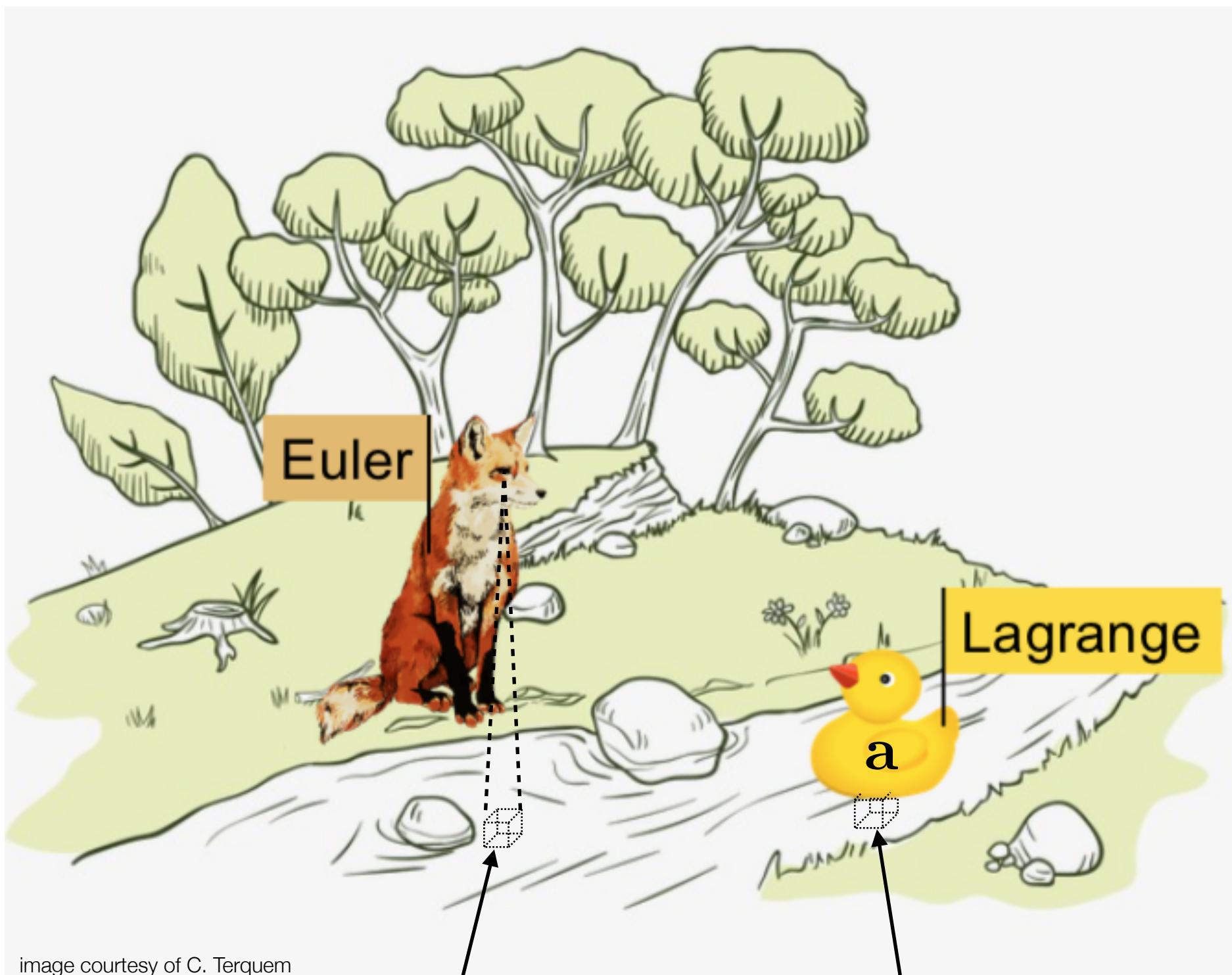


image courtesy of C. Terquem

$$\frac{\partial Q}{\partial t}$$

$$\frac{DQ}{Dt}$$

Performing a Taylor series expansion to first order in δt :

$$Q(\mathbf{r} + \mathbf{v}\delta t, t + \delta t) = Q(\mathbf{r}, t) + \delta t \frac{\partial Q(\mathbf{r}, t)}{\partial t} + \mathbf{v}\delta t \cdot \nabla Q(\mathbf{r}, t),$$

$$\frac{DQ}{Dt} = \lim_{\delta t \rightarrow 0} \frac{Q(\mathbf{r} + \mathbf{v}\delta t, t + \delta t) - Q(\mathbf{r}, t)}{\delta t}.$$



$$\frac{DQ}{Dt} = \frac{\partial Q}{\partial t} + \mathbf{v} \cdot \nabla Q.$$

Another note about (index) notation

- A vector has one free (unrepeated) index.

$$F_i$$

- The gradient operator ∇ is a vector.

$$\frac{\partial}{\partial x_i}$$

- A second-rank tensor has two free indicies.

$$A_{ij}$$

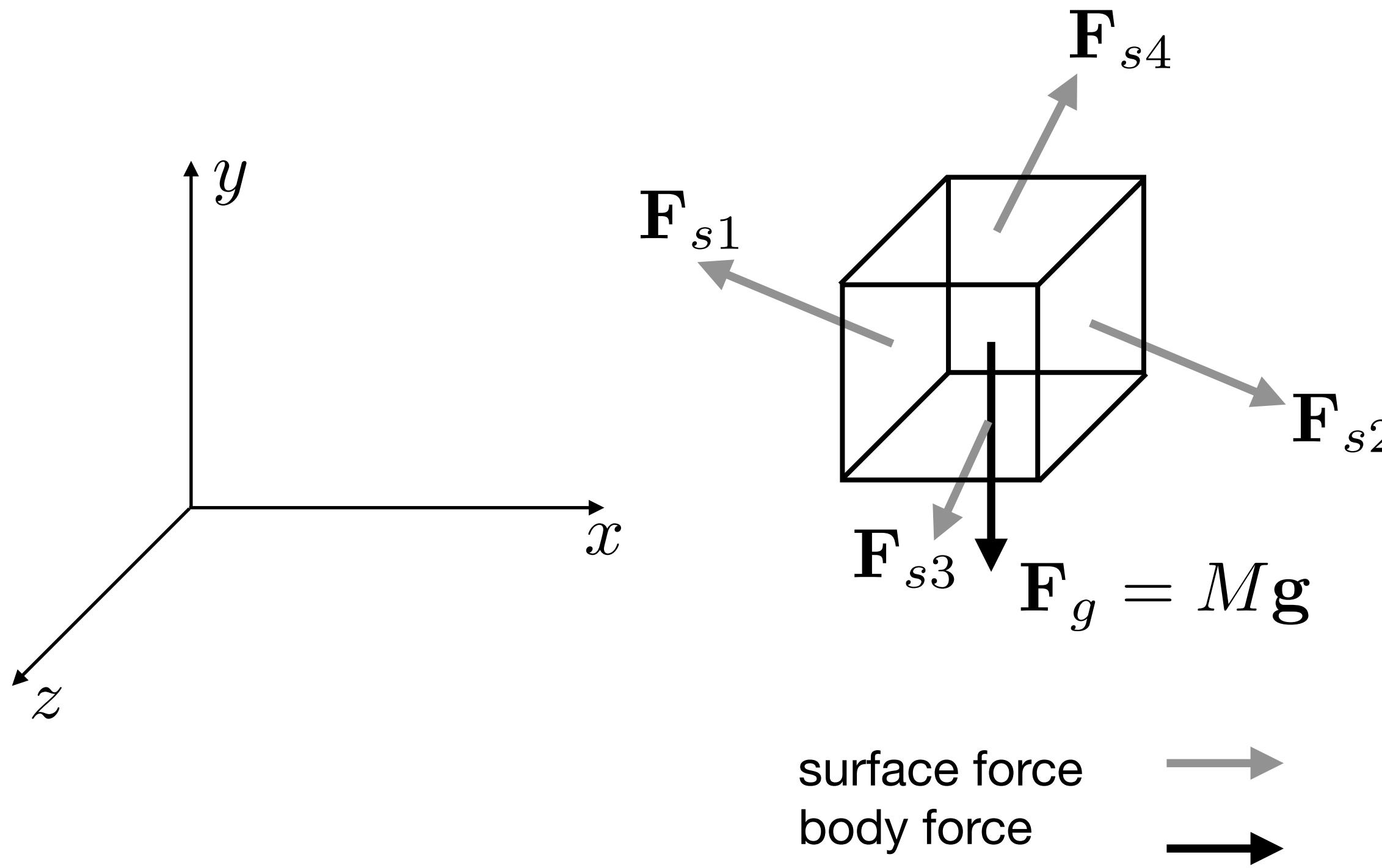
- Summation is indicated by a repeated index (Einstein summation convention).

$$F_i A_{ij} = \sum_i F_i A_{ij}$$

$$\frac{\partial v_i}{\partial x_i} = \sum_i \frac{\partial v_i}{\partial x_i}$$

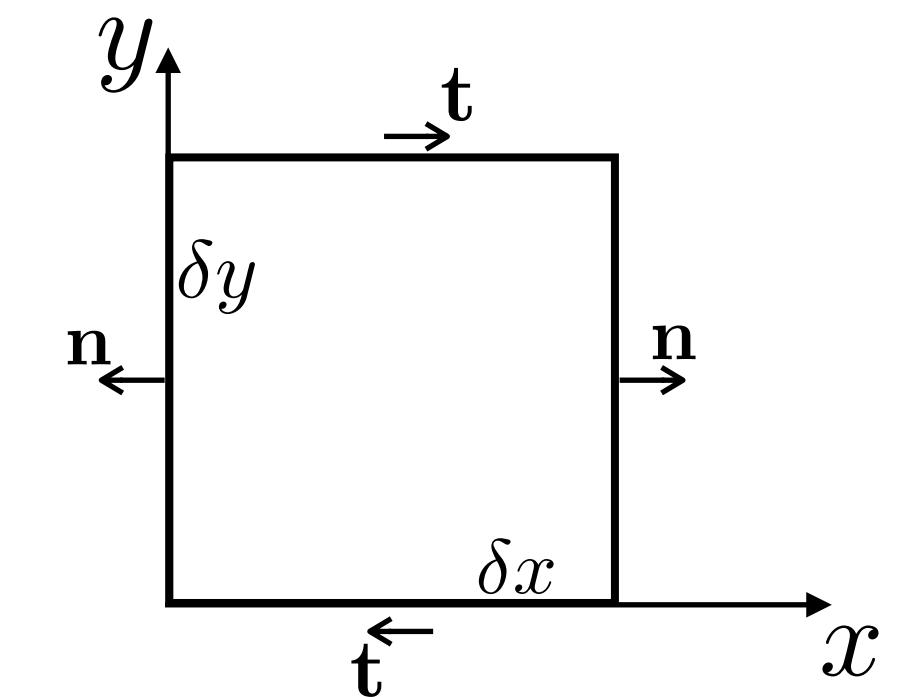
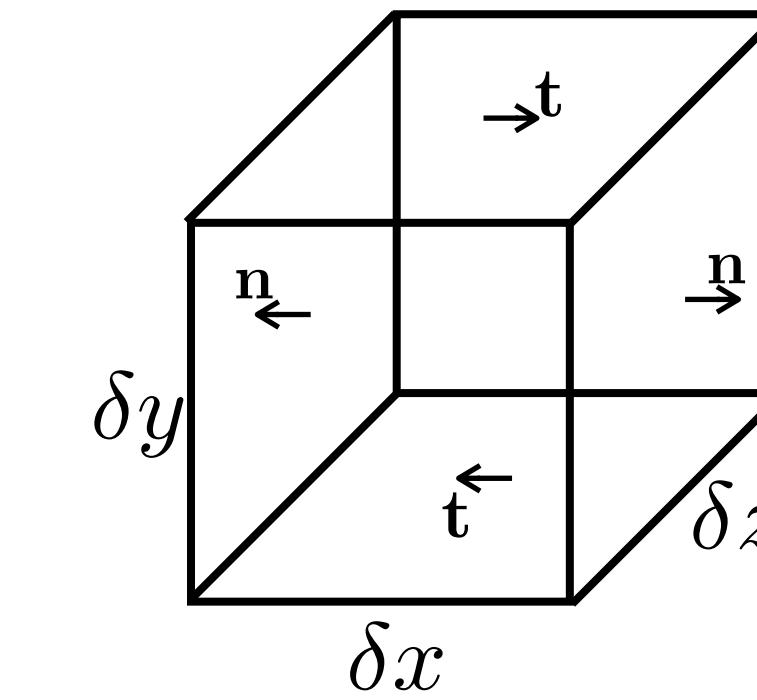
Forces in a continuum

forces on a (very small) parcel of mass in a continuum – in the x direction



$$\left(\sum \mathbf{F} = M \frac{D\mathbf{v}}{Dt} \right) \cdot \hat{\mathbf{x}}$$

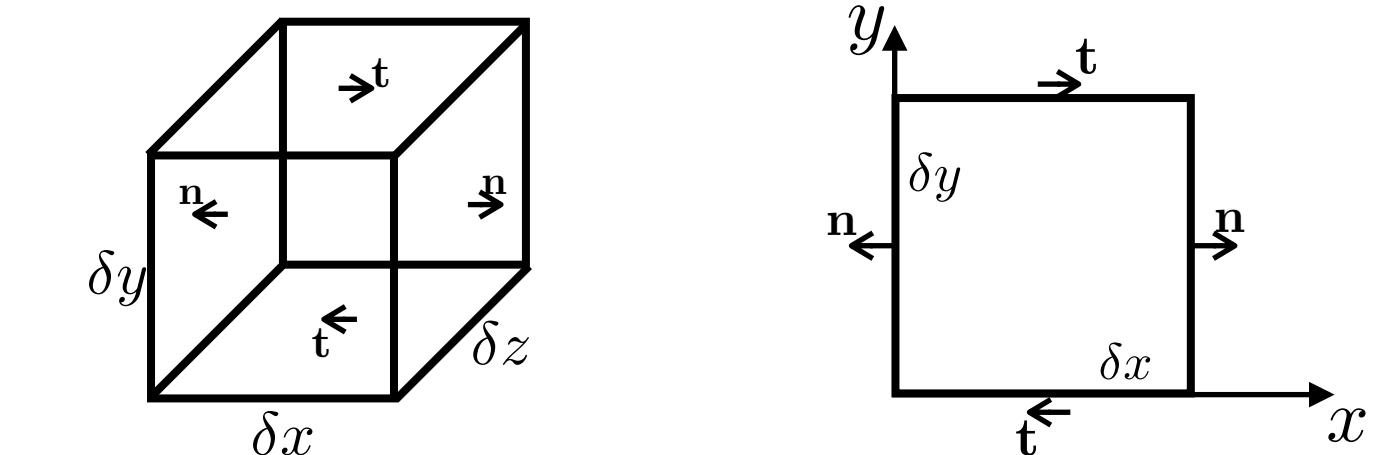
Define unit normal \mathbf{n} and tangent \mathbf{t} vectors



$$\sum \mathbf{F} \cdot \hat{\mathbf{x}} = (\mathbf{F}_{s2} \cdot \mathbf{n})|_{x+\delta x} - (\mathbf{F}_{s1} \cdot \mathbf{n})|_x + (\mathbf{F}_{s4} \cdot \mathbf{t})|_{y+\delta y} - (\mathbf{F}_{s3} \cdot \mathbf{t})|_y + \dots$$

forces on a (very small) parcel of mass in a continuum – in the x direction

$$\begin{aligned}\sum \mathbf{F} \cdot \hat{\mathbf{x}} &= (\mathbf{F}_{s2} \cdot \mathbf{n})|_{x+\delta x} - (\mathbf{F}_{s1} \cdot \mathbf{n})|_x + (\mathbf{F}_{s4} \cdot \mathbf{t})|_{y+\delta y} - (\mathbf{F}_{s3} \cdot \mathbf{t})|_y + \dots \\ &= [\sigma_{xx}(x + \delta x) - \sigma_{xx}(x)]\delta y \delta z + [\sigma_{yx}(y + \delta y) - \sigma_{yx}(y)]\delta x \delta z + \dots\end{aligned}$$



$$\sigma \equiv \frac{\mathbf{F} \cdot (\mathbf{n}, \mathbf{t})}{\text{area}} \quad \text{define the (normal, shear) stress as } \textit{force per unit area!!} \text{ Units: Pascals = N/m}^2$$

$$\left(\sum \mathbf{F} = M \frac{D\mathbf{v}}{Dt} \right) \cdot \hat{\mathbf{x}}$$

$$[\sigma_{xx}(x + \delta x) - \sigma_{xx}(x)]\delta y \delta z + [\sigma_{yx}(y + \delta y) - \sigma_{yx}(y)]\delta x \delta z + [\sigma_{zx}(z + \delta z) - \sigma_{zx}(z)]\delta x \delta y + Mg_x = M \frac{Dv_x}{Dt}$$

Taylor expand: $\sigma_{xx}(x + \delta x) = \sigma_{xx}(x) + \delta x \frac{\partial \sigma_{xx}}{\partial x} + O(\delta x^2)$ for example

Substitute and divide by $V = \delta x \delta y \delta z$ then take $\lim_{\delta x, \delta y, \delta z \rightarrow 0}$

$$\Rightarrow \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{yx}}{\partial y} + \frac{\partial \sigma_{zx}}{\partial z} + \frac{M}{V} g_x = \frac{M}{V} \frac{Dv_x}{Dt}$$

forces on a (very small) parcel of mass in a continuum

In the x -direction:

$$\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{yx}}{\partial y} + \frac{\partial \sigma_{zx}}{\partial z} + \frac{M}{V} g_x = \frac{M}{V} \frac{Dv_x}{Dt}$$

$$\frac{\partial \sigma_{ix}}{\partial x_i} + \rho g_x = \rho \frac{Dv_x}{Dt} \quad \rho \equiv \frac{M}{V}, \text{ density}$$

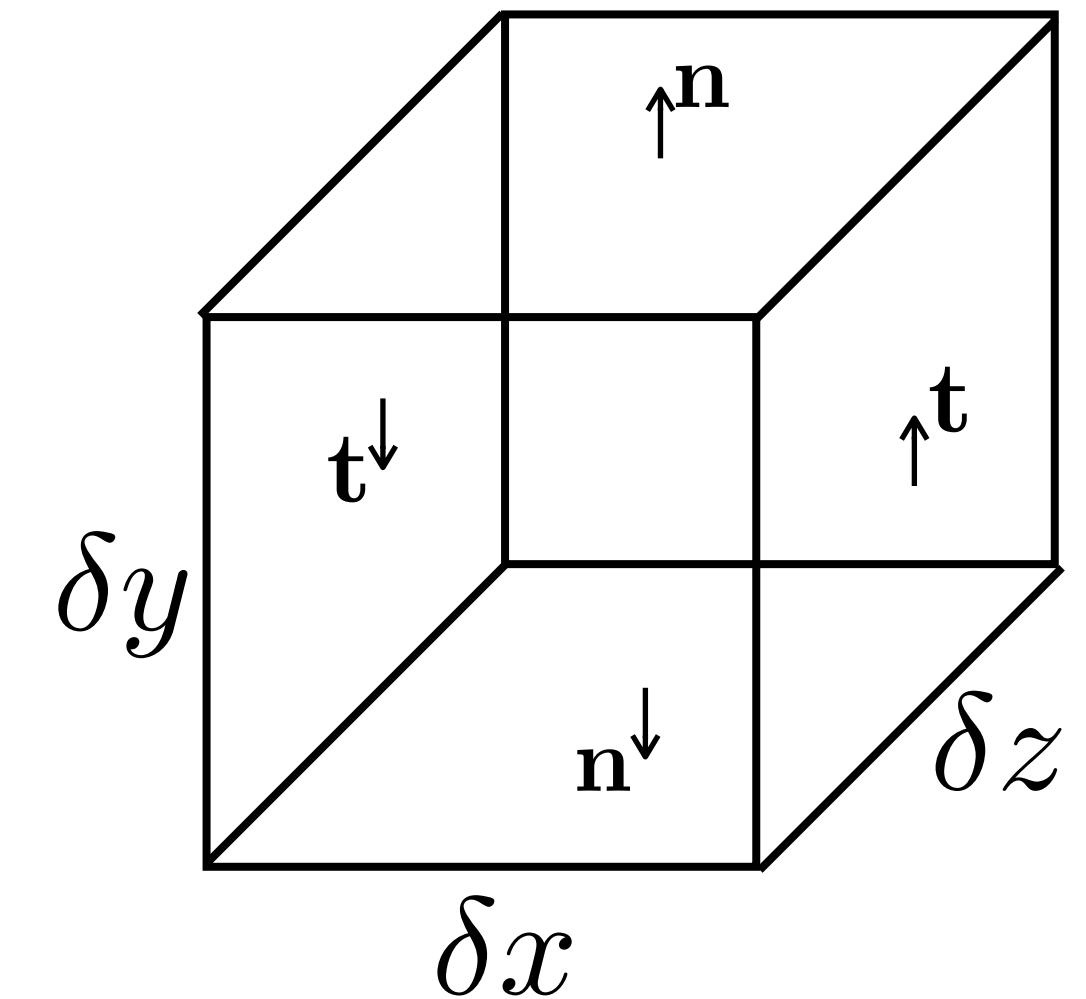
In the y -direction:

$$\frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_{zy}}{\partial z} + \frac{M}{V} g_y = \frac{M}{V} \frac{Dv_y}{Dt}$$

$$\frac{\partial \sigma_{iy}}{\partial x_i} + \rho g_y = \rho \frac{Dv_y}{Dt}$$

In the j^{th} -direction:

$$\frac{\partial \sigma_{ij}}{\partial x_i} + \rho g_j = \rho \frac{Dv_j}{Dt} \quad \text{or} \quad \nabla \cdot \boldsymbol{\sigma} + \rho \mathbf{g} = \rho \frac{D\mathbf{v}}{Dt}$$



The Cauchy momentum equation for any continuum

the Cauchy momentum equation

$$\sum \mathbf{F} = M\mathbf{a} \quad \Rightarrow \quad \nabla \cdot \boldsymbol{\sigma} + \mathbf{f}_b = \rho \frac{D\mathbf{v}}{Dt}$$

net surface force **body force (e.g., gravity)** **Lagrangian acceleration**

Units: Newtons, i.e., kg-m/s²
[Force]

Applies to the centre of mass
of a rigid particle

Units: N/m³, i.e., kg/m²s²
[Force per unit volume]

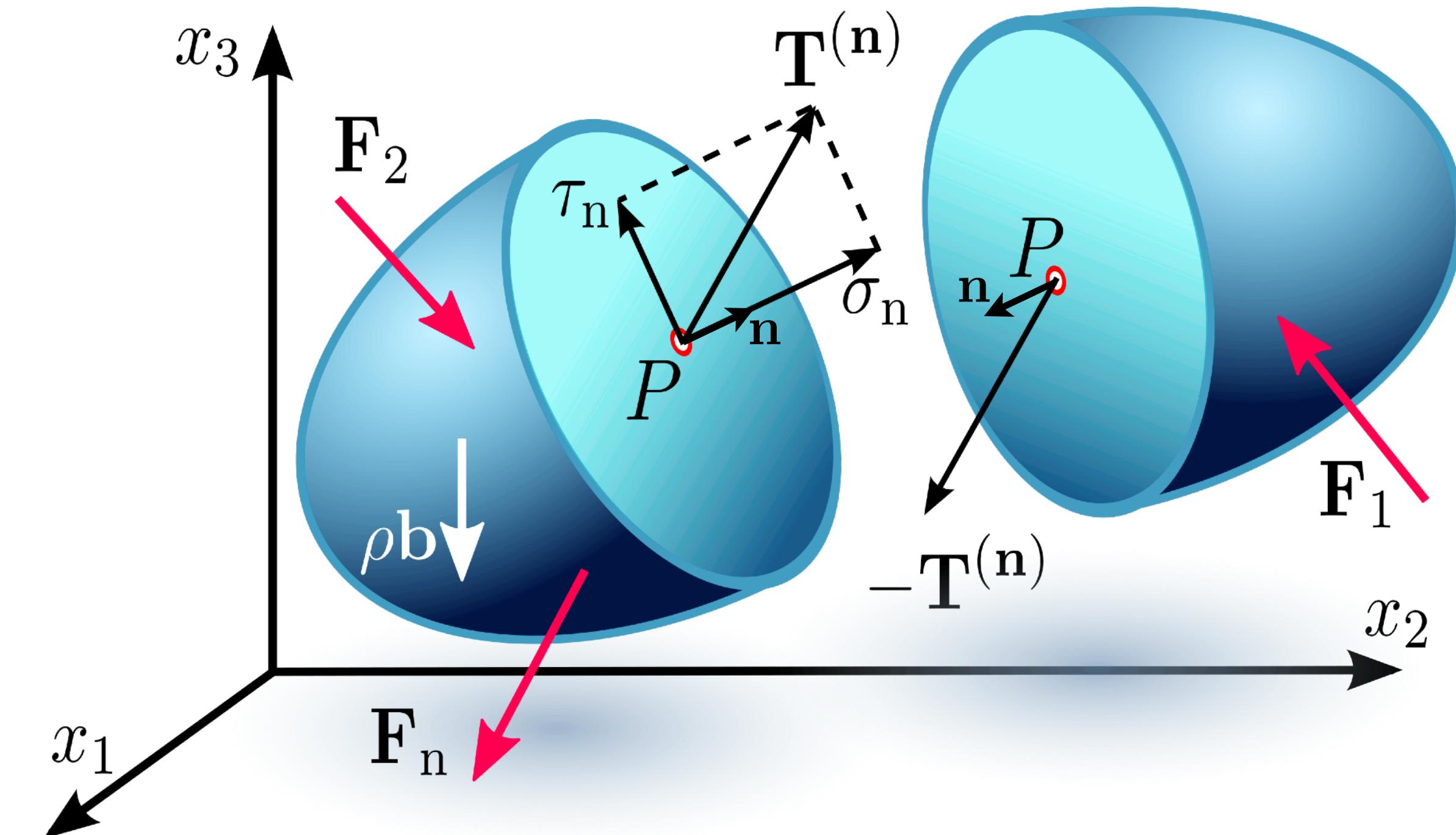
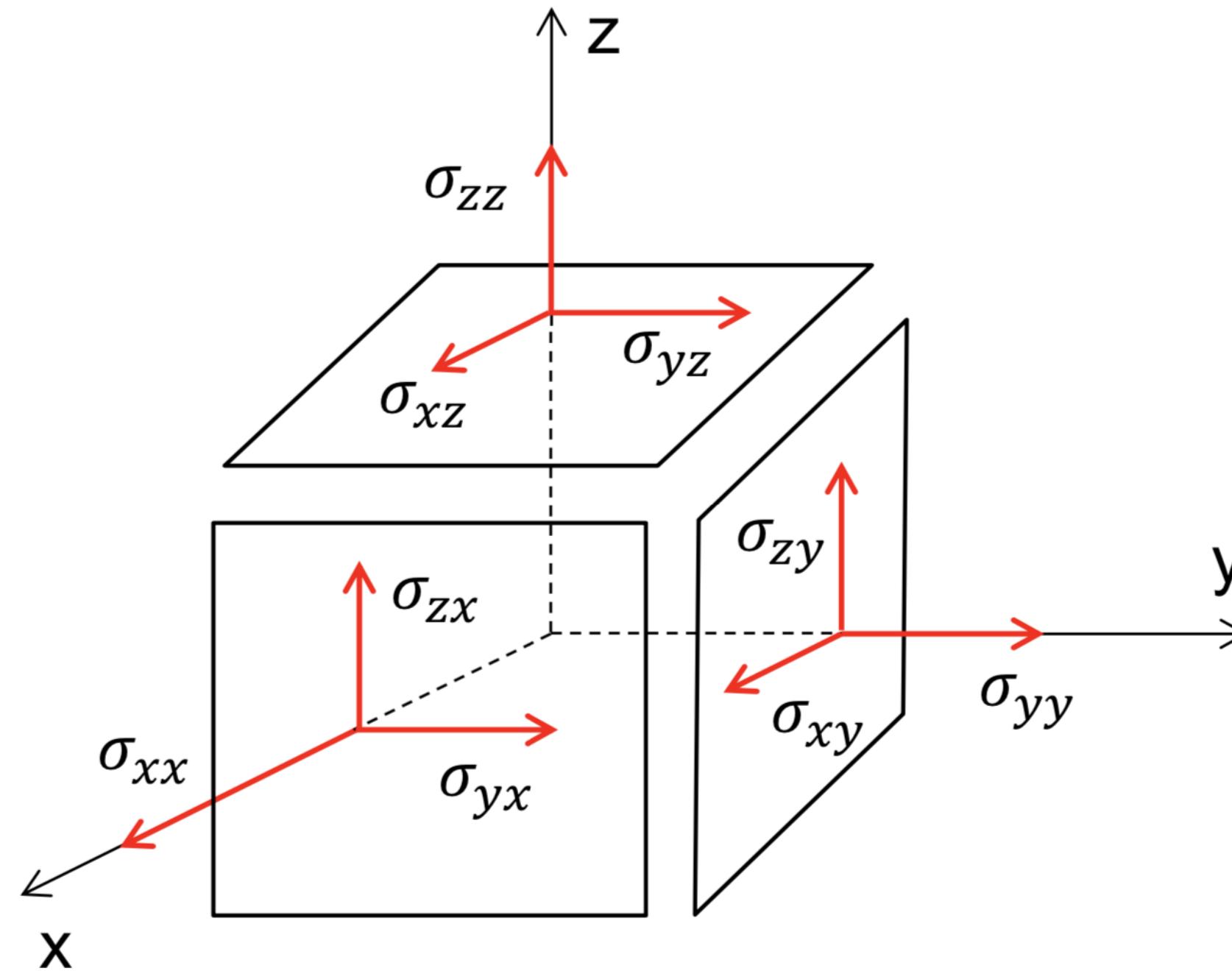
Applies to all continua
(e.g., fluids, solids, complex)

Conservation of angular momentum

We learn that the stress tensor must be symmetrical

$$\sigma_{ij} = \sigma_{ji}$$

the Cauchy stress tensor



The force per unit area at a point P is given by the **traction vector** $\mathbf{T}^{(n)}$

Given the stress tensor,
it is easy to compute traction
on any surface in continuum
or on boundary

$$\mathbf{T} = \boldsymbol{\sigma} \cdot \mathbf{n}$$

$$T_i = \sigma_{ij} n_j$$

$$T_x = \sigma_{xx} n_x + \sigma_{xy} n_y + \sigma_{xz} n_z,$$

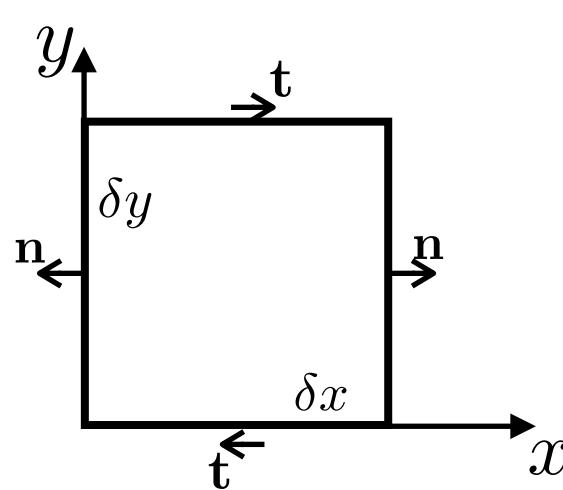
$$T_y = \sigma_{yx} n_x + \sigma_{yy} n_y + \sigma_{yz} n_z,$$

$$T_z = \sigma_{zx} n_x + \sigma_{zy} n_y + \sigma_{zz} n_z.$$

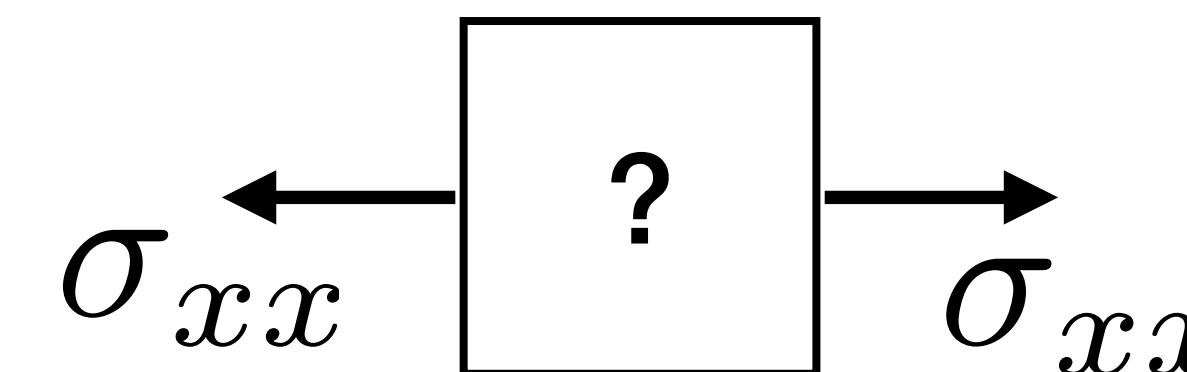
Cauchy reciprocal theorem: $\mathbf{T}^{(n)} = -\mathbf{T}^{(-n)}$ (which is Newton's 3rd law!)

Rheology & deformation (scalar version)

extension in one dimension (no body force, no acceleration)



Cauchy momentum equation becomes $\frac{\partial \sigma_{xx}}{\partial x} = 0$



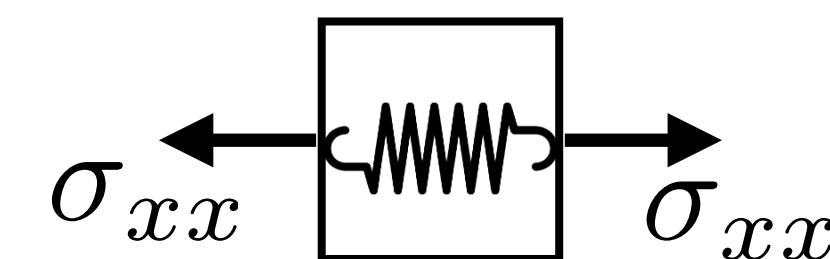
Solid (elastic) response:

$$\sigma = \mu(L - L_0)/L_0$$

Elastic modulus μ Units: Pa

Define strain: $\epsilon \equiv \frac{L - L_0}{L_0} = \frac{\partial U}{\partial x}$

$$\sigma = \mu \epsilon$$



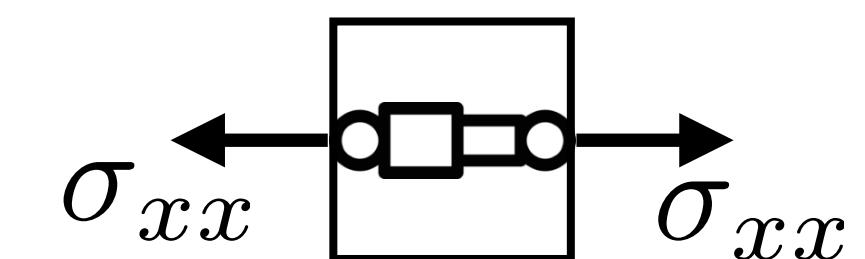
Fluid (viscous) response:

$$\sigma = \eta \frac{\partial}{\partial t} \frac{L - L_0}{L_0} = \eta (\dot{L}/L_0)$$

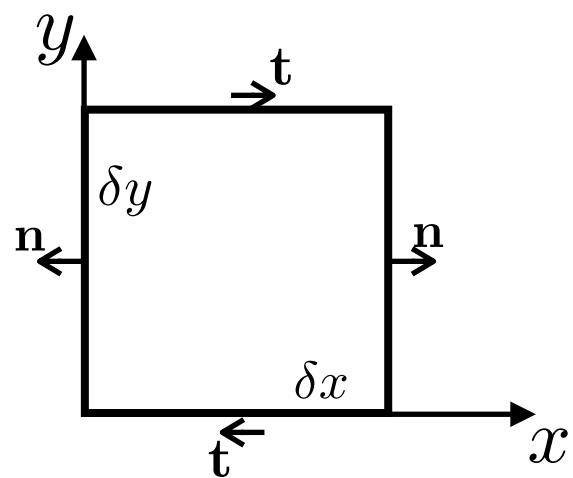
Viscosity η Units: Pa-s

Define strain rate: $\dot{\epsilon} \equiv \frac{\partial}{\partial t} \frac{L - L_0}{L_0} = \frac{\partial}{\partial t} \frac{\partial U}{\partial x} = \frac{\partial v}{\partial x}$

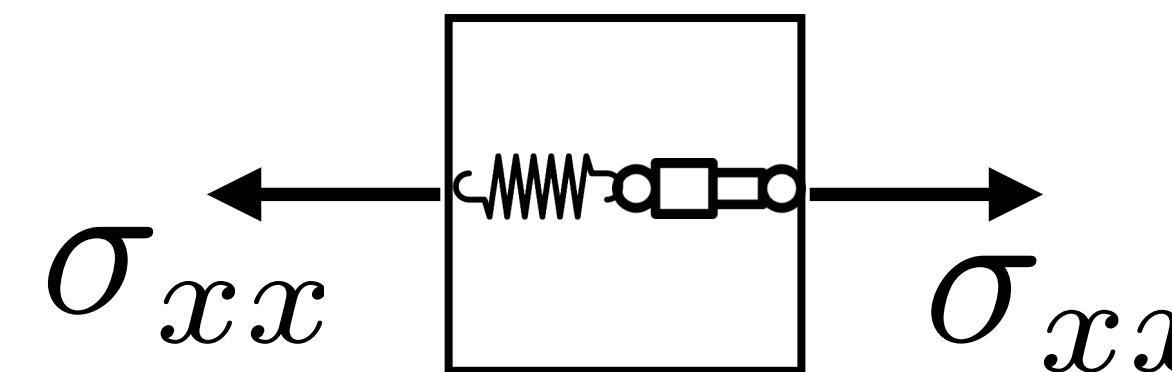
$$\sigma = \eta \dot{\epsilon}$$



extension in one dimension (no body force, no acceleration)



Cauchy momentum equation becomes $\frac{\partial \sigma_{xx}}{\partial x} = 0$



Visco-elastic response

Strain is additive:

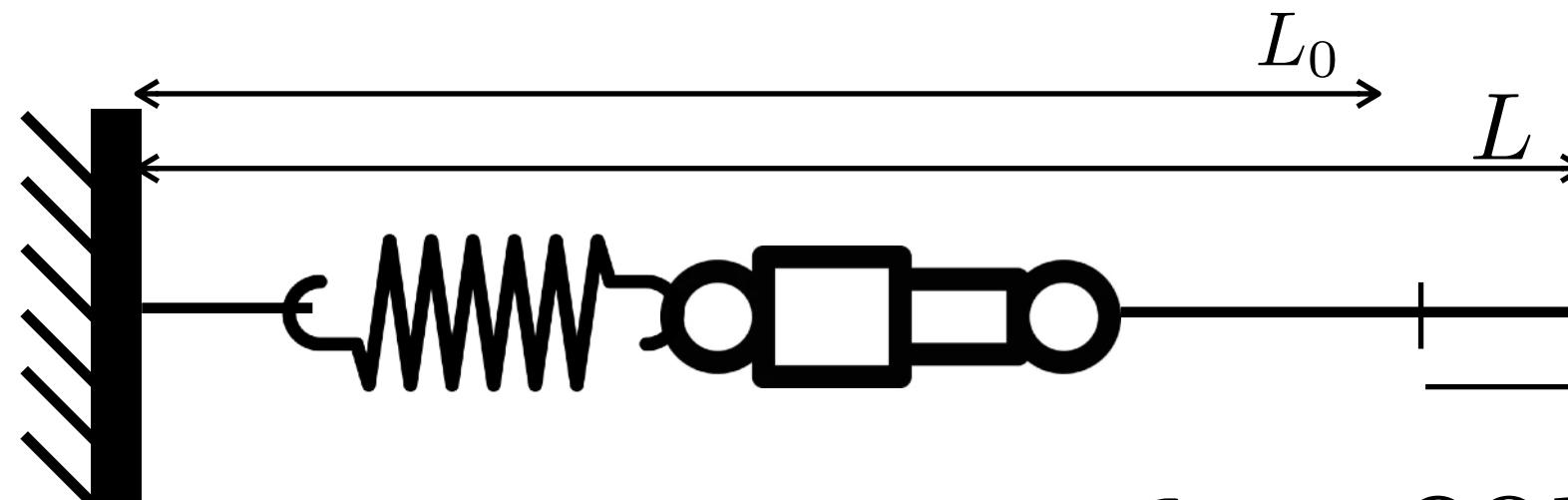
$$\epsilon = \epsilon^{\text{vi}} + \epsilon^{\text{el}} \quad \Rightarrow \quad \dot{\epsilon} = \dot{\epsilon}^{\text{vi}} + \dot{\epsilon}^{\text{el}}$$

elastic: $\sigma = \mu \epsilon^{\text{el}}$

viscous: $\sigma = \eta \dot{\epsilon}^{\text{vi}}$

$$\begin{aligned} \dot{\epsilon} &= \frac{\sigma}{\eta} + \frac{\dot{\sigma}}{\mu}, \\ &= \left(\frac{1}{\eta} + \frac{1}{\mu} \frac{d}{dt} \right) \sigma \end{aligned}$$

a viscoelastic problem (no body force, no acceleration)



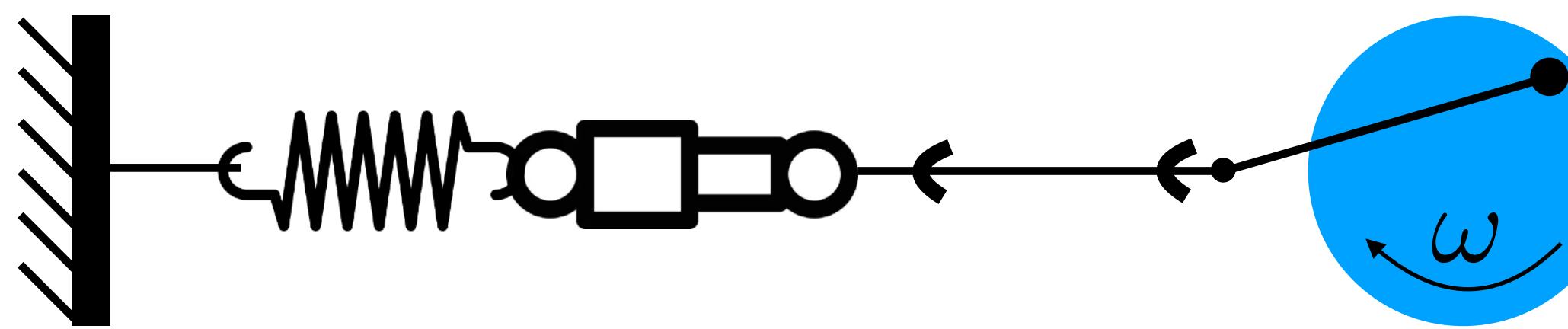
$$\epsilon = \text{const} = (L - L_0)/L_0$$

$$\dot{\epsilon} = \left(\frac{1}{\eta} + \frac{1}{\mu} \frac{d}{dt} \right) \sigma = 0 \quad \Rightarrow \quad \frac{d\sigma}{dt} = -\frac{\mu}{\eta} \sigma$$

Solution: $\sigma = \sigma_0 e^{-\mu t / \eta},$ where: $\sigma_0 = \mu \epsilon_0,$
 $= \sigma_0 e^{-t / \tau_{\text{Maxwell}}}$ $\tau_{\text{Maxwell}} \equiv \eta / \mu$

Maxwell time is the timescale for relaxation of elastic stress by viscous flow

a viscoelastic problem (no body force, no acceleration)



$$\epsilon = \operatorname{Re} (A e^{i\omega t}) = A \cos \omega t$$

To solve: $\dot{\epsilon} = \left(\frac{1}{\eta} + \frac{1}{\mu} \frac{d}{dt} \right) \sigma$

$$\sigma = \frac{A\mu}{\sqrt{1 + De^{-2}}} \sin (\omega t + \Phi)$$

Guess: $\sigma = \operatorname{Re} (ae^{i\omega t})$ with complex a .

Substitute & simplify: $i\omega A = \left(\frac{1}{\eta} + \frac{i\omega}{\mu} \right) a$

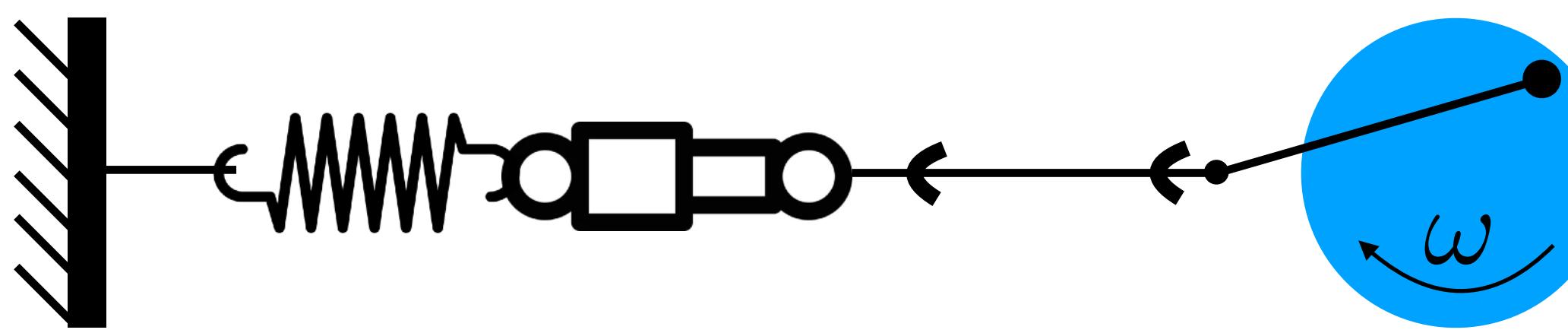
Solution: $\sigma = \operatorname{Re} \left(\frac{i\omega A}{1/\eta + i\omega/\mu} e^{i\omega t} \right)$

where
$$\begin{cases} \Phi \equiv \tan^{-1} De \\ De \equiv \frac{\eta/\mu}{1/\omega} \equiv 2\pi \frac{\tau_{\text{Maxwell}}}{\tau_{\text{forcing}}} \end{cases}$$

$$\tau_{\text{Maxwell}} \ll \tau_{\text{forcing}}, \quad De \ll 1 \quad \Rightarrow \quad \sigma \sim A\omega\eta \sin \omega t$$

$$\tau_{\text{Maxwell}} \gg \tau_{\text{forcing}}, \quad De \gg 1 \quad \Rightarrow \quad \sigma \sim A\mu \cos \omega t$$

a viscoelastic problem (no body force, no acceleration)



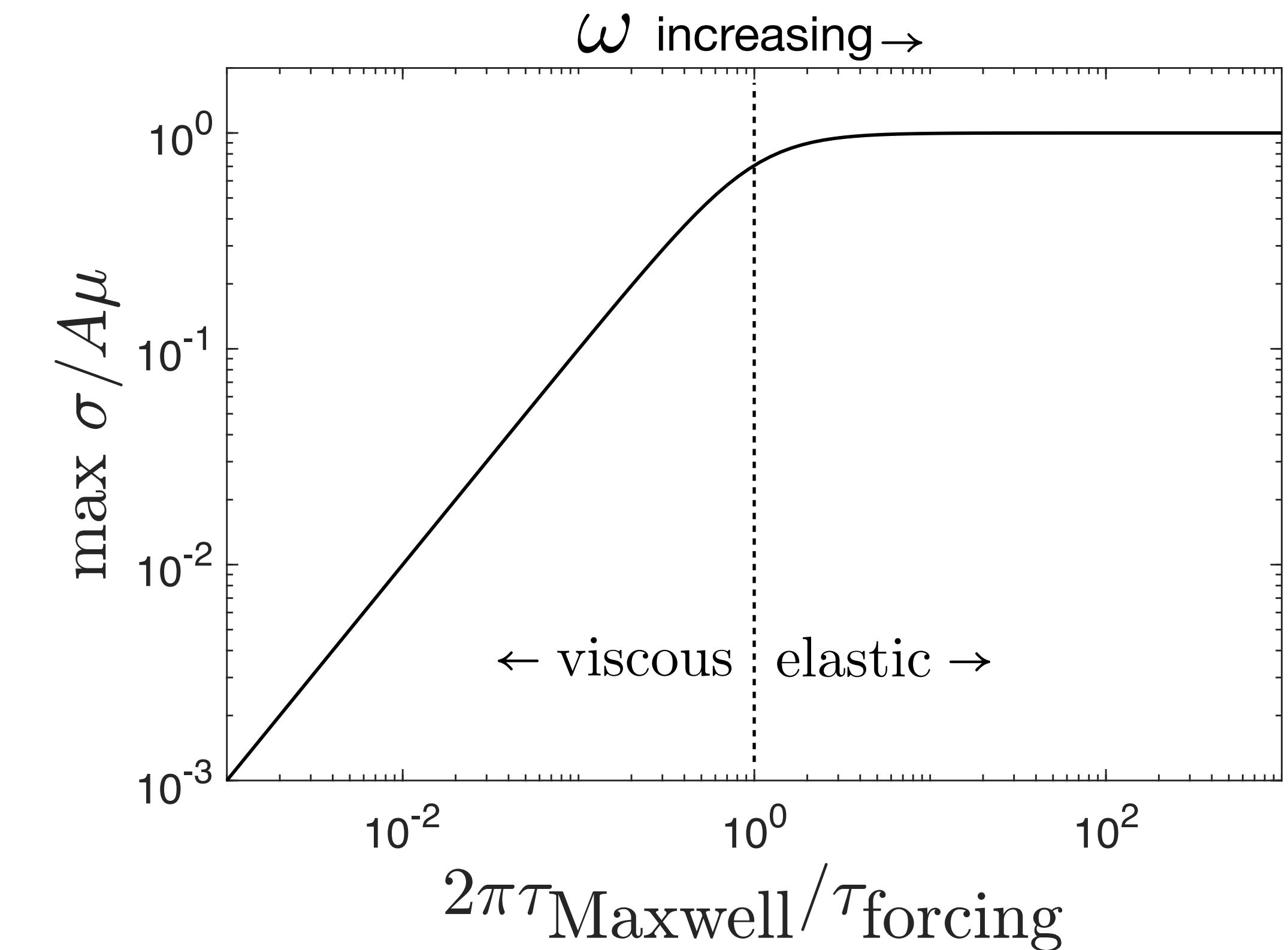
$$\epsilon = \operatorname{Re} (A e^{i\omega t}) = A \cos \omega t$$

$$\sigma = \frac{A\mu}{\sqrt{1 + \operatorname{De}^{-2}}} \sin(\omega t + \Phi)$$

where $\left\{ \begin{array}{l} \Phi \equiv \tan^{-1} \operatorname{De} \\ \operatorname{De} \equiv \frac{\eta/\mu}{1/\omega} \equiv 2\pi \frac{\tau_{\text{Maxwell}}}{\tau_{\text{forcing}}} \end{array} \right.$

$$\tau_{\text{Maxwell}} \ll \tau_{\text{forcing}}, \operatorname{De} \ll 1 \Rightarrow \sigma \sim A\omega\eta \sin \omega t$$

$$\tau_{\text{Maxwell}} \gg \tau_{\text{forcing}}, \operatorname{De} \gg 1 \Rightarrow \sigma \sim A\mu \cos \omega t$$



The Maxwell time & Deborah number of rocks

$$\eta = \eta_0 \exp \left[\frac{E^*}{R} \left(\frac{1}{T} - \frac{1}{T_0} \right) \right] \gtrsim 10^{18} \text{ Pa-s}$$

$$\tau_{\text{Maxwell}} \gtrsim 10^8 \text{ s}$$

$$\mu \sim \text{const.} \sim 10^{10} \text{ Pa}$$

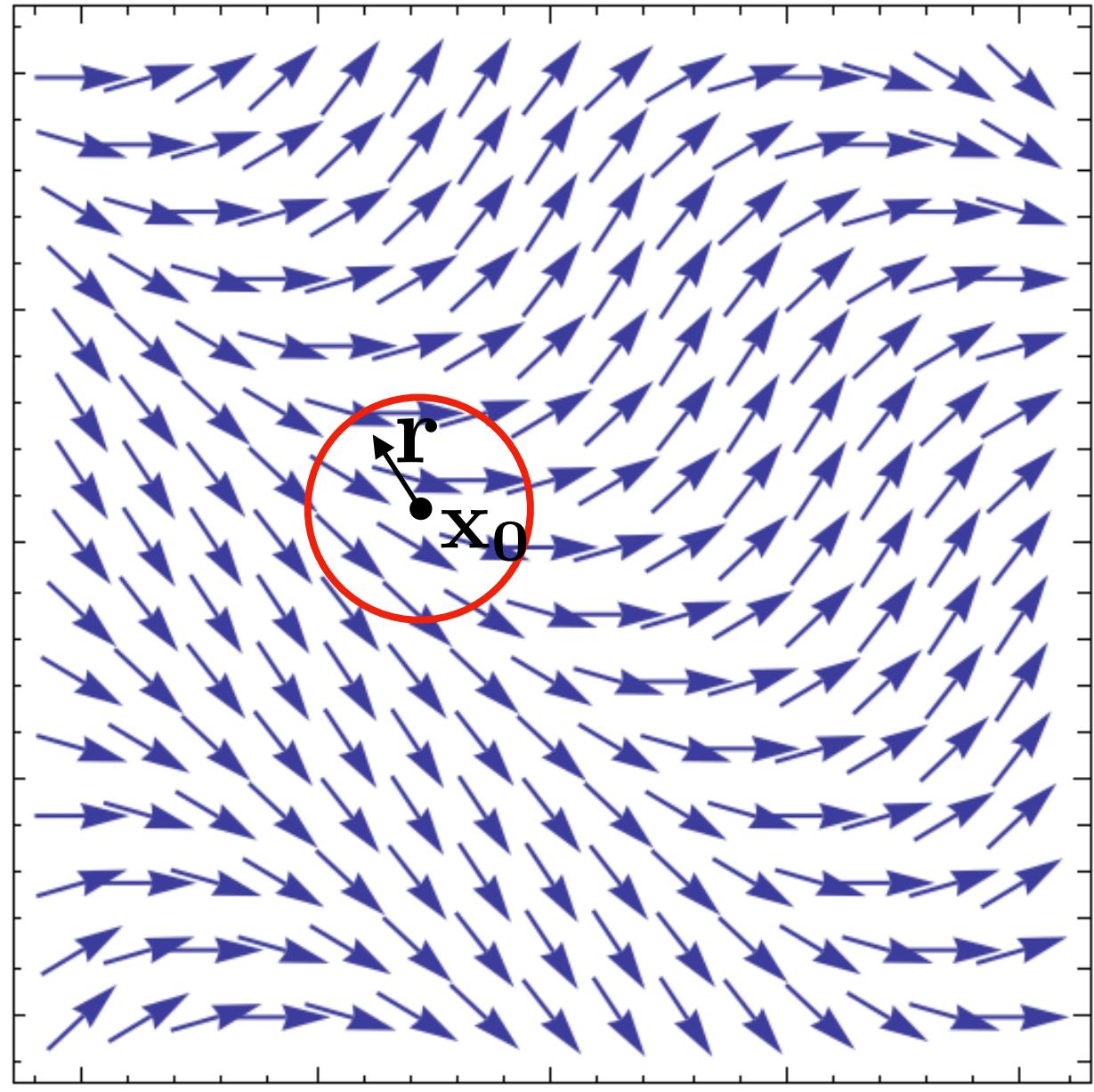
$$\text{De} \sim \frac{\tau_{\text{Maxwell}}}{\tau_{\text{forcing}}}$$

“The mountains flowed before the Lord” (Deborah in Judges 5:5)

$\tau_{\text{seismic}} \approx 10^0 \text{ s}$	$\left. \right\} \text{De} \gg 1$	elastic
$\tau_{\text{tides}} \approx 10^4 \text{ s}$		
$\tau_{\text{deglaciation}} \approx 10^{10} \text{ s}$	$\text{De} > 1$	viscoelastic
$\tau_{\text{mantle convection}} \approx 10^{15} \text{ s}$	$\text{De} \ll 1$	viscous

Rheology & deformation (tensor version)

Kinematics of a continuum: the strain (rate) tensor



$$\mathbf{u}(\mathbf{x}) \approx \mathbf{u}(\mathbf{x}_0) + \mathbf{r} \cdot \mathbf{E} + \mathbf{r} \cdot \boldsymbol{\Omega}$$

translation (rate) strain (rate) rotation (rate)

Taylor expanding the displacement (or velocity) about a point:

$$\mathbf{u}(\mathbf{x}) = \mathbf{u}(\mathbf{x}_0) + (\mathbf{x} - \mathbf{x}_0) \cdot \nabla \mathbf{u}|_{\mathbf{x}_0} + O(|\mathbf{x} - \mathbf{x}_0|^2)$$

$$\approx \mathbf{u}(\mathbf{x}_0) + \mathbf{r} \cdot \nabla \mathbf{u}$$

$$\text{where } \mathbf{r} = \mathbf{x} - \mathbf{x}_0$$

$$\nabla \mathbf{u} = \frac{\partial u_i}{\partial x_j} \equiv D_{ij} = \mathbf{D}$$

deformation tensor

$$\mathbf{D} = \mathbf{E} + \boldsymbol{\Omega}$$

tensor decomposition

$$\left\{ \begin{array}{l} E_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \\ \Omega_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i} \right) \end{array} \right. \begin{array}{l} \text{symmetric part} \\ \text{antisymmetric part} \end{array}$$

The strain (rate) tensor: isotropic & deviatoric parts

$$\mathbf{u}(\mathbf{x}) \approx \mathbf{u}(\mathbf{x}_0) + \mathbf{r} \cdot \mathbf{E} + \mathbf{r} \cdot \boldsymbol{\Omega}$$

strain (rate) tensor:

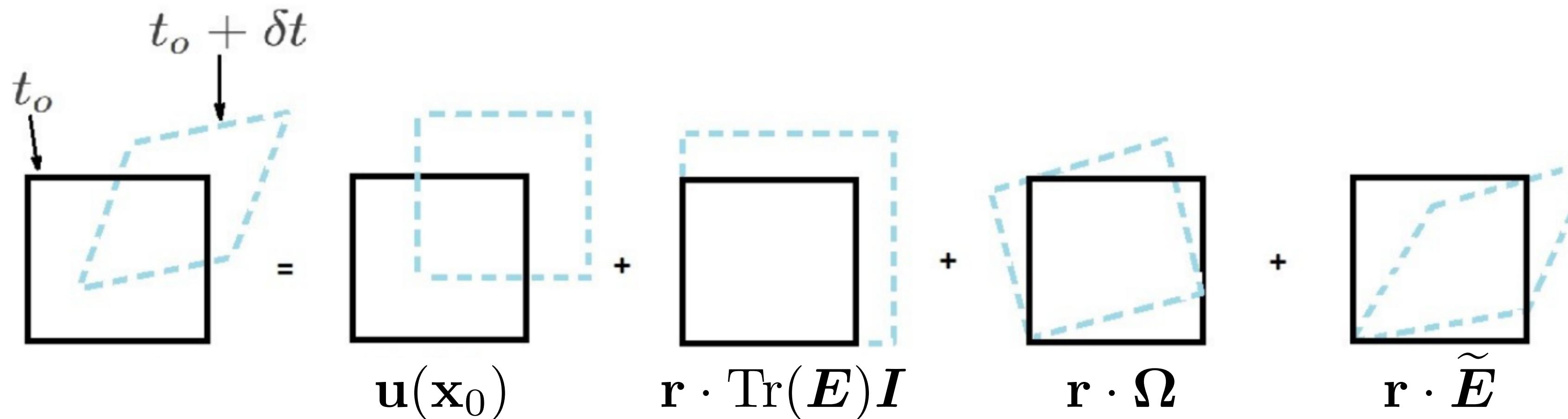
$$\mathbf{E} = \frac{1}{2} \begin{pmatrix} 2\partial_x u_x & (\partial_x u_y + \partial_y u_x) & (\partial_x u_z + \partial_z u_x) \\ (\partial_x u_y + \partial_y u_x) & 2\partial_y u_y & (\partial_z u_y + \partial_y u_z) \\ (\partial_x u_z + \partial_z u_x) & (\partial_z u_y + \partial_y u_z) & 2\partial_z u_z \end{pmatrix}$$

isotropic part:

$$\text{Tr}(\mathbf{E}) = \frac{\partial u_x}{\partial x} + \frac{\partial u_y}{\partial y} + \frac{\partial u_z}{\partial z} = \nabla \cdot \mathbf{u} \quad (\text{a scalar})$$

deviatoric part:

$$\tilde{\mathbf{E}} \equiv \mathbf{E} - \frac{1}{3} \text{Tr}(\mathbf{E}) \mathbf{I}$$



The stress tensor: isotropic & deviatoric parts

stress tensor:

$$\boldsymbol{\sigma} = \begin{pmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{pmatrix}$$

isotropic part: $\sigma \equiv \frac{1}{3} \text{Tr}(\boldsymbol{\sigma}) = \frac{\sigma_{xx} + \sigma_{yy} + \sigma_{zz}}{3}$

deviatoric part: $\boldsymbol{\tau} \equiv \boldsymbol{\sigma} - \sigma \mathbf{I}$

} (stress decomposition)

We choose to take *tensile* stress as positive (caution: some authors make a different choice!)

Rheology of a continuum

stress tensor

$$\boldsymbol{\sigma} = \begin{pmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{pmatrix}$$

“ ∞ ”

strain (rate) tensor

$$\boldsymbol{E} = \frac{1}{2} \begin{pmatrix} 2\partial_x u_x & (\partial_x u_y + \partial_y u_x) & (\partial_x u_z + \partial_z u_x) \\ (\partial_x u_y + \partial_y u_x) & 2\partial_y u_y & (\partial_z u_y + \partial_y u_z) \\ (\partial_x u_z + \partial_z u_x) & (\partial_z u_y + \partial_y u_z) & 2\partial_z u_z \end{pmatrix}$$

Solid (elastic) response: 

$$\boldsymbol{\epsilon} \equiv \boldsymbol{E}$$

key assumption!

$$\begin{cases} \boldsymbol{\sigma} \propto \text{Tr}(\boldsymbol{\epsilon}) \\ \boldsymbol{\tau} \propto \boldsymbol{\epsilon} - \frac{1}{3} \text{Tr}(\boldsymbol{\epsilon}) \mathbf{I} \end{cases}$$

$$\boldsymbol{\sigma} = 2\mu \left[\boldsymbol{\epsilon} - \frac{1}{3} \text{Tr}(\boldsymbol{\epsilon}) \mathbf{I} \right] + K \text{Tr}(\boldsymbol{\epsilon}) \mathbf{I}$$

shear
modulus

bulk
modulus

Fluid (viscous) response: 

$$\dot{\boldsymbol{\epsilon}} \equiv \boldsymbol{E}$$

key assumption!

$$\begin{cases} \boldsymbol{\sigma} + p \propto \text{Tr}(\dot{\boldsymbol{\epsilon}}) \\ \boldsymbol{\tau} \propto \dot{\boldsymbol{\epsilon}} - \frac{1}{3} \text{Tr}(\dot{\boldsymbol{\epsilon}}) \mathbf{I} \end{cases}$$

$$\boldsymbol{\sigma} = -p \mathbf{I} + 2\eta \left[\dot{\boldsymbol{\epsilon}} - \frac{1}{3} \text{Tr}(\dot{\boldsymbol{\epsilon}}) \mathbf{I} \right] + \zeta \text{Tr}(\dot{\boldsymbol{\epsilon}}) \mathbf{I}$$

thermodynamic
pressure shear
viscosity

bulk
viscosity

viscoelasticity of a continuum

Viscoelastic response



Strain is additive:

$$\dot{\epsilon} = \dot{\epsilon}^{\text{vi}} + \dot{\epsilon}^{\text{el}}$$

Simplifying assumption of incompressibility:

$$\text{Tr}(\dot{\epsilon}^{\text{vi}}) = \text{Tr}(\dot{\epsilon}^{\text{el}}) = 0$$

$$\sigma + \tau_{\text{Maxwell}} \overset{\nabla}{\dot{\sigma}} = 2\eta \dot{\epsilon}$$

where $\overset{\nabla}{\dot{\sigma}}$ is the *upper convected derivative*
that is something like $\frac{D\sigma}{Dt}$

The elastic stress gets transported by the flow but relaxes on the timescale of the Maxwell time

Principle components of stress and strain (rate)

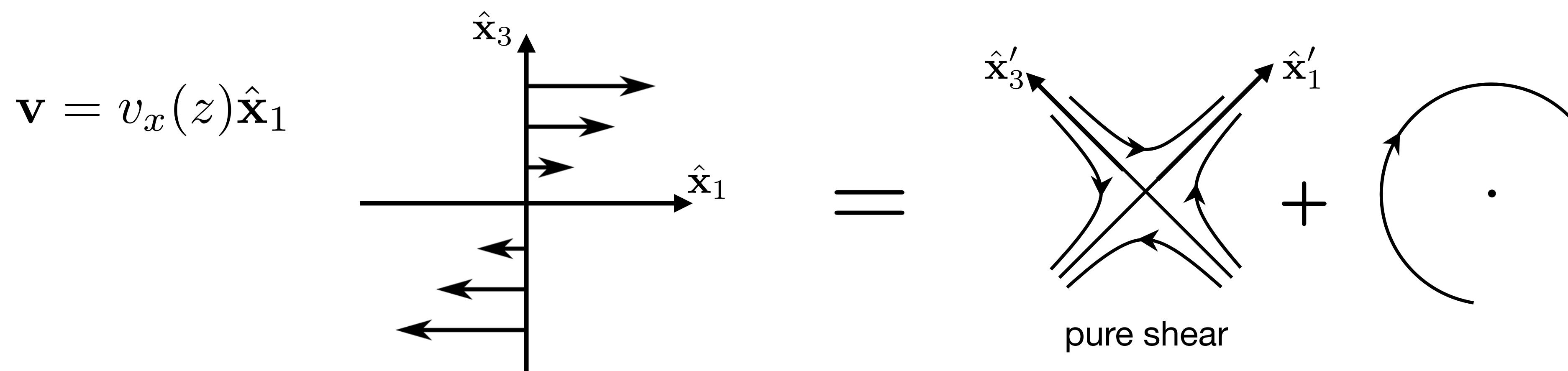
$\sigma, \epsilon, \dot{\epsilon}$

Real, symmetric tensors – can be *diagonalised*

Solve $\det|\sigma - \lambda_j I| = 0$ to determine the eigenvalues λ_j

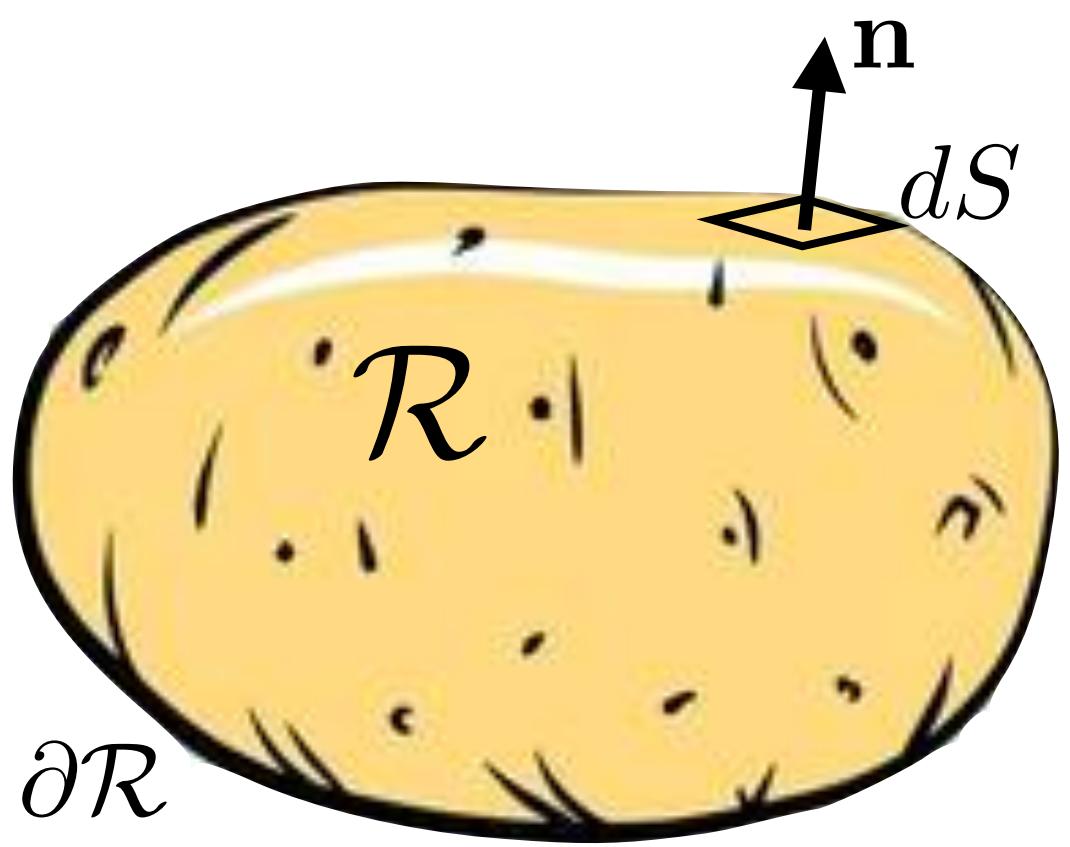
Solve $(\sigma - \lambda_j I)\mathbf{b}_j = 0$ to determine the eigenvectors \mathbf{b}_j

$$\sigma' = \begin{pmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_3 \end{pmatrix} \quad \dot{\epsilon}' = \begin{pmatrix} \epsilon_1 & 0 & 0 \\ 0 & \epsilon_2 & 0 \\ 0 & 0 & \epsilon_3 \end{pmatrix} \quad \hat{\mathbf{x}}'_j = \mathbf{b}_j / |\mathbf{b}_j|$$



Conservation of mass

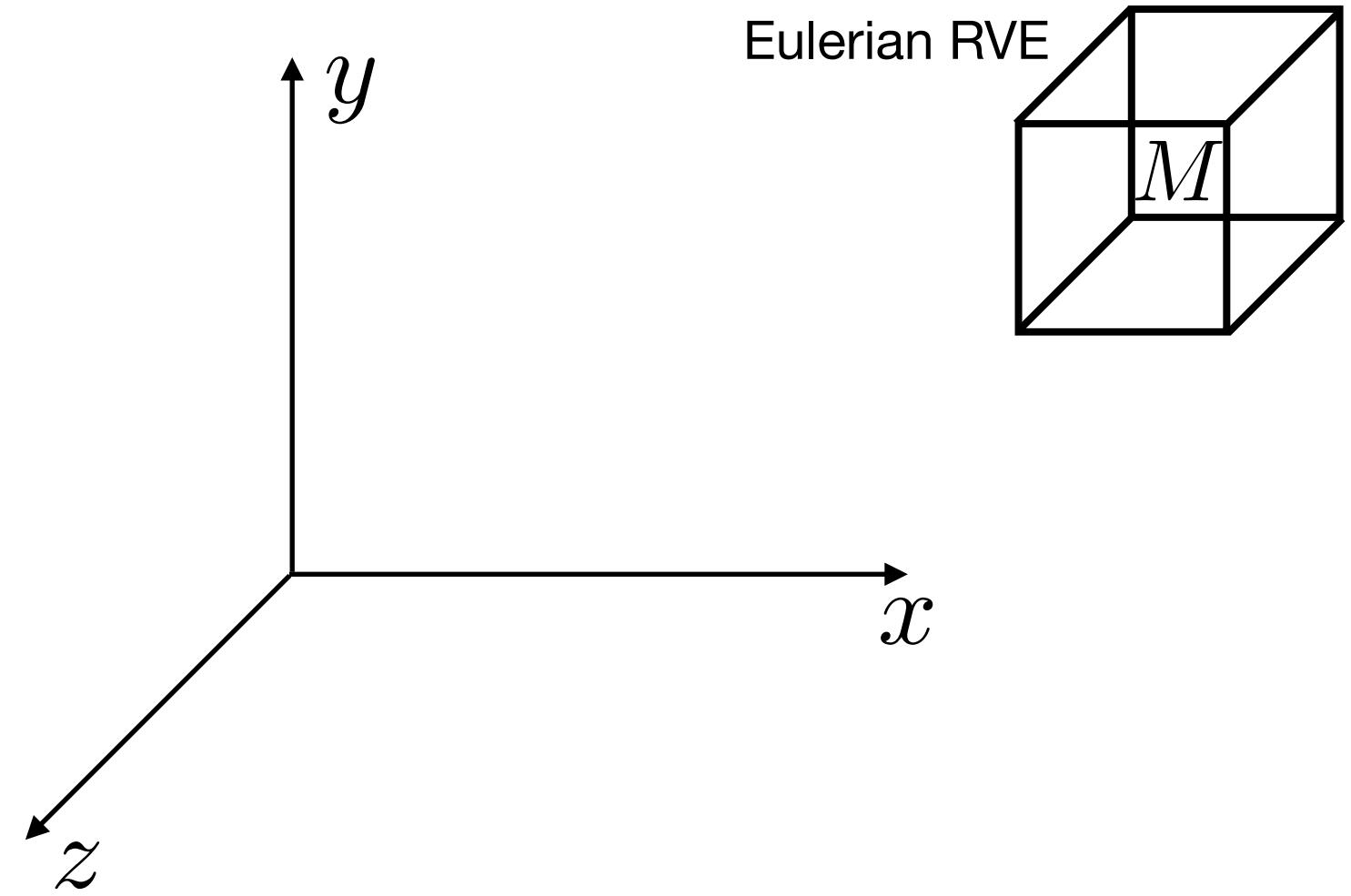
The divergence theorem



$$\int_{\partial\mathcal{R}} \mathbf{J} \cdot \mathbf{n} \, dS = \int_{\mathcal{R}} \nabla \cdot \mathbf{J} \, dV$$

Mass conservation

$$\rho = \rho(\mathbf{x}, t)$$



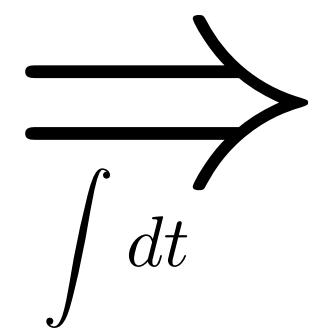
$\frac{dM}{dt} = \text{(net inward flux of mass)}$

$$\begin{aligned}\frac{d}{dt} \int_{\text{RVE}} \rho dV &= - \int_{\partial \text{RVE}} \rho \mathbf{v} \cdot \mathbf{n} dS, \\ &= - \int_{\text{RVE}} \nabla \cdot (\rho \mathbf{v}) dV\end{aligned}$$

$$\int_{\text{RVE}} \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) dV = 0$$

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

The mass conservation equation

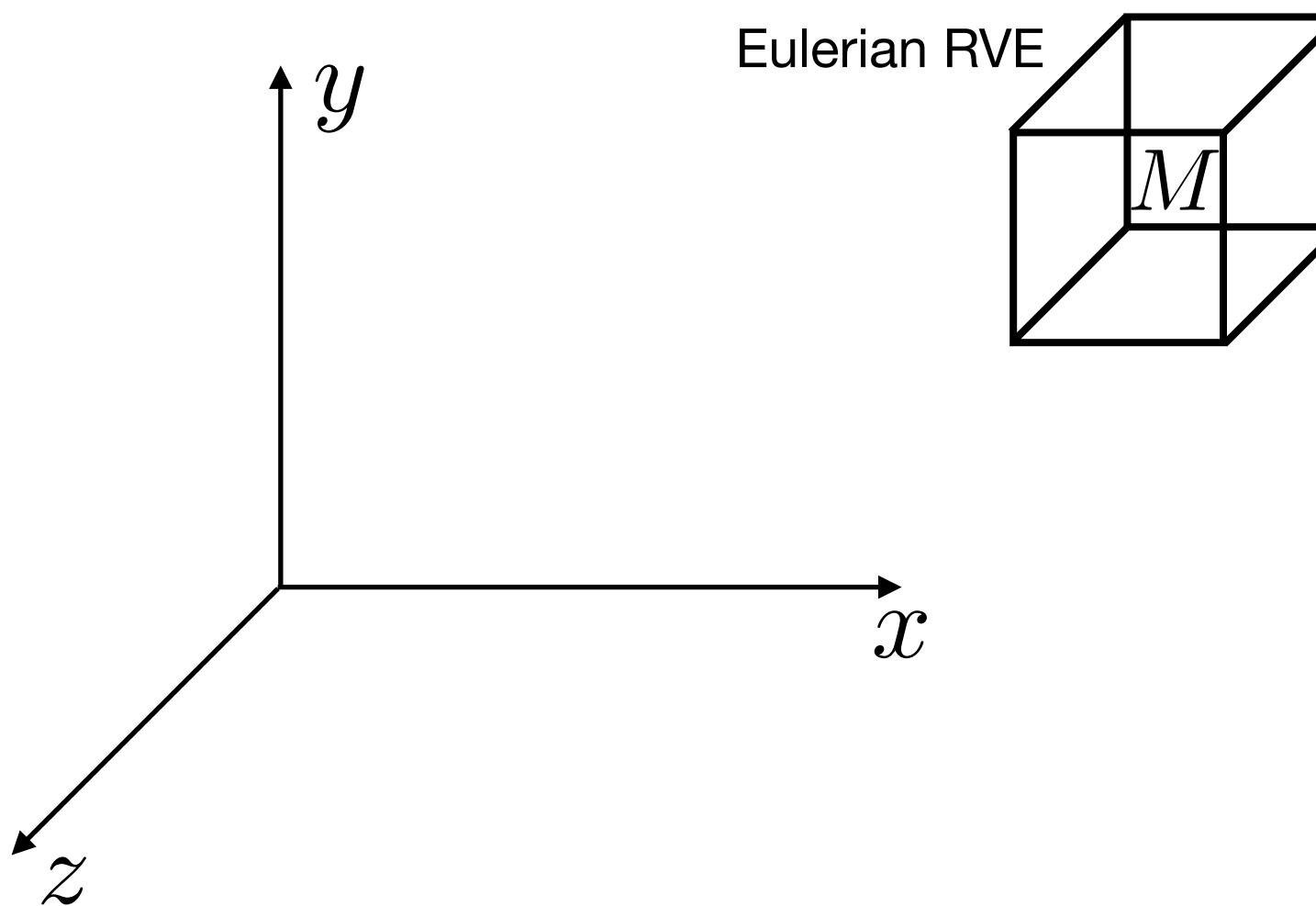


$$\rho \approx \rho_0 (1 - \nabla \cdot \mathbf{U})$$

(for small displacements)

Mass conservation with constant density

$$\rho = \text{const}$$



$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

$$\frac{D\rho}{Dt} + \rho \nabla \cdot \mathbf{v} = 0$$

$$\nabla \cdot \mathbf{v} = 0$$

The continuity equation

$$\Rightarrow \nabla \cdot \mathbf{U} = 0$$

This equation applies for *incompressible flows* of compressible fluids!

Full equations of motion

Elastodynamics and incompressible flow

Recall the Cauchy momentum equation:

$$\nabla \cdot \sigma + \rho g = \rho \frac{D\mathbf{v}}{Dt}$$

Solid (elastic) response: 

$$\sigma = 2\mu \underbrace{\left[\epsilon - \frac{1}{3} \text{Tr}(\epsilon) \mathbf{I} \right]}_{\epsilon} + K \underbrace{\nabla \cdot \mathbf{U}}_{\nabla \cdot \mathbf{U}}$$

We will assume uniformity, $K, \mu = \text{const}$

$$\rho \frac{D\mathbf{v}}{Dt} = K \nabla (\nabla \cdot \mathbf{U}) + \nabla \cdot 2\mu \epsilon + \rho g$$

The Navier equation (linear elastodynamics)

$$\rho \frac{\partial^2 \mathbf{U}}{\partial t^2} = K \nabla (\nabla \cdot \mathbf{U}) + \mu \nabla^2 \mathbf{U} + \rho g$$

Fluid (viscous) response: 

$$\sigma = -p \mathbf{I} + 2\eta \underbrace{\left[\dot{\epsilon} - \frac{1}{3} \text{Tr}(\dot{\epsilon}) \mathbf{I} \right]}_{\dot{\epsilon}} + \zeta \underbrace{\nabla \cdot \mathbf{v}}_{\nabla \cdot \mathbf{v}}$$

We will assume incompressibility, $\nabla \cdot \mathbf{v} = 0$

$$\rho \frac{D\mathbf{v}}{Dt} = -\nabla p + \nabla \cdot 2\eta \dot{\epsilon} + \rho g$$

The Navier-Stokes equation (incompressible flow)

$$\rho \frac{D\mathbf{v}}{Dt} = -\nabla p + \nabla \cdot \eta [\nabla \mathbf{v} + (\nabla \mathbf{v})^T] + \rho g$$

Isoviscous incompressible flow (Navier-Stokes)

$$\rho \frac{D\mathbf{v}}{Dt} = -\nabla p + \nabla \cdot \eta [\nabla \mathbf{v} + (\nabla \mathbf{v})^T] + \rho \mathbf{g}$$
$$\eta = \text{const}$$

Inertia

pressure
gradient

viscous
stress

body
force

$$\rho \left(\frac{\partial \mathbf{v}}{\partial t} + \mathbf{v} \cdot \nabla \mathbf{v} \right) = -\nabla p + \eta \nabla^2 \mathbf{v} + \rho \mathbf{g},$$

$$\nabla \cdot \mathbf{v} = 0$$

Units: N/m³

Viscoelastic incompressible flow

$$\left\{ \begin{array}{l} \rho \frac{D\mathbf{v}}{Dt} = \nabla \cdot \boldsymbol{\sigma} + \rho \mathbf{g} \\ \overset{\nabla}{\dot{\boldsymbol{\sigma}}} = \frac{2\eta \dot{\boldsymbol{\epsilon}} - \boldsymbol{\sigma}}{\tau_{\text{Maxwell}}} \\ \nabla \cdot \mathbf{v} = 0 \end{array} \right.$$

where: $\overset{\nabla}{\dot{\boldsymbol{\sigma}}} \equiv \frac{\partial \boldsymbol{\sigma}}{\partial t} + \mathbf{v} \cdot \nabla \boldsymbol{\sigma} - [\boldsymbol{\sigma} \cdot \nabla \mathbf{v} + (\nabla \mathbf{v})^T \cdot \boldsymbol{\sigma}]$

The upper convected time derivative

Scaling analysis & simplification

Choosing characteristic scales to non-dimensionalise the equation

$$\rho \frac{D\mathbf{v}}{Dt} = -\nabla p + \eta \nabla^2 \mathbf{v} + \rho \mathbf{g}$$

Navier-Stokes equation

$$\mathbf{v} = [v]\mathbf{v}', \quad \mathbf{x} = [x]\mathbf{x}', \quad t = \frac{[x]}{[v]}t', \quad p = [p]p'$$

Define *characteristic scales*
& non-dimensional variables

$$\frac{\rho[v]^2}{[x]} \frac{D\mathbf{v}'}{Dt'} = -\frac{[p]}{[x]} \nabla' p' + \frac{\eta[v]}{[x]^2} \nabla'^2 \mathbf{v}' + \rho \mathbf{g}$$

Substitute into Navier-Stokes

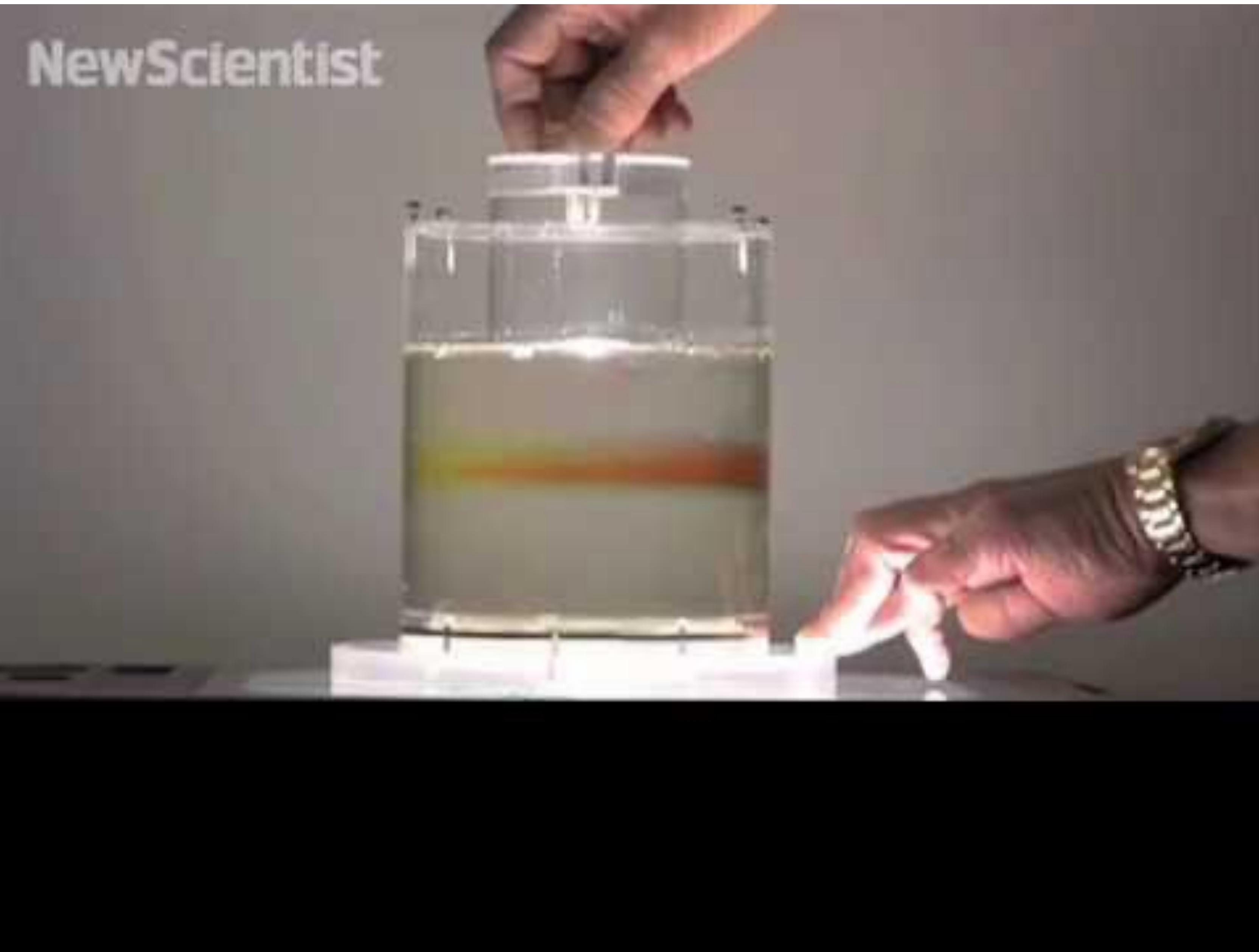
$$\frac{\rho[v][x]}{\eta} \frac{D\mathbf{v}'}{Dt'} = -\frac{[p][x]}{\eta[v]} \nabla' p' + \nabla'^2 \mathbf{v}' + \frac{\rho[x]^2}{\eta[v]} \mathbf{g}$$

Rearrange so that the viscous term
is of order 1

$$\text{Re} \frac{D\mathbf{v}'}{Dt'} = -\nabla' p' + \nabla'^2 \mathbf{v}' + \frac{\rho[x]^2}{\eta[v]} \mathbf{g}$$

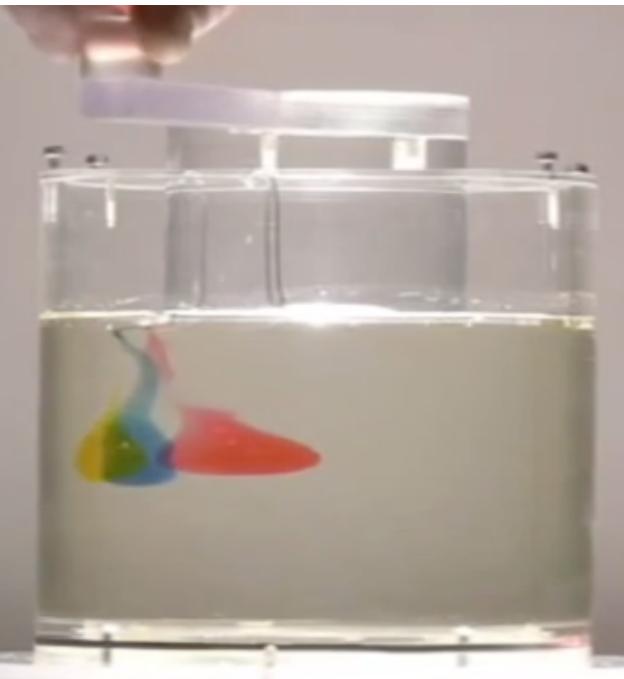
Define $\text{Re} \equiv \frac{\rho[v][x]}{\eta}$, choose $[p] = \frac{\eta[v]}{[x]}$

NewScientist



The Reynolds number

$$\text{Re} \equiv \frac{\rho[v][x]}{\eta} = \frac{\rho[v]^2/[x]}{\eta[v]/[x]^2} = \frac{\text{inertia}}{\text{viscous force}},$$



$$\left. \begin{array}{l} [x] \approx 10 \text{ cm} \\ [v] \approx 10 \text{ cm/sec} \\ \eta \approx 10 \text{ Pa-sec} \\ \rho \approx 1000 \text{ kg/m}^3 \end{array} \right\} \text{Re} = \frac{1000 \cdot 10^{-1} \cdot 10^{-1}}{10} = 1$$

Laminar!

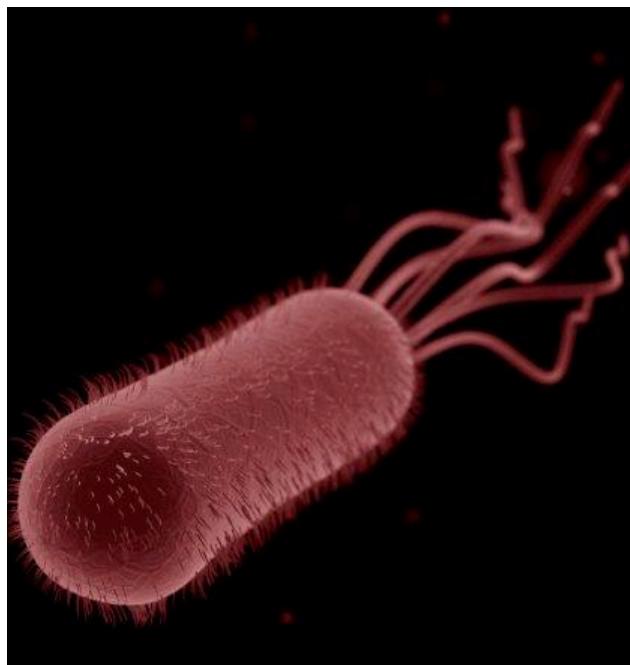


$$\left. \begin{array}{l} [x] \approx 1 \text{ m} \\ [v] \approx 1 \text{ m/sec} \\ \eta \approx 10^{-3} \text{ Pa-sec} \\ \rho \approx 1000 \text{ kg/m}^3 \end{array} \right\} \text{Re} = \frac{1000 \cdot 1 \cdot 1}{10^{-3}} = 10^6$$

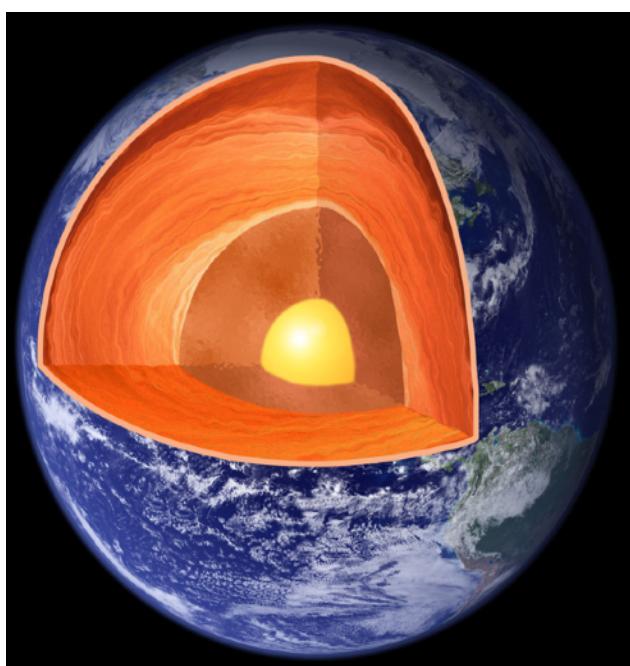
Turbulent!

The Reynolds number

$$\text{Re} \equiv \frac{\rho[v][x]}{\eta} = \frac{\rho[v]^2/[x]}{\eta[v]/[x]^2} = \frac{\text{inertia}}{\text{viscous force}},$$



$$\left. \begin{array}{l} [x] \approx 10^{-6} \text{ m} \\ [v] \approx 10^{-5} \text{ m/s} \\ \eta \approx 10^{-3} \text{ Pa-s} \\ \rho \approx 1000 \text{ kg/m}^3 \end{array} \right\} \text{Re} = \frac{100 \cdot 10^{-6} \cdot 10^{-5}}{10^{-3}} = 10^{-6}$$



$$\left. \begin{array}{l} [x] \approx 3000 \text{ km} \\ [v] \approx 10 \text{ cm/yr} \\ \eta \approx 10^{20} \text{ Pa-sec} \\ \rho \approx 3000 \text{ kg/m}^3 \end{array} \right\} \text{Re} \approx \frac{3000 \cdot (3 \times 10^6) \cdot (3 \times 10^{-7})}{10^{20}} \approx 3 \times 10^{-17}$$

Incompressible, zero-Re flows

$$\rho \frac{D\mathbf{v}}{Dt} = -\nabla p + \eta \nabla^2 \mathbf{v} + \rho \mathbf{g}$$

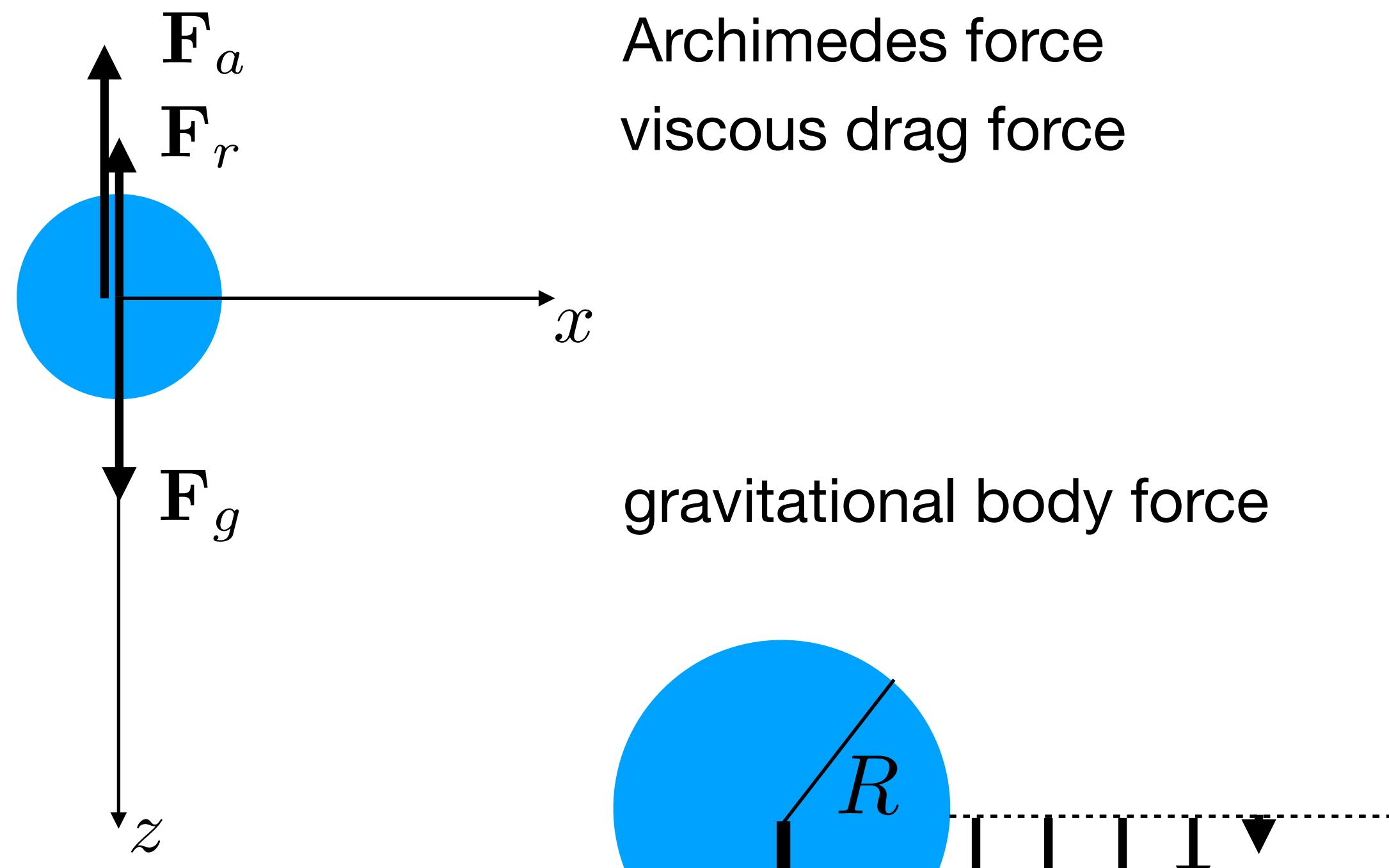
~ 0

$$\nabla p = \nabla \cdot \eta [\nabla \mathbf{v} + (\nabla \mathbf{v})^T] + \rho \mathbf{g},$$

$$\nabla \cdot \mathbf{v} = 0$$

Stokes equations for viscous, incompressible flows

Example: ball falling through fluid



Archimedes force
viscous drag force

gravitational body force

$$\sum \mathbf{F} = M\mathbf{a}$$

$$\Rightarrow \nabla p = \nabla \cdot \eta [\nabla \mathbf{v} + (\nabla \mathbf{v})^T] + \rho \mathbf{g},$$

pressure
force

viscous stress

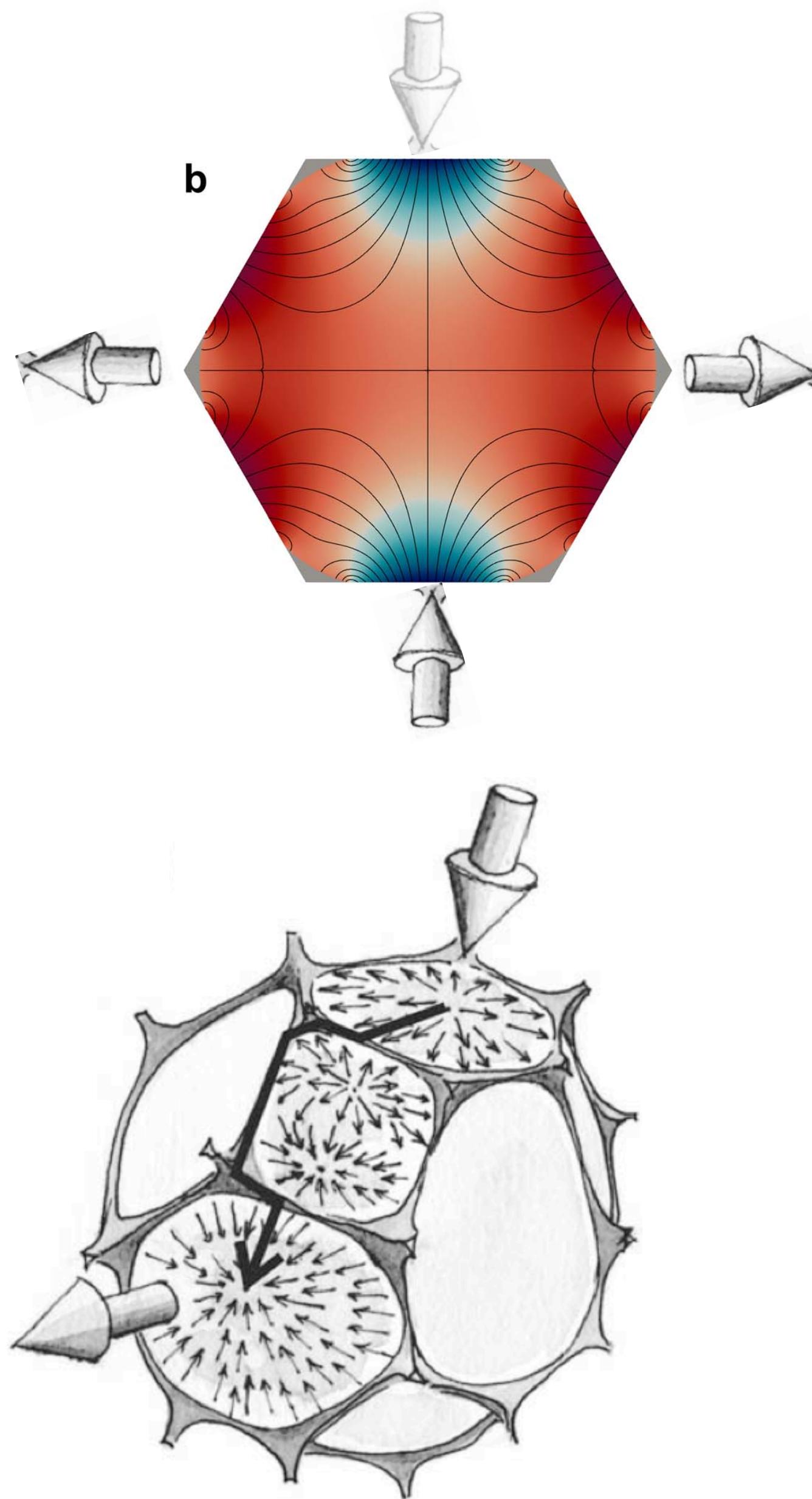
body
force

$$-\rho g \sim \eta v / R^2 - \rho_b g$$

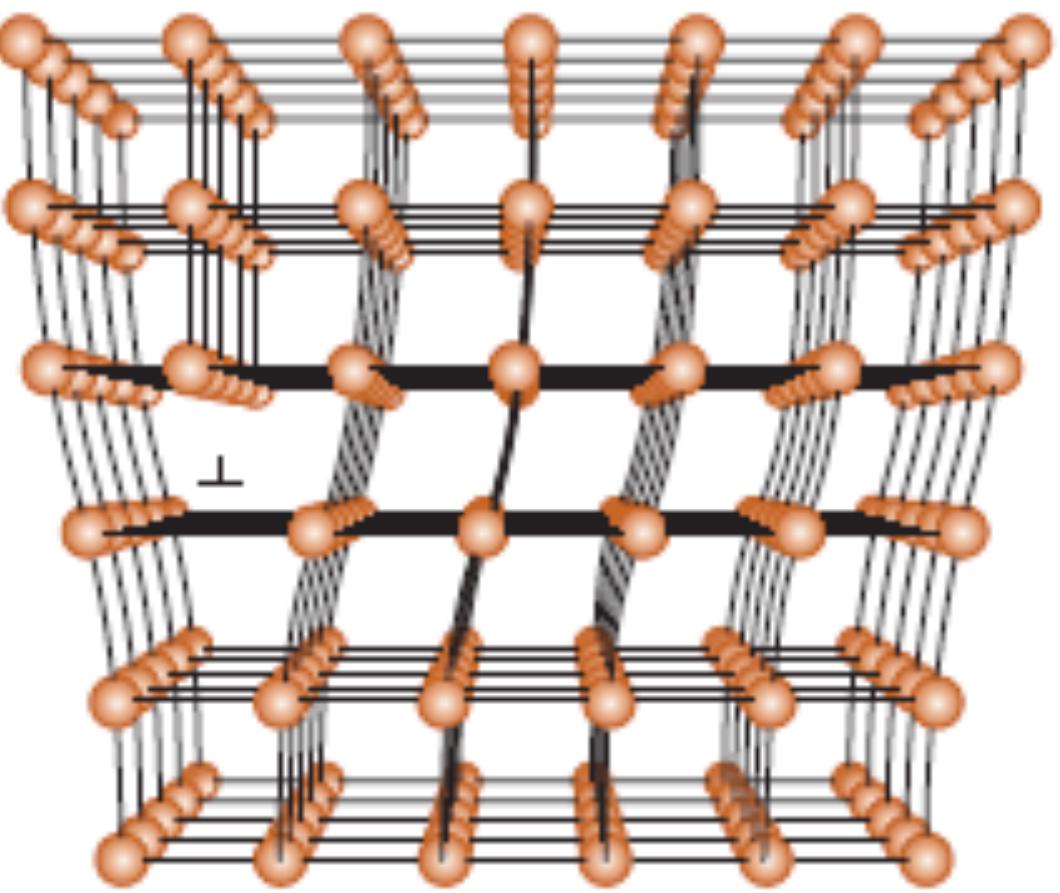
$$v \sim \frac{\Delta \rho g R^2}{\eta}$$

Stokes settling speed

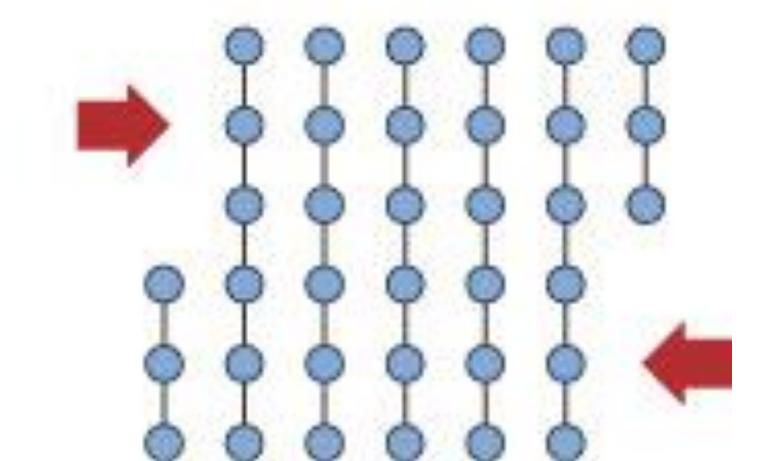
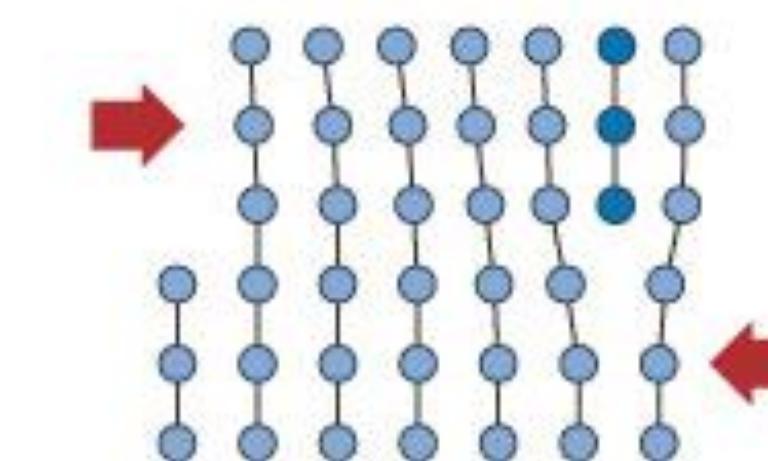
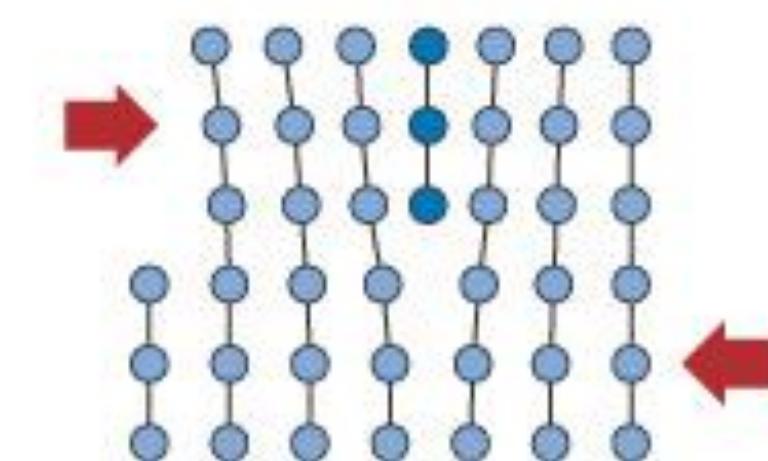
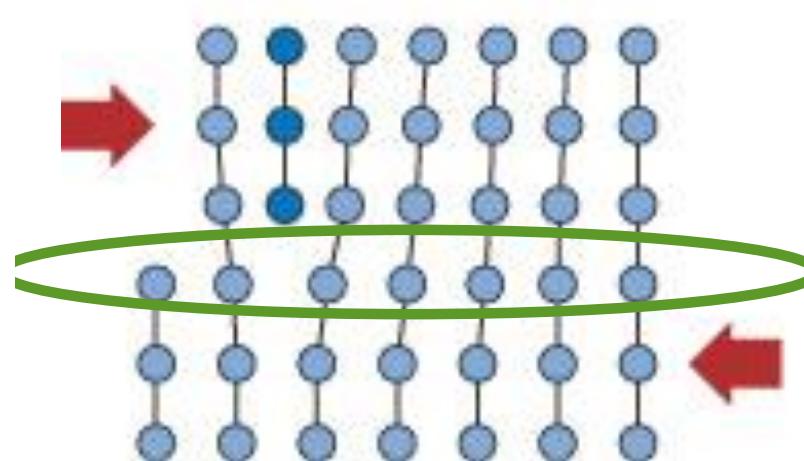
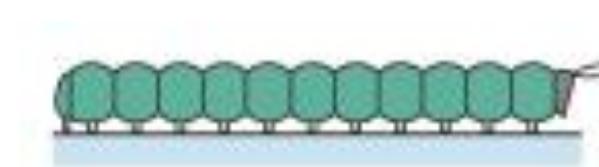
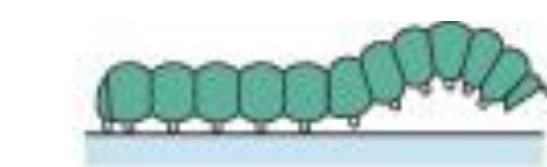
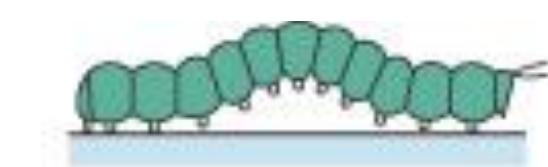
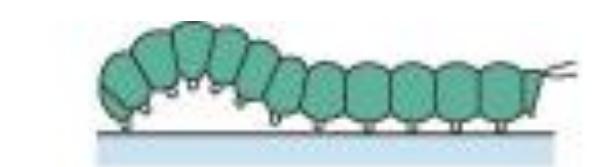
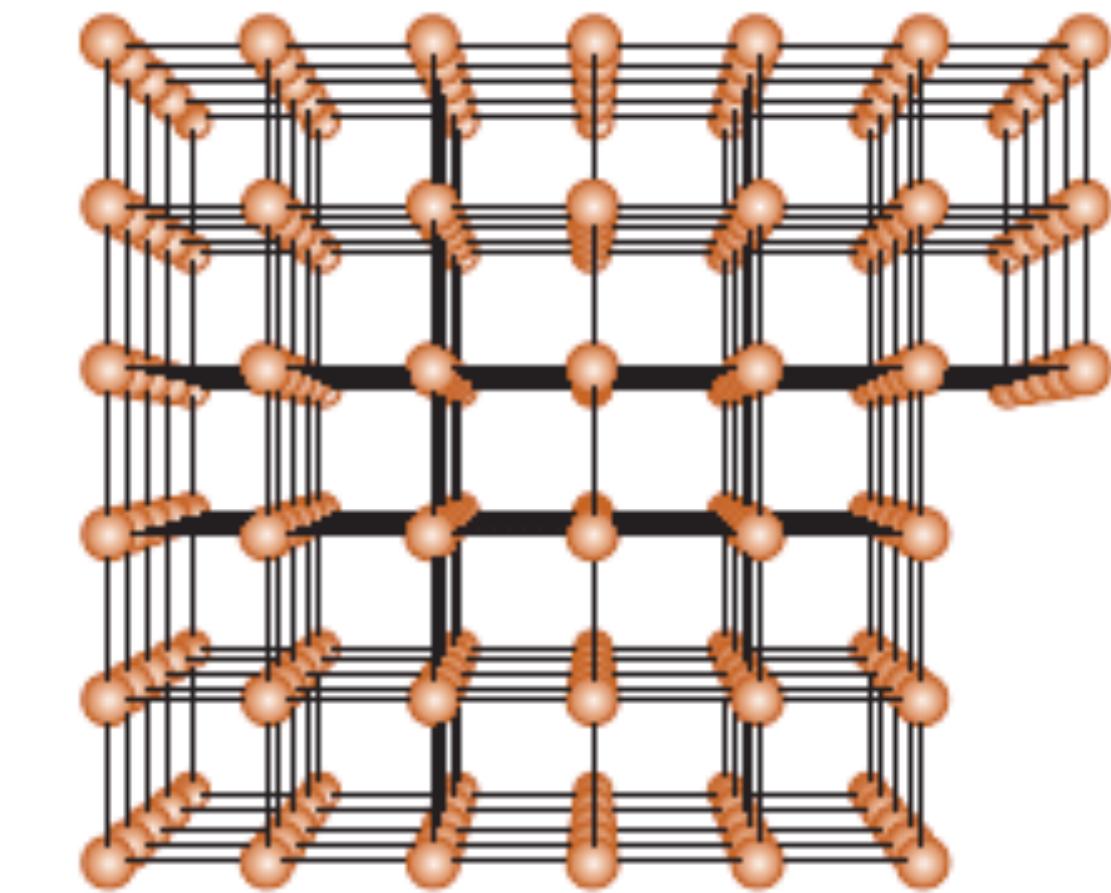
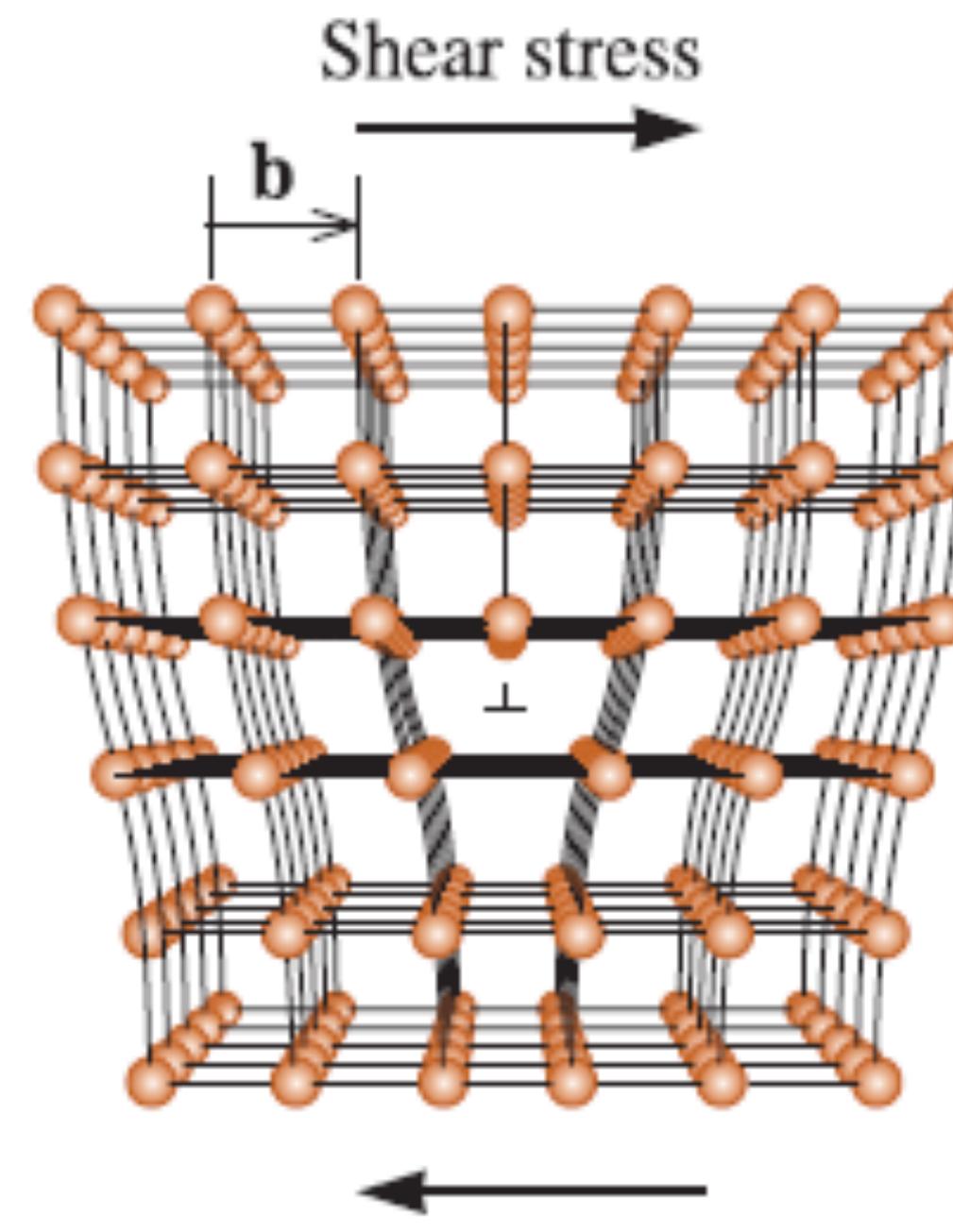
Why do high-temperature rocks flow (creep)?



Diffusion creep

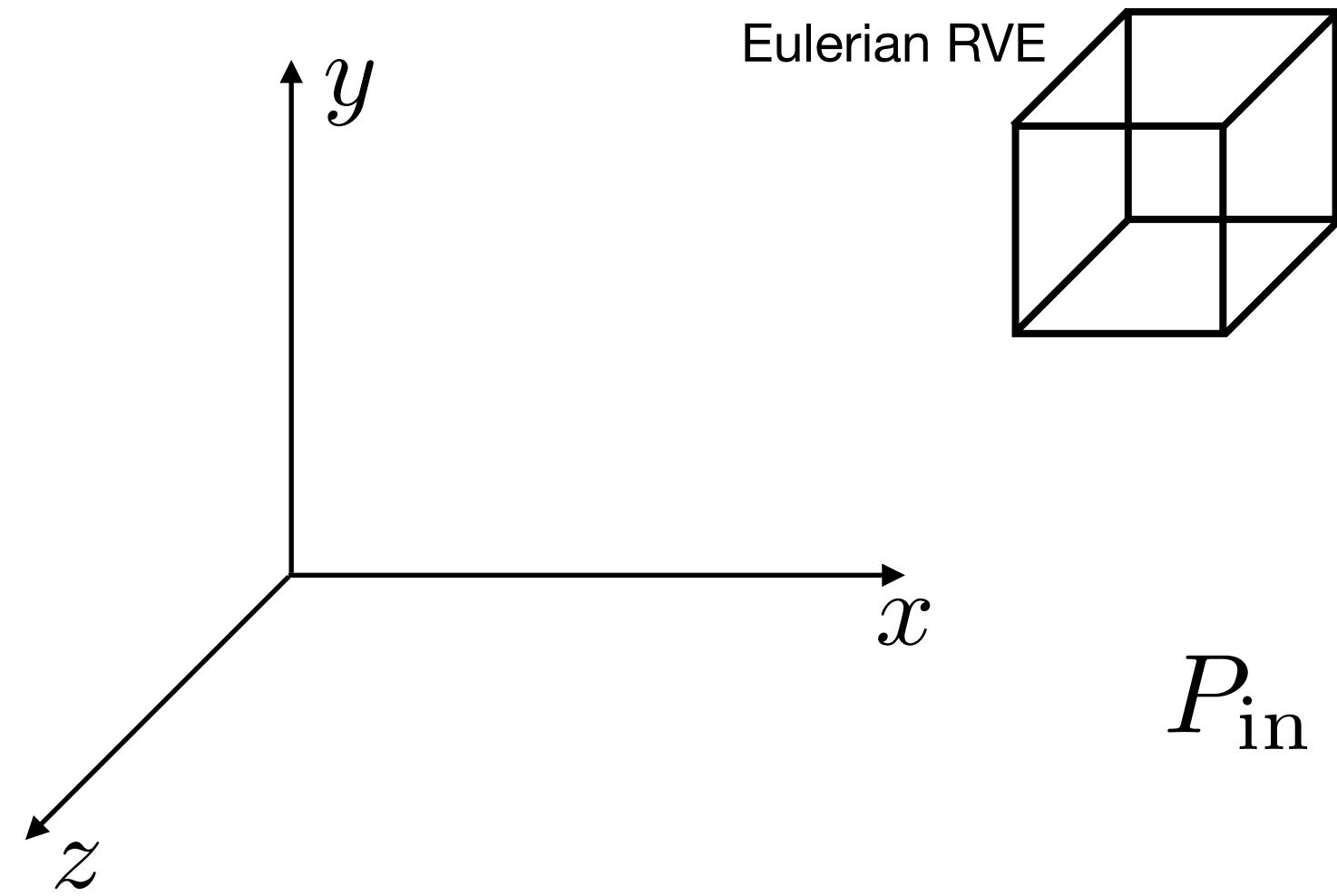


Dislocation creep



Work, Energy & Power

The First law of thermodynamics



$$d\mathcal{E} = (P_{\text{in}} + Q_{\text{in}})dt$$

$$P = \frac{dW}{dt} = \mathbf{F} \cdot \mathbf{v}$$

$$P_{\text{in}} = \int_{\partial \text{RVE}} \mathbf{T} \cdot \mathbf{v} dS + \int_{\text{RVE}} \rho \mathbf{g} \cdot \mathbf{v} dV, \quad \mathbf{T} = \boldsymbol{\sigma} \cdot \mathbf{n}$$

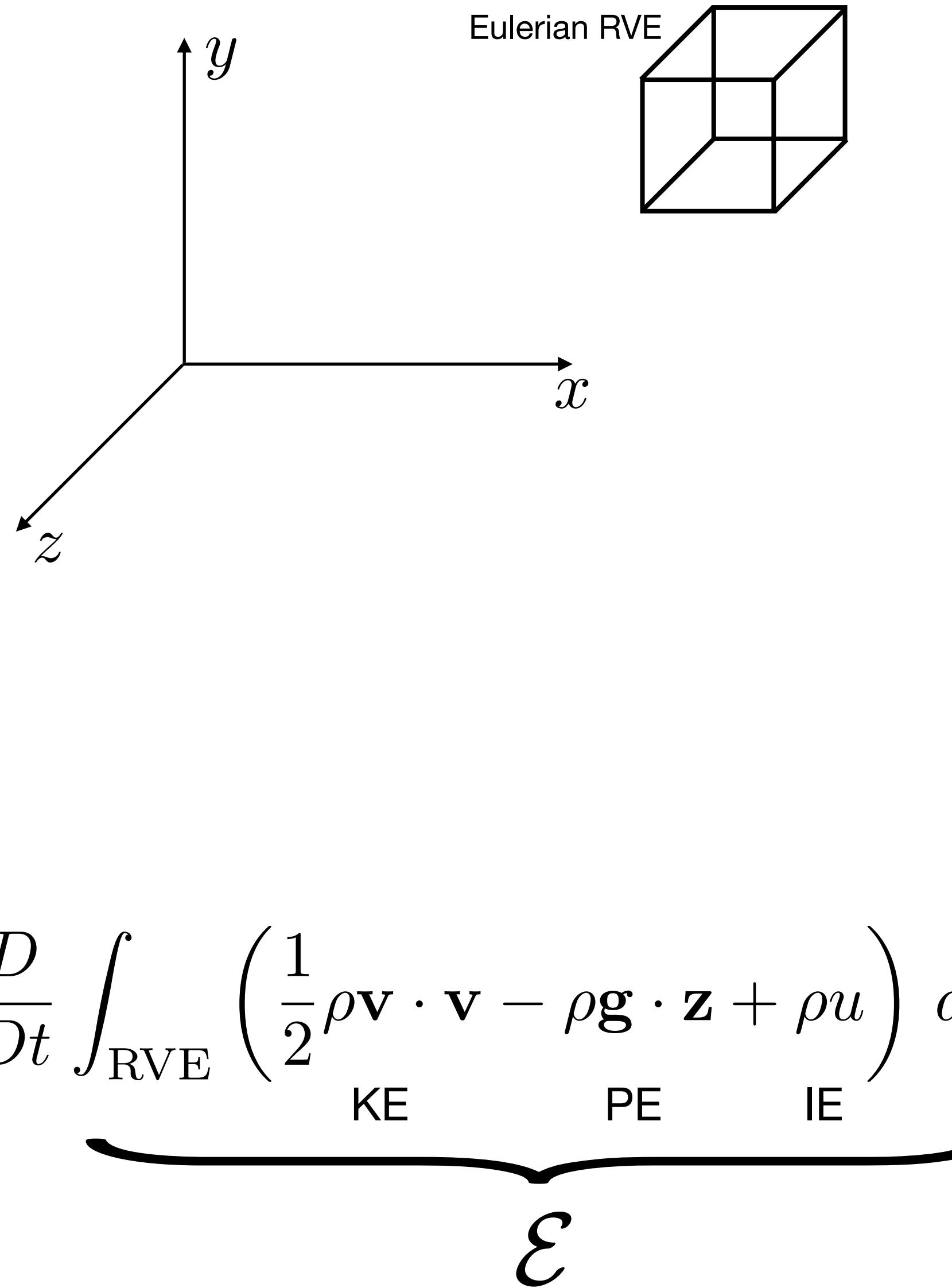
$$= \int_{\text{RVE}} [\mathbf{v} \cdot (\nabla \cdot \boldsymbol{\sigma} + \rho \mathbf{g}) + \boldsymbol{\sigma} : \nabla \mathbf{v}] dV,$$

$$= \int_{\text{RVE}} \left[\rho \mathbf{v} \cdot \frac{D\mathbf{v}}{Dt} + \boldsymbol{\sigma} : \mathbf{D} \right] dV \quad D = E + \boldsymbol{\Omega}$$

$$= \int_{\text{RVE}} \left[\frac{\rho}{2} \frac{D}{Dt} \mathbf{v} \cdot \mathbf{v} + \boldsymbol{\sigma} : \mathbf{E} \right] dV$$

$$= \frac{D}{Dt} \int_{\text{RVE}} \frac{1}{2} \rho \mathbf{v} \cdot \mathbf{v} dV + \int_{\text{RVE}} \boldsymbol{\sigma} : \mathbf{E} dV$$

The First law of thermodynamics



$$d\mathcal{E} = (P_{\text{in}} + Q_{\text{in}})dt$$

$$\begin{aligned} Q_{\text{in}} &= - \int_{\partial \text{RVE}} \mathbf{q} \cdot \mathbf{n} dS + \int_{\text{RVE}} \rho H dV, \\ &= \int_{\text{RVE}} (\rho H - \nabla \cdot \mathbf{q}) dV, \end{aligned}$$

$$\underbrace{\frac{D}{Dt} \int_{\text{RVE}} \left(\frac{1}{2} \rho \mathbf{v} \cdot \mathbf{v} - \rho \mathbf{g} \cdot \mathbf{z} + \rho u \right) dV}_{\mathcal{E}} = \frac{D}{Dt} \int_{\text{RVE}} \frac{1}{2} \rho \mathbf{v} \cdot \mathbf{v} dV + \int_{\text{RVE}} \boldsymbol{\sigma} : \mathbf{E} dV + \int_{\text{RVE}} (\rho H - \nabla \cdot \mathbf{q}) dV$$

KE PE IE

The energy equation and the temperature equation

$$\rho \frac{Du}{Dt} = \rho \mathbf{g} \cdot \mathbf{v} - \nabla \cdot \mathbf{q} + \rho H + \boldsymbol{\sigma} : \mathbf{E}$$

Maxwell relations of thermodynamics

$$du = c_p dT - P d(1/\rho) - \alpha T dP/\rho \quad \text{Fourier's law} \quad \mathbf{q} = -k \nabla T$$

$$\rho c_p \frac{DT}{Dt} = \alpha T \mathbf{g} \cdot \mathbf{v} + k \nabla^2 T + \rho H + \boldsymbol{\sigma} : \mathbf{E}$$

The temperature equation (from conservation of energy)

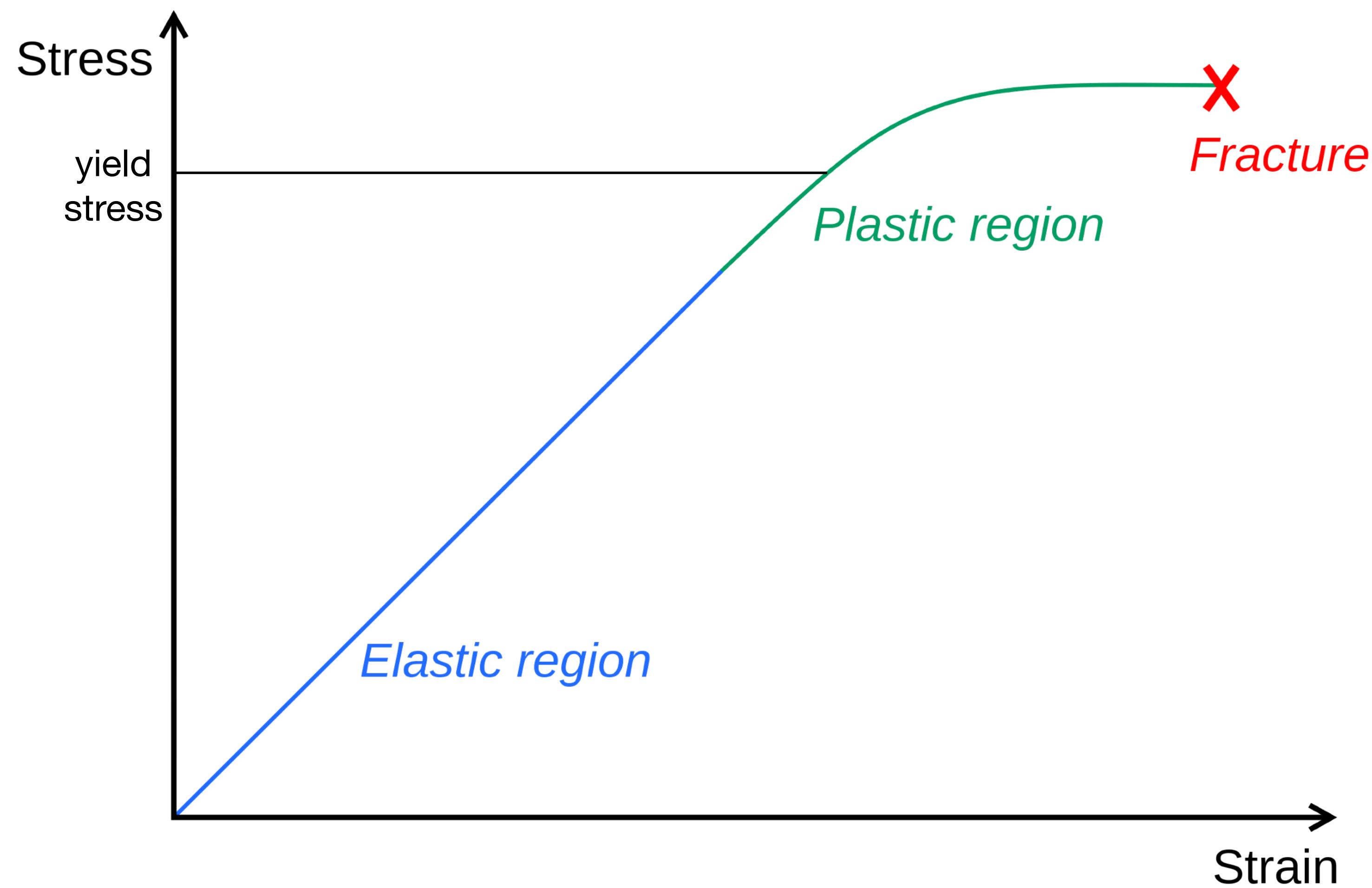
$\boldsymbol{\sigma} : \mathbf{E}$ energy storage (elastic) and energy dissipation (viscous)

Plasticity

Definition

Plasticity is a change in rheology that occurs when the stress exceeds a *yield stress*

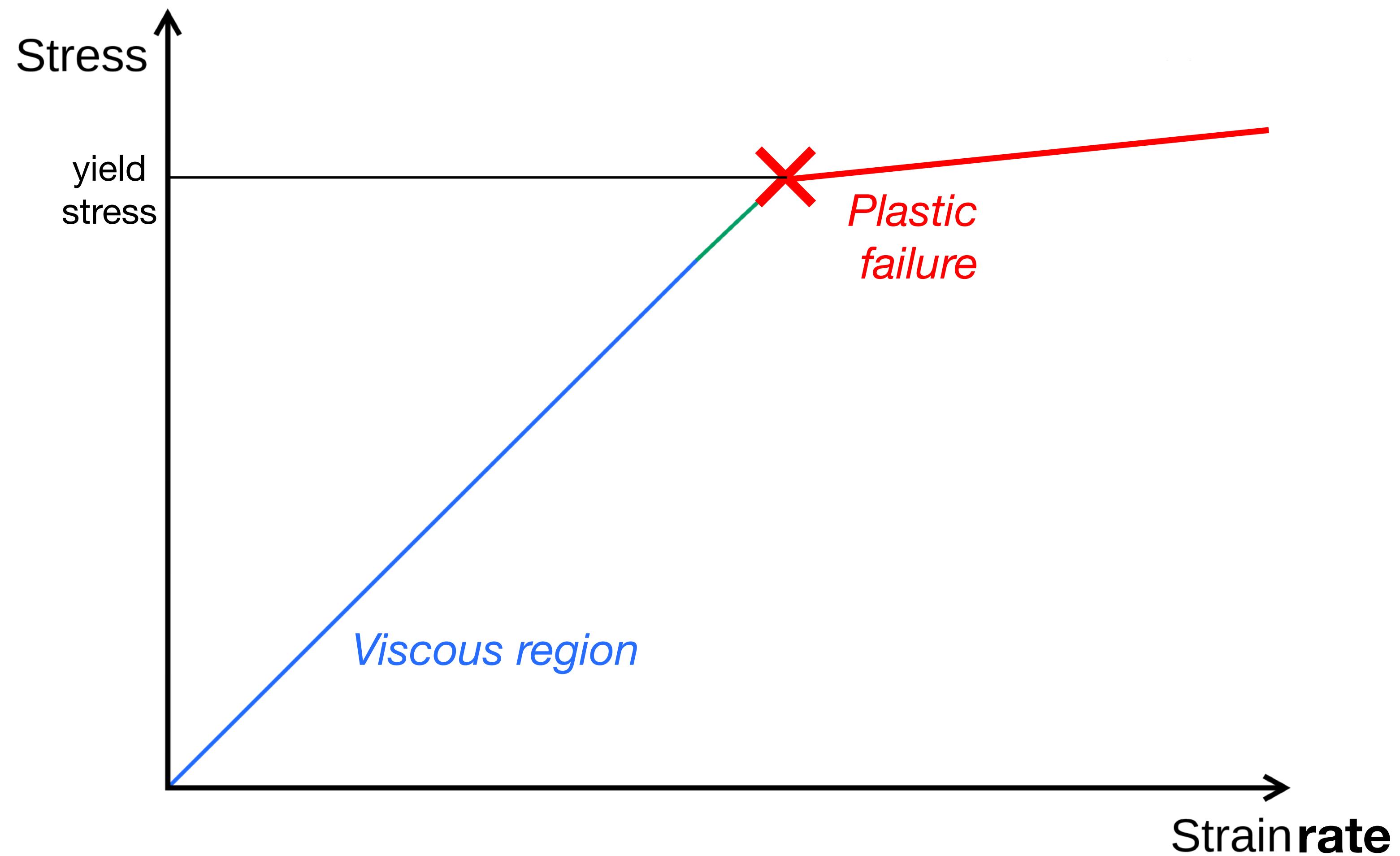
Engineering view:



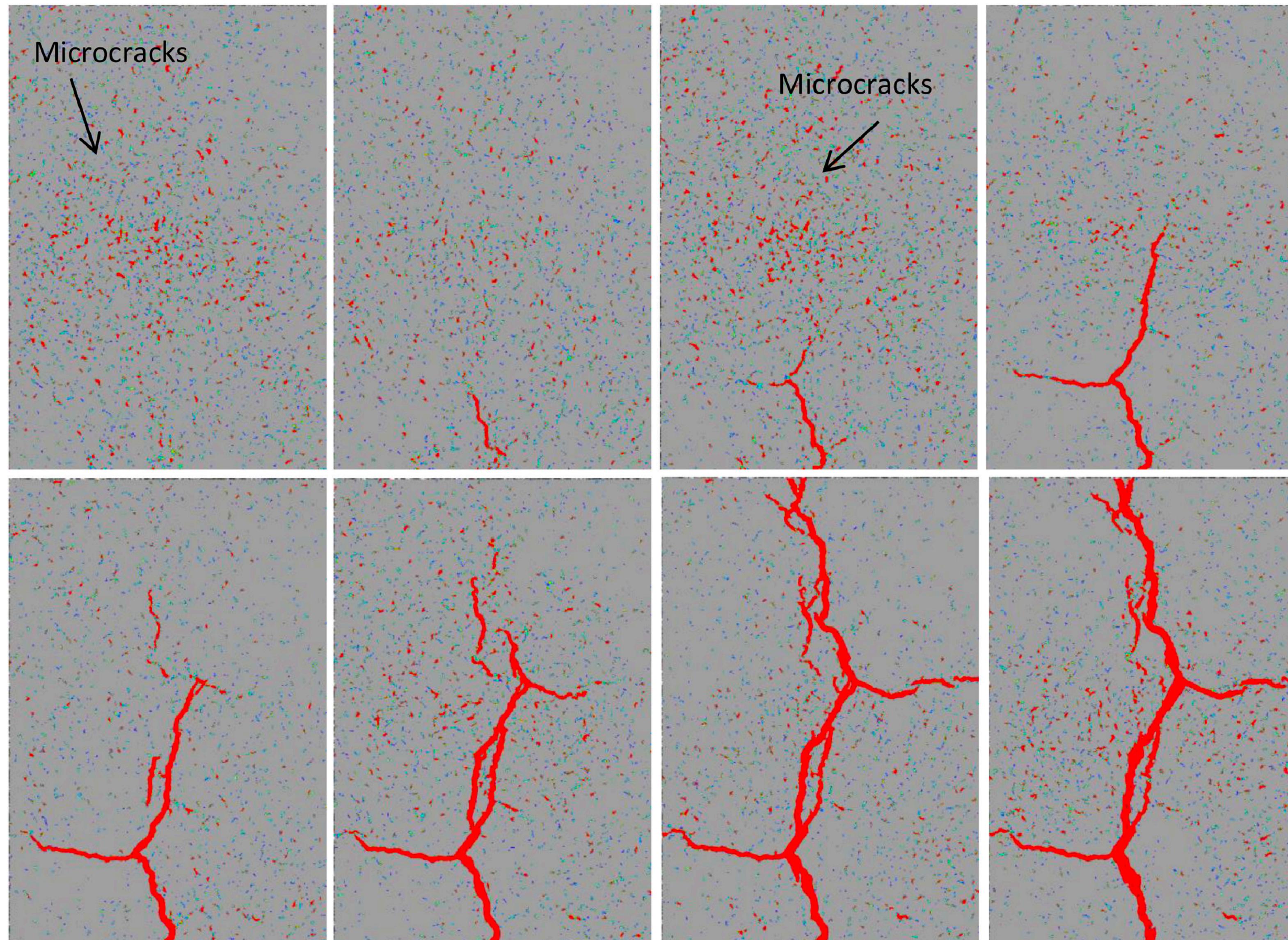
Definition

Plasticity is a change in rheology that occurs when the stress exceeds a *yield stress*

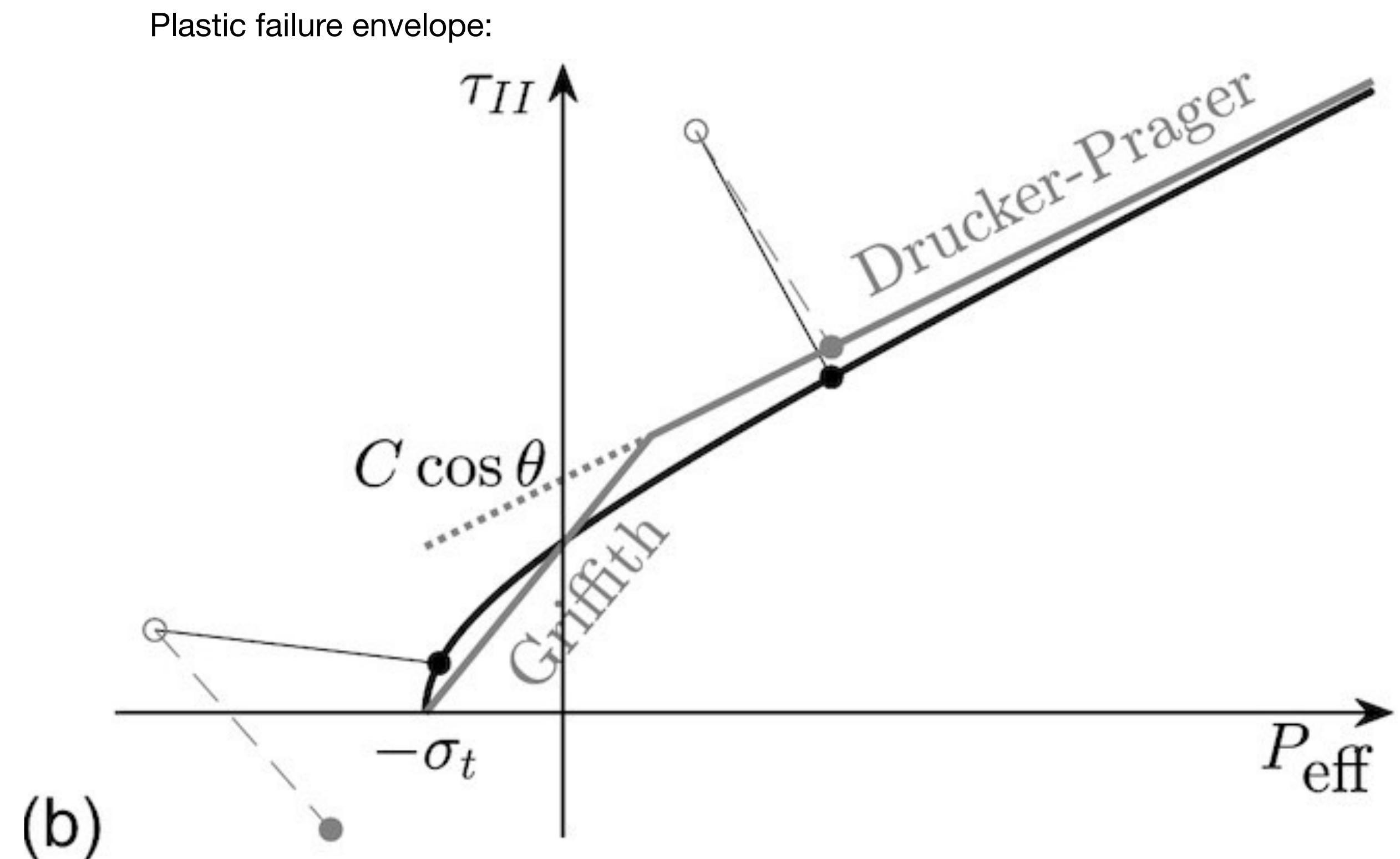
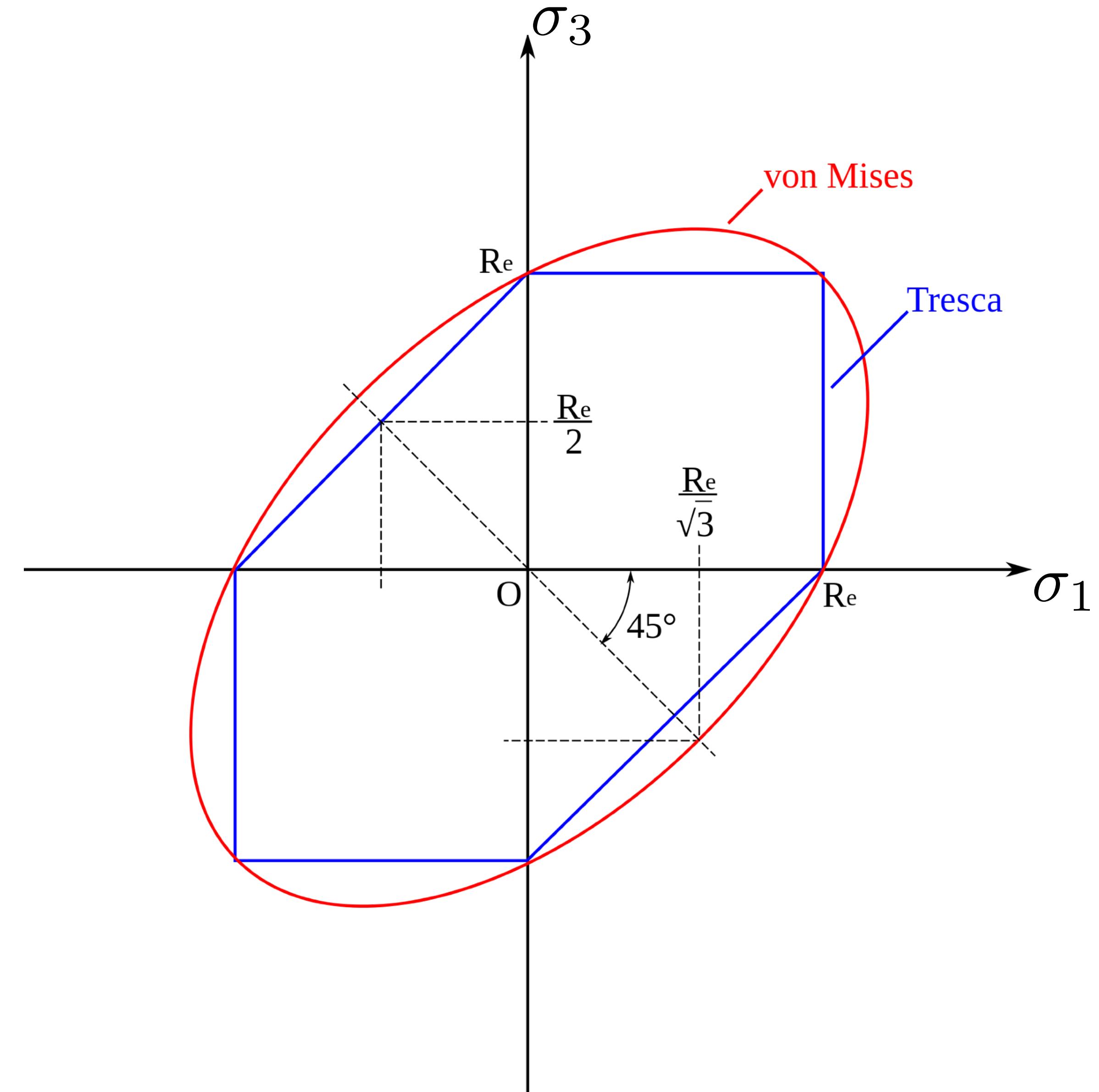
Geodynamics view:



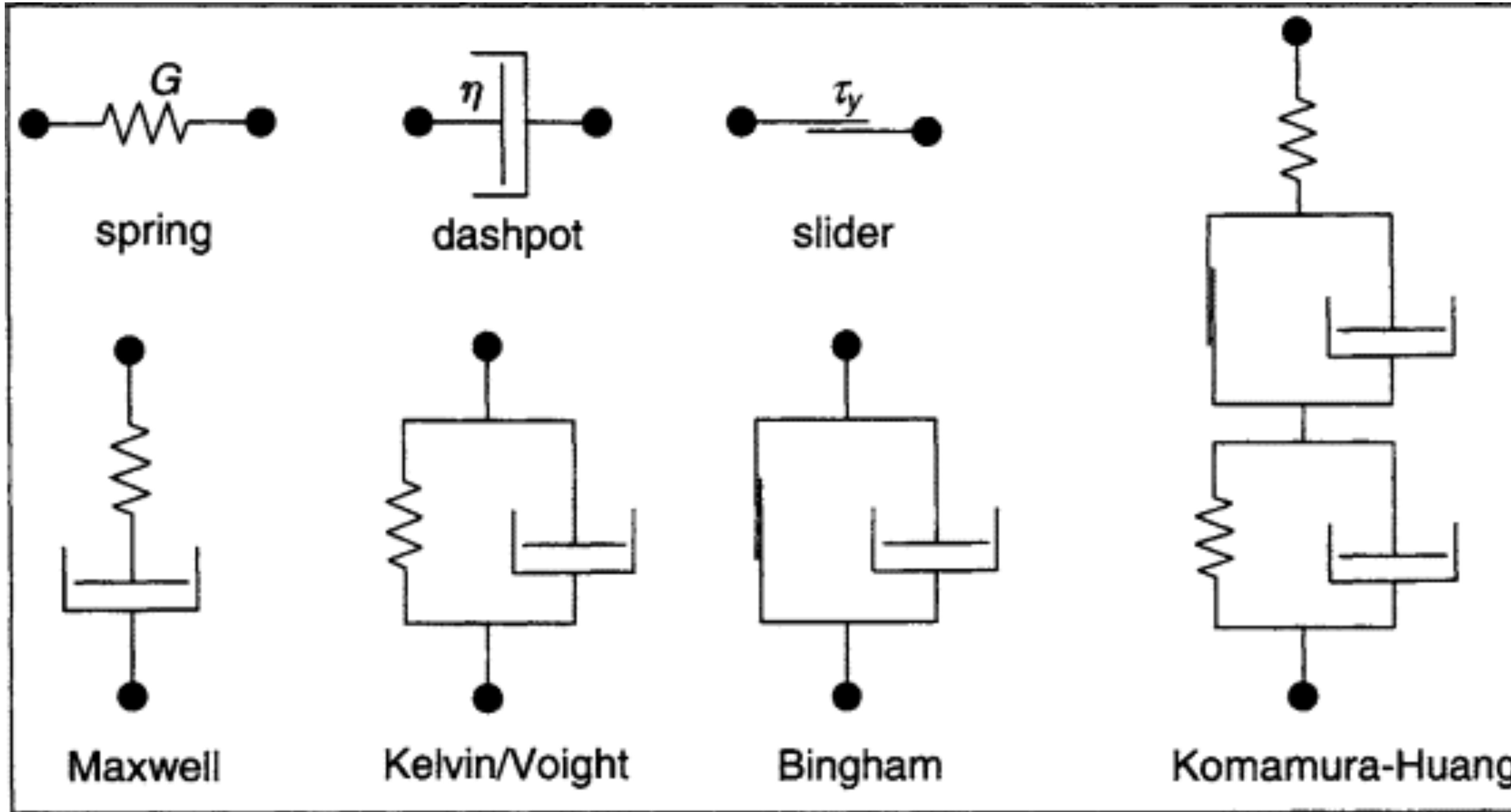
What is plasticity, really?



The yield criterion



Other composite rheologies



Faulting & frictional slip modelled as plastic deformation

