

# Visualizing Counterflow Heat Transport in Superfluid Helium (He II)\*

S. W. Van Sciver

Acknowledgements:

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L. Lourenco

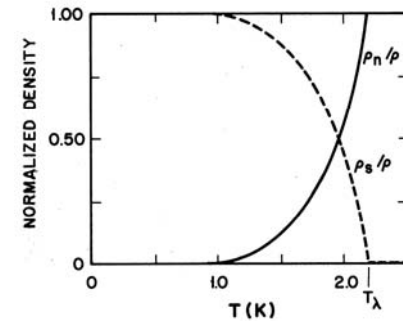
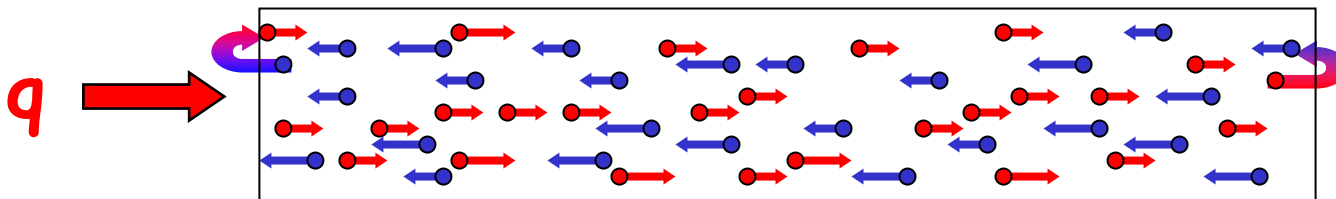
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# Two Fluid Model for He II

- The model (developed by Landau and Tisza) is similar to the two fluid model for superconductivity
- Semi-empirical fluid dynamics model for describing the dynamic behavior of He II.
- He II can be thought to consist of two interpenetrating fluids that are fully miscible and have temperature dependent densities ( $\rho_s$  and  $\rho_n$ )



- These two components (• superfluid and • normal fluid) flow under influence of a heat current that produces pressure and temperature gradients ( $q = \rho_s T \langle v_n \rangle$ )

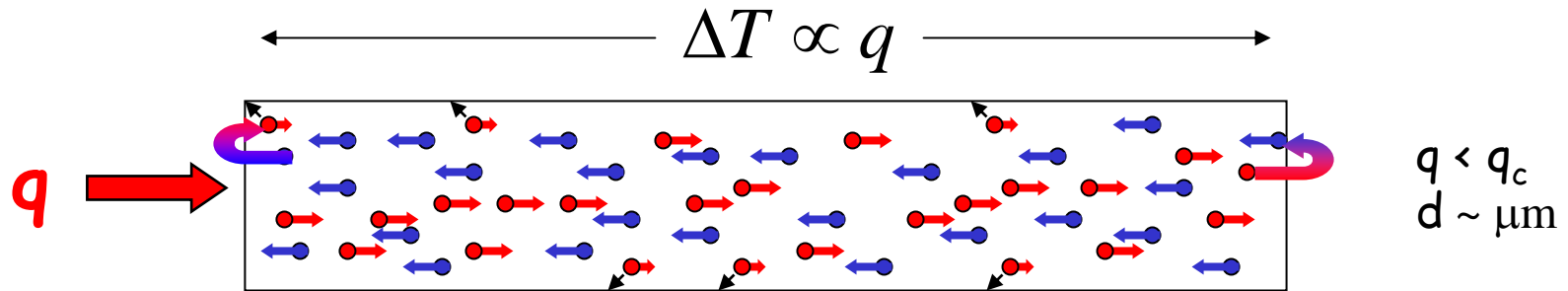
“It should, however, be most decidedly emphasized that regarding the liquid (He II) as a mixture of normal and superfluid parts is no more than a convenient description of the phenomena which occur in a quantum fluid”

L. D. Landau & E. M. Lifshitz  
Fluid Mechanics, p. 515

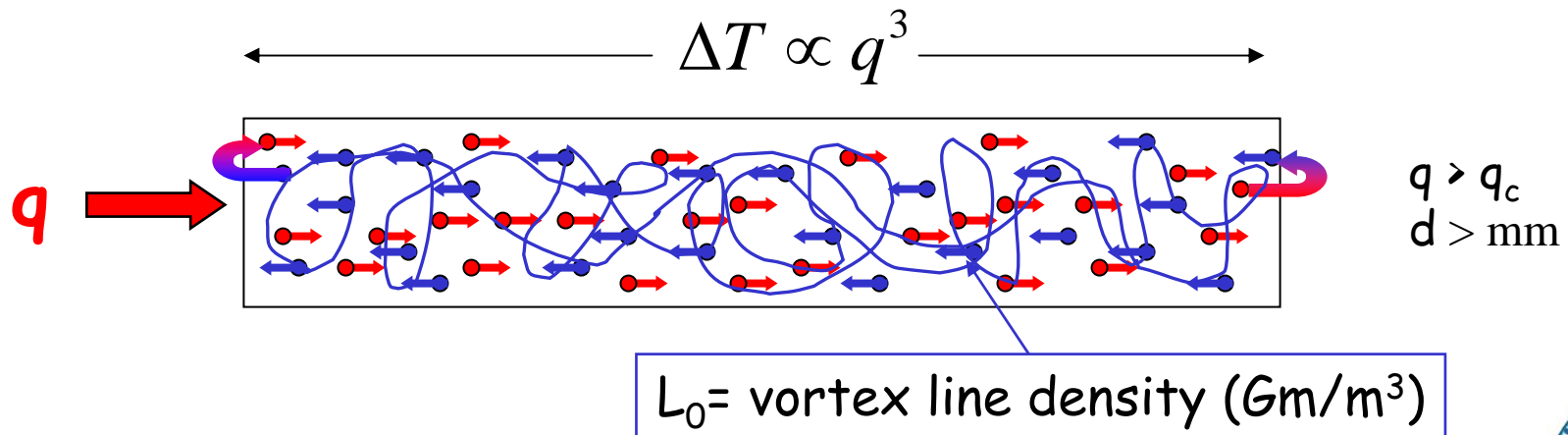


# Counterflow heat transport in He II

- Normal fluid viscous interaction with channel walls (laminar flow)

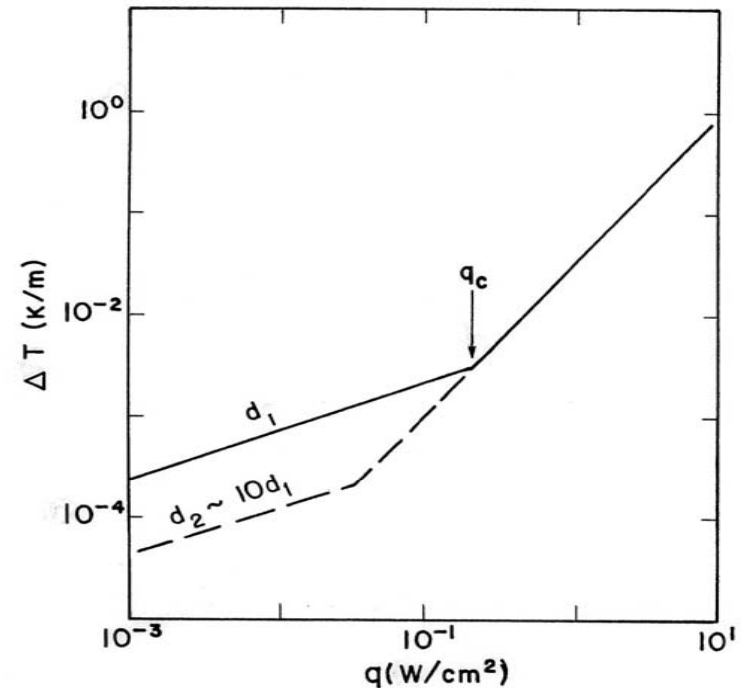
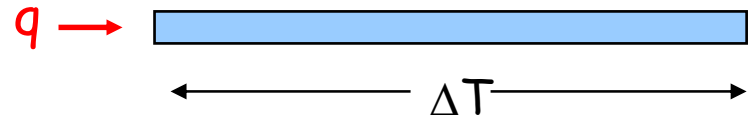


- Mutual friction between normal fluid and turbulent superfluid



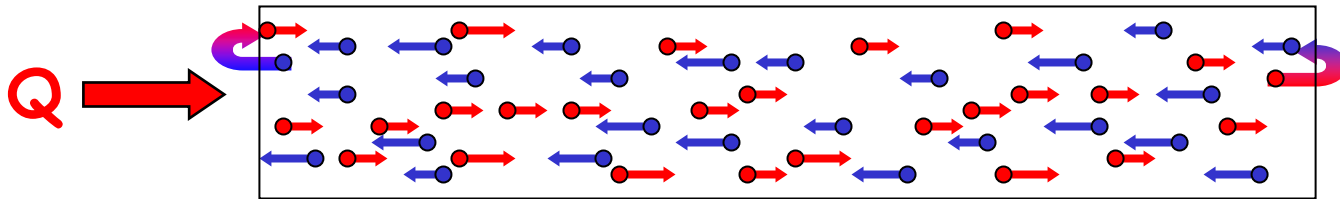
# Steady state heat transport in He II

- Anomalous heat transport
  - Effective heat conductivity comparable to that of high purity metals
  - Low flux, laminar regime  
 $dT/dx \sim q$
  - High flux, turbulent regime  
 $dT/dx \sim q^3$
  - Transition between two regimes marks onset of quantum turbulence and depends on diameter of channel

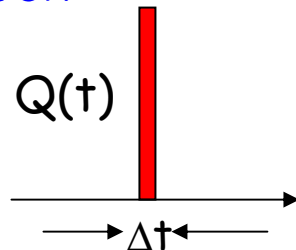


# Transient Heat Transport (2<sup>nd</sup> sound)

- He II can sustain ordinary first sound (pressure waves,  $\rho(p)$ ) and "second sound" (entropy waves,  $\rho_s/\rho_n(T)$ ).
- Second sound can propagate as a standing wave or a pulse that carries entropy



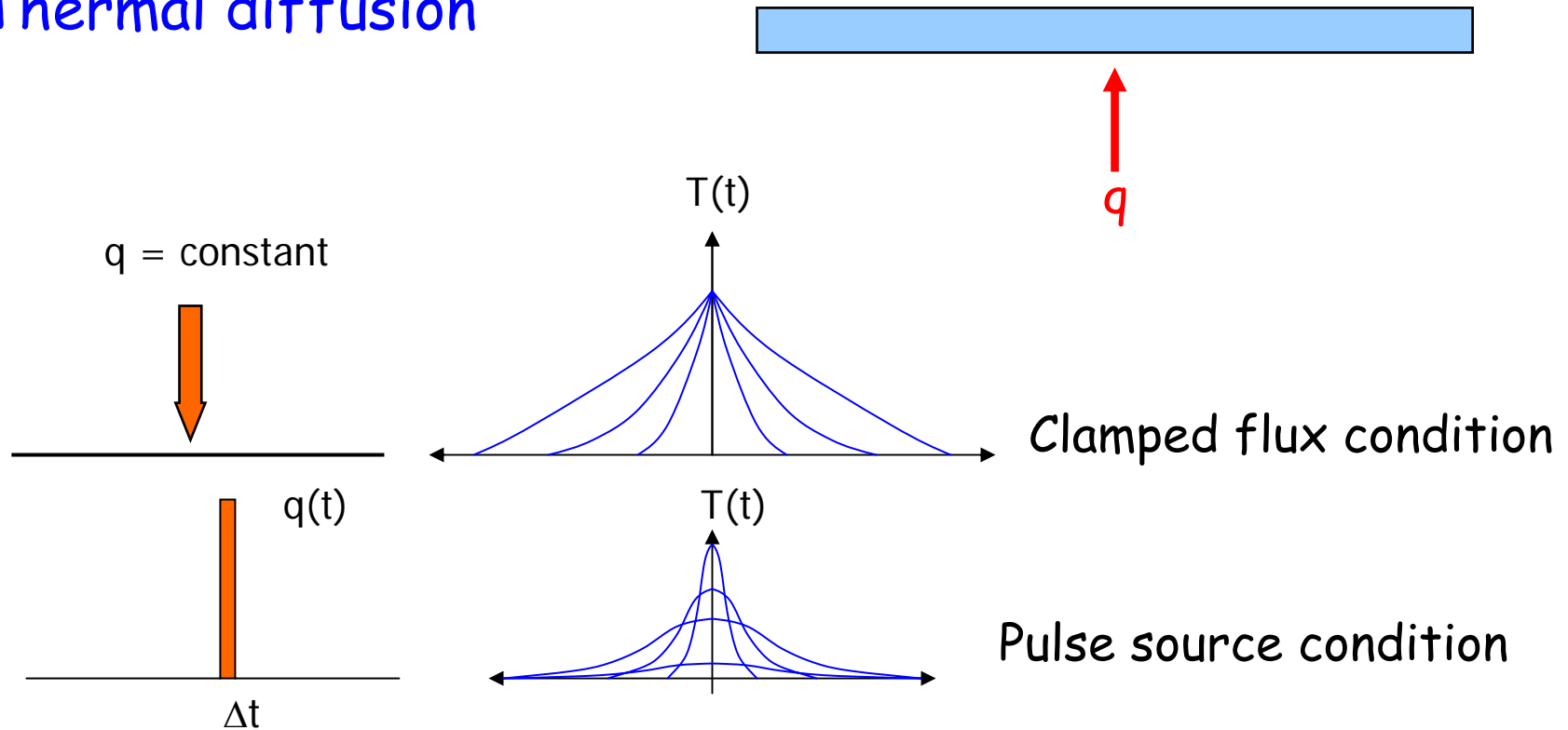
- Characteristic velocity of second sound  $C_2 \sim 20$  m/s ( $\sim 1/12$  that of first sound,  $C_1$ )
- High intensity second sound pulses create turbulence and thermal shock



$$Q > 10 \text{ W/cm}^2$$
$$\Delta t < 1 \text{ ms}$$

# Transient Heat Transport: Diffusion

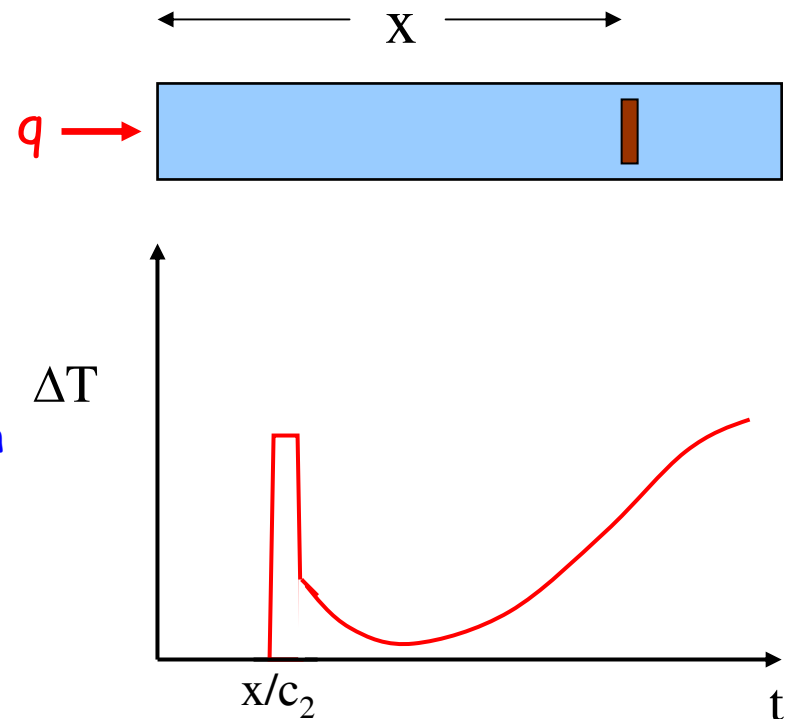
- Thermal diffusion



These processes are described by the He II non-linear diffusion equation

# Summary: He II diffusion + 2<sup>nd</sup> sound shock

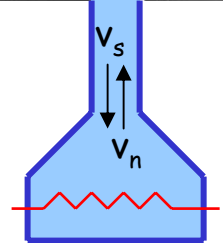
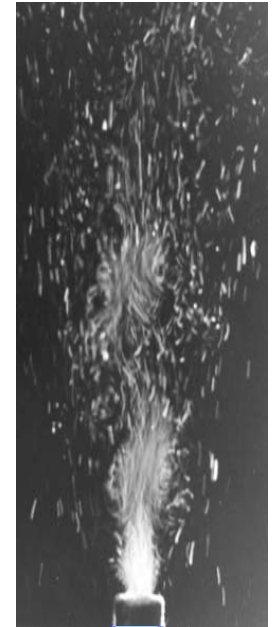
- Short, intense pulses carried by second sound
- Longer, less intense pulses diffuse from heat source into bulk He II
- Both processes are described in terms of average values of  $v_n$  and  $v_s$ .
- Previous work limited to counterflow in 1D
  - $dv_n/dx$  is small





# How do we visualize these phenomena?

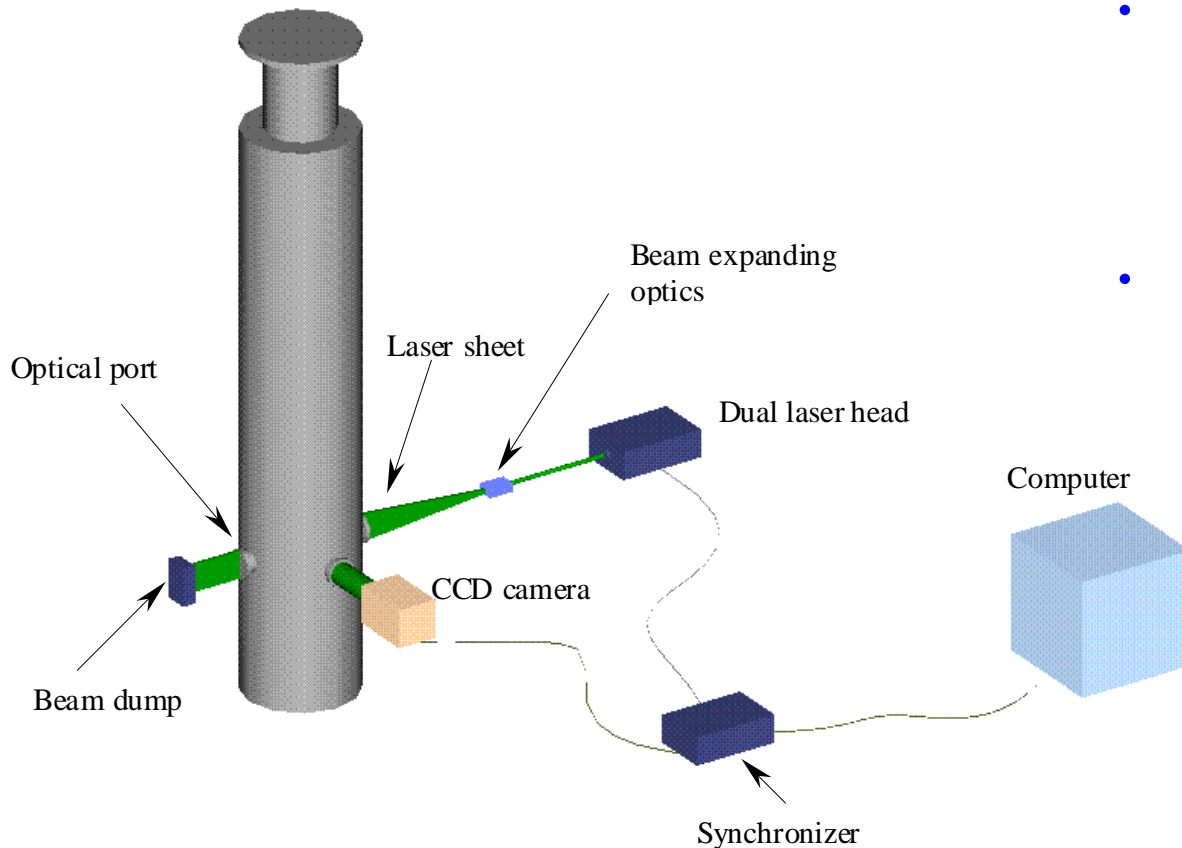
- Previous attempts to visualize motion of the fluid components in He II
  - Tagging with ions (Schwarz, et al)
  - Droplets of  $^3\text{He}$  (Lucas, et al)
  - Laser Doppler Velocimetry (Murakami, et al)
    - $\text{H}_2/\text{D}_2$  solid particles  $d \sim 10 \mu\text{m}$
    - Glass spheres  $20 \mu\text{m} < d < 100 \mu\text{m}$ ,  $\text{Re} \sim 8000$  →
- Particle Image Velocimetry (PIV)
  - Insert neutral density particles ( $d \sim 1 \mu\text{m}$ )
  - Monitor motion of particles to determine if they follow the normal fluid component
  - Whole field measurement
  - Study complex geometries and transient effects



Counterflow Jet

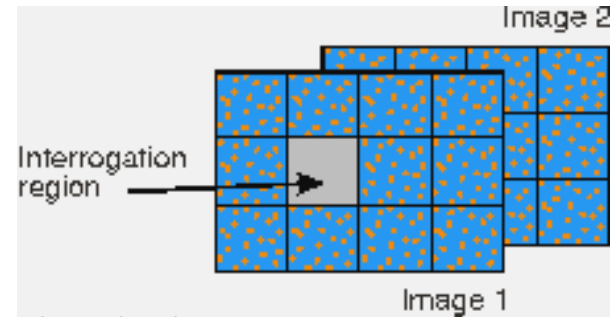
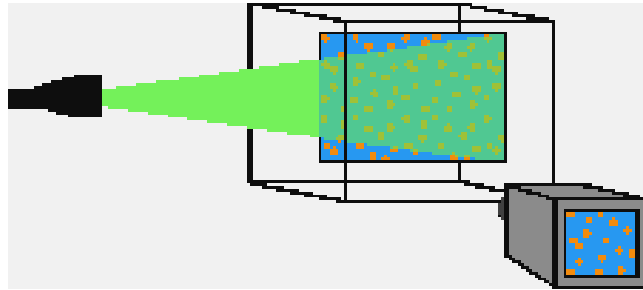


# Particle-Image-Velocimetry Measurement in He II

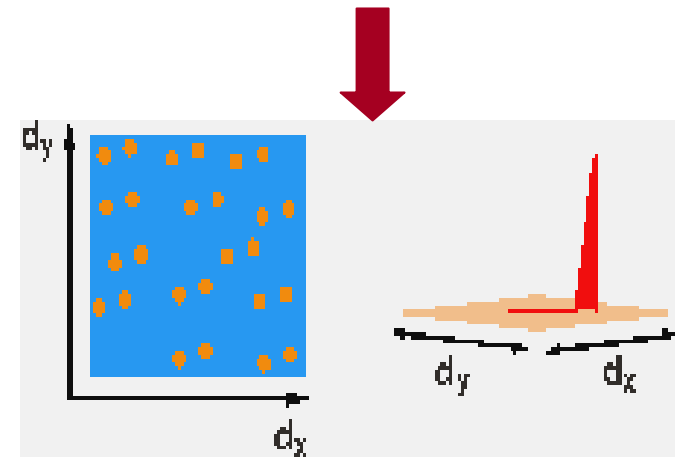
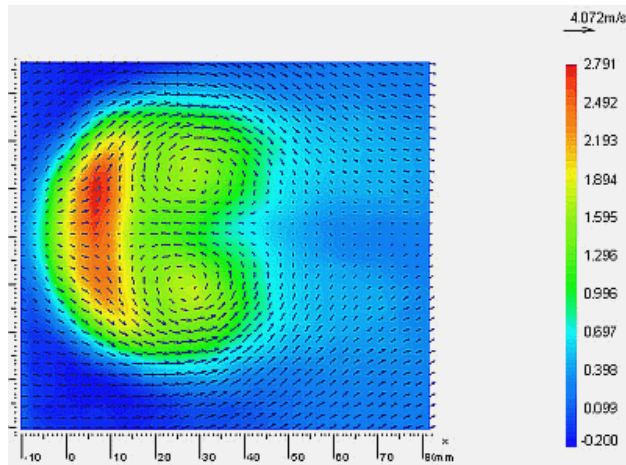


- Particle Image Velocimetry (PIV) studies of flow fields in He II
- Requirements of technique
  - Pulse laser
    - Nd:YAG: (532 nm)
    - Diode (795 nm)
  - Optical cryostat with image collection system
  - Neutral density particles ( $d < 10 \mu\text{m}$ )

# Particle Image Velocimetry Technique



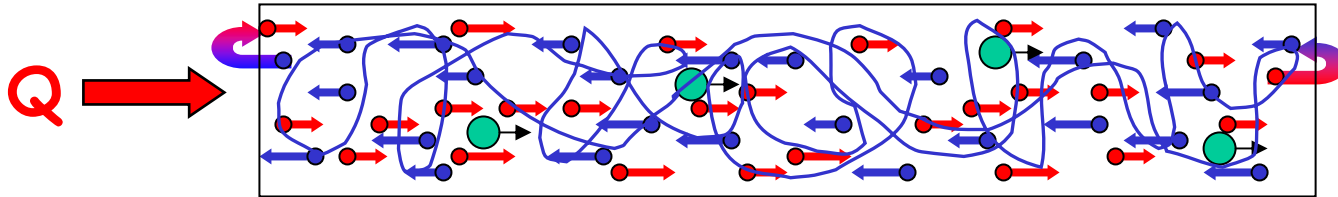
1. Flow seeding and image acquisition    2. Subdivided into interrogation windows



4. Result - velocity field

3. Correlation process

# Particle -interaction with two fluid components



- Normal and superfluid components are not visible or separable. Need tracer particles
- Insert solid particles in He II channel
- Dimensional considerations
  - Particle diameter ( $\sim 1 \mu\text{m}$ )
  - Vortex core ( $< 1 \text{ nm}$ )
  - Vortex line spacing ( $\delta \sim \mu\text{m}$ )
- How do these particles interact with the He II?



Potential flow  
around a sphere

# Particle Tracking Characteristics

- Concentration

- Statistics for velocity measurement
- Particle-fluid interaction affect dynamics

$$\left. \begin{array}{l} C \sim 10^9/\text{m}^3 \\ d_p \sim 10 \mu\text{m} \end{array} \right\}$$

- Slip velocity between particles and fluid

$$v_n - v_p = \frac{(\rho_p - \rho_{\text{HeII}})d_p^2 g}{18\phi\mu_n}$$

$$\phi = 1 + f(\text{Re}_p) \quad \text{For deviation from Stokes law}$$

- Small viscosity ( $\sim 1.4 \mu\text{Pa}\cdot\text{s}$ ) makes slip a significant issue
- Particle density should be  $\sim \rho_{\text{HeII}} \sim 145 \text{ kg}/\text{m}^3$
- Extremely small particle size ( $d < 10 \mu\text{m}$ )

- Response time

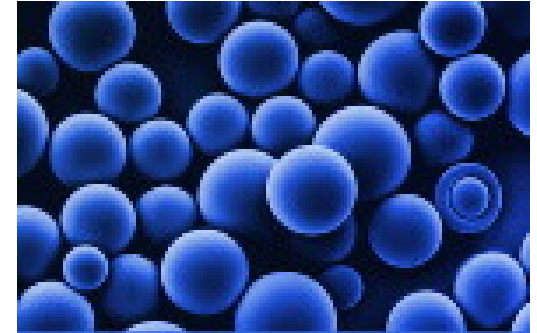
$$\tau \equiv \frac{(\rho_p + \frac{1}{2}\rho_{\text{HeII}})d_p^2}{18\phi\mu_n}$$

$$\tau \sim 1 \text{ ms for } d_p = 10 \mu\text{m}$$



# Candidate particles for seeding

- Hollow glass spheres
  - Density around  $160 \text{ kg/m}^3$
  - Particle size distributed from  $30$  to  $120 \text{ }\mu\text{m}$
- Polymer micro-spheres
  - High density around  $1100 \text{ kg/m}^3$
  - Small uniform diameter  $\sim 1.7 \text{ }\mu\text{m}$
- Solidified  $\text{H}_2/\text{D}_2$  particles (condensed from gas phase)
  - Density around that of  $\text{LHe}$  ( $125$  to  $145 \text{ kg/m}^3$ )



Type of particles	Size	Slip velocity	Relaxation time
Hollow glass spheres	$30 \text{ }\mu\text{m}$	$3.12 \text{ mm/s}$	$8.1 \text{ ms}$
	$120 \text{ }\mu\text{m}$	$13.8 \text{ mm/s}$	$130 \text{ ms}$
Polymer micro-spheres	$1.7 \text{ }\mu\text{m}$	$1.0 \text{ mm/s}$	$0.13 \text{ ms}$
Solid $\text{H}_2/\text{D}_2$ particles	$\sim 10 \text{ }\mu\text{m}$	$\sim 0.2 \text{ mm/s}$	$\sim 0.9 \text{ ms}$

# Introducing Particles into He II

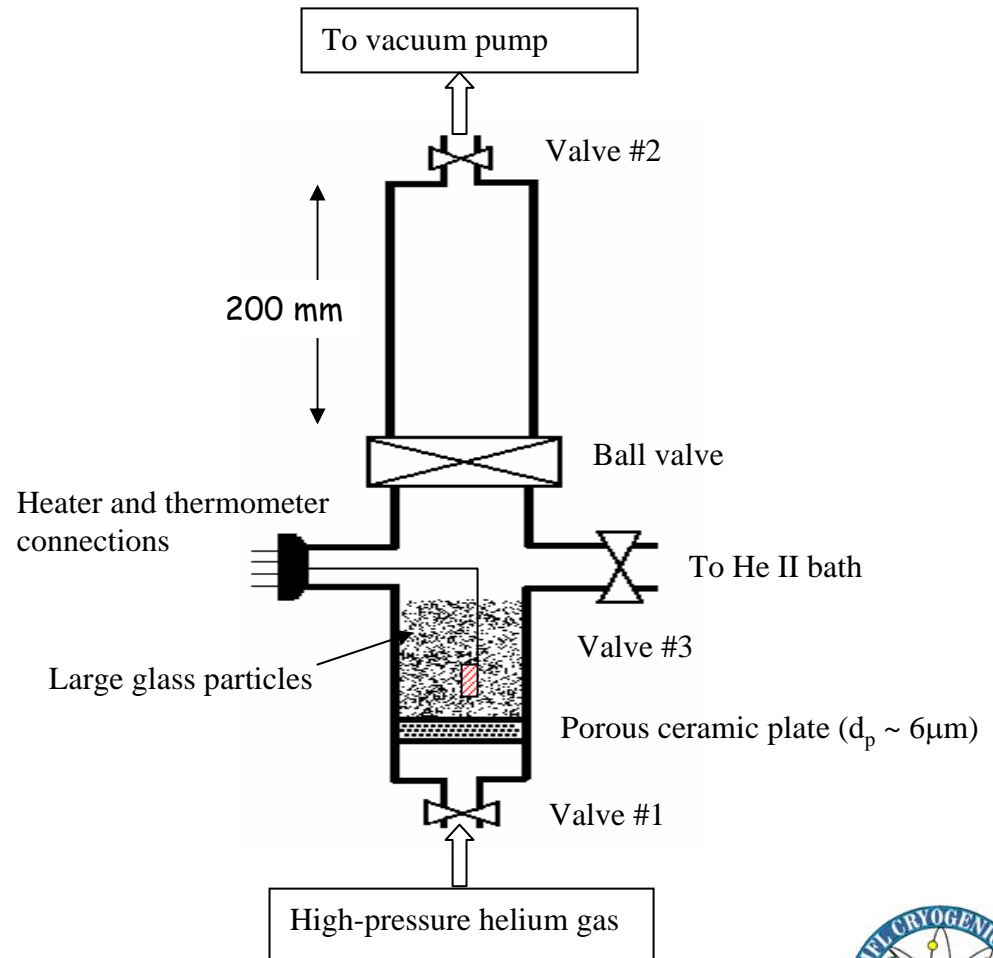
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- Initial conditions:
  - Solid particles are stored at room temperature in air (or other gas)
  - $H_2$  &  $D_2$  is in gaseous state at room temperature
- Experimental conditions
  - $T_{op} \sim 2$  K
  - $P_{op} < 0.05$  atm
- Issues of concern in He II application
  - Since all gases (except He) freeze at 2 K, careful purging with He gas is necessary prior to injection into He II
  - Introducing particles into low pressure environment requires particle transport system



# Seeding solid particles

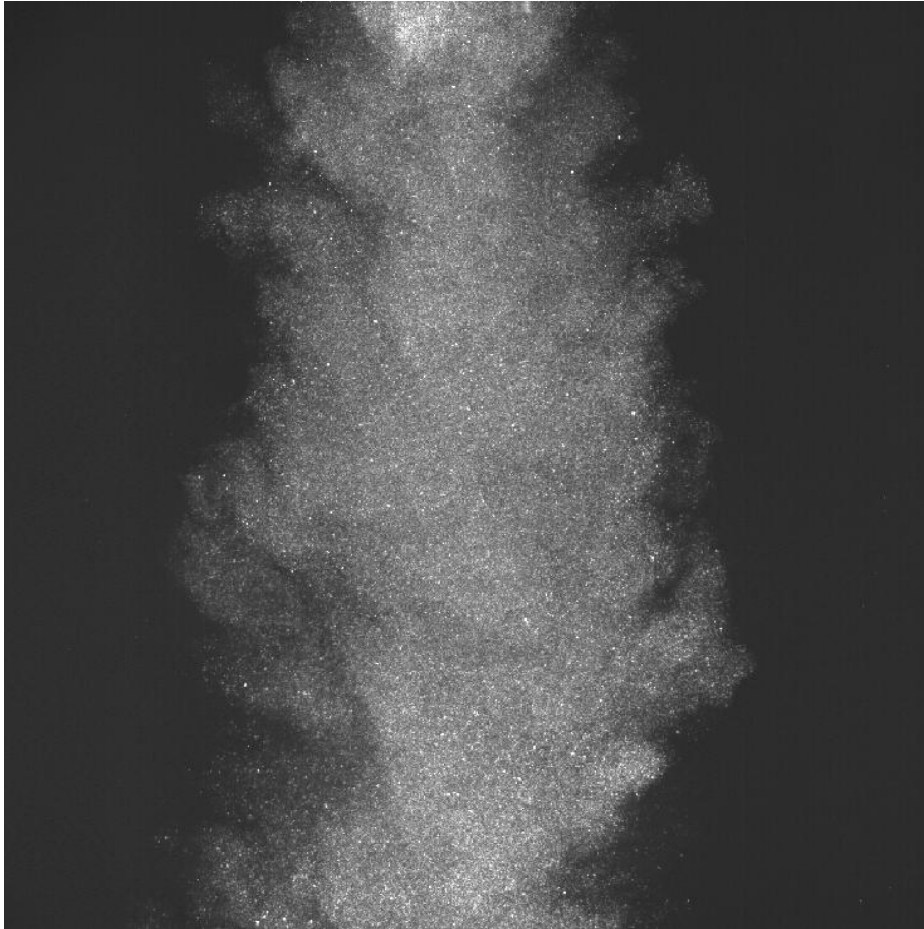
- Seeding requirements
  - Adequate seeding concentration
  - Minimize coagulation of particles
- Method
  - Seeding by fluidized bed technique to reduce aggregation :
    - large glass particles mixed with visualization particles
    - Purging with pure helium gas
    - Suspended particles transported in He gas stream





# Particle seeding results

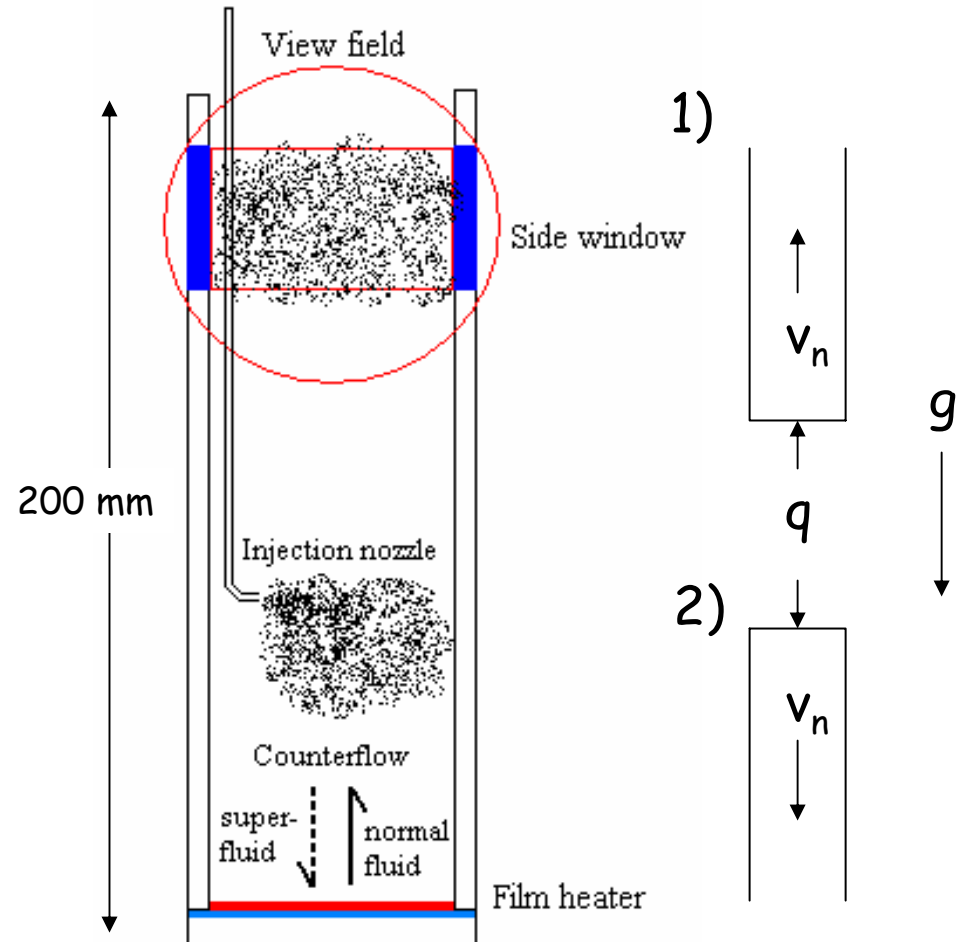
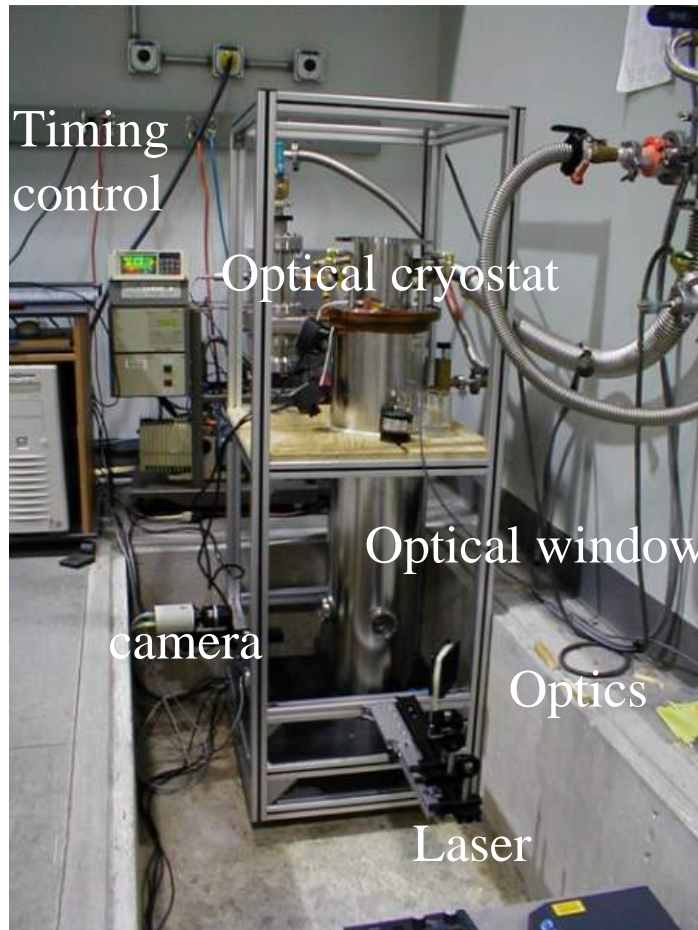
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- Stable delivery of particles.
- Adequate seeding concentration.
- Aggregation of particles is reduced.
- The size of seeded particle is more uniform.
- See: T. Zhang, D. Celik and S. van Sciver, JLTP Vol. 134, 985 (2004)

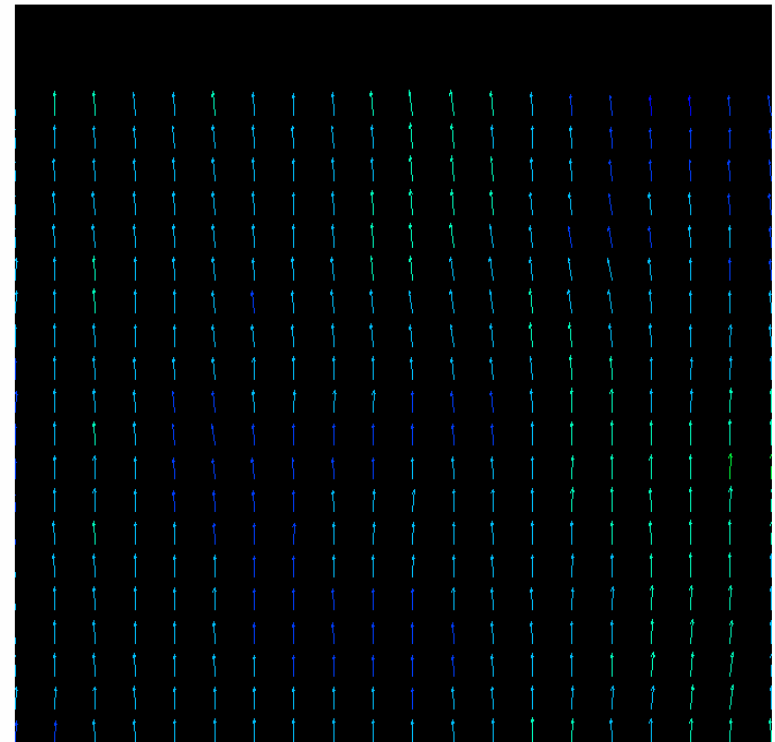
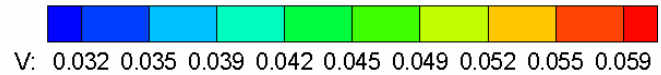
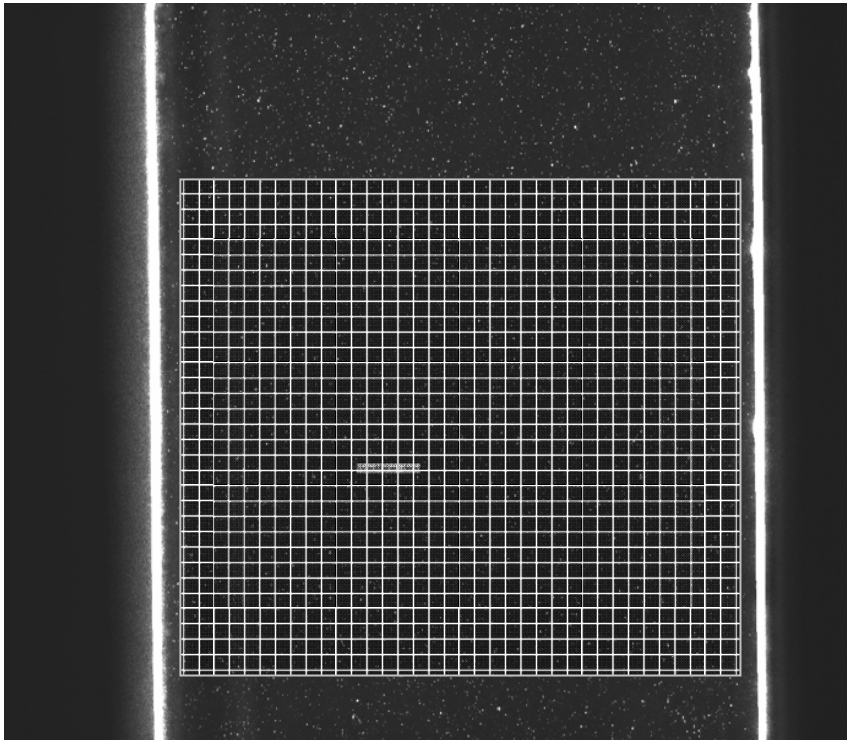


# PIV He II Counterflow Experiment



43 x 19.5 mm rectangular channel

# Steady State Results



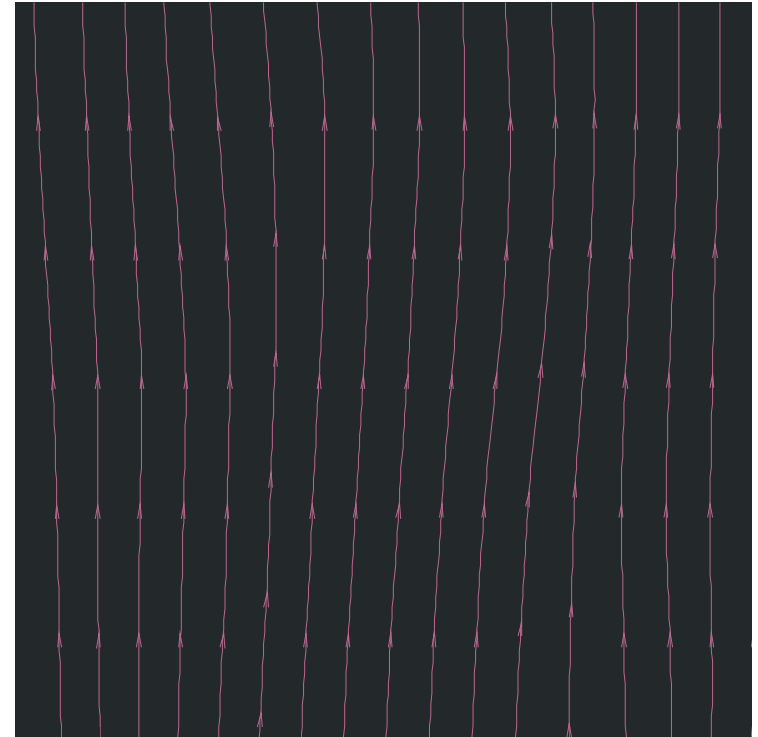
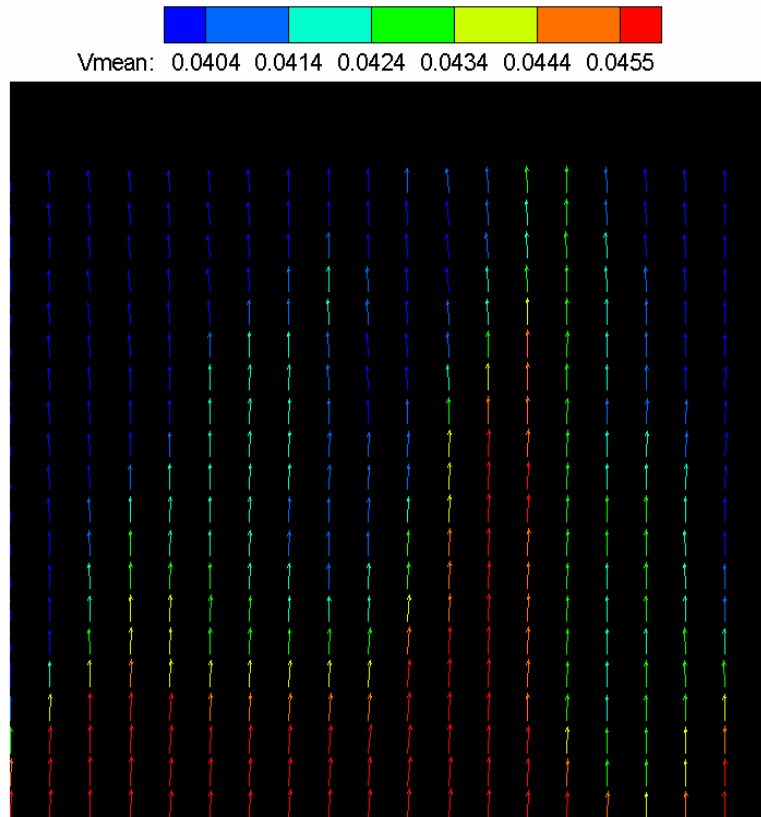
## Image Acquisition Process

- Seeding particles are polymer spheres of  $\phi = 1.7 \mu\text{m}$ ,  $1100 \text{ kg/m}^3$
- Calculated slip velocity  $\sim 1 \text{ mm/s}$
- Bright lines are channel walls

For each pair of images, the instant velocity field is obtained

- CCD camera operated at 5 Hz.
- 25 Pairs of images collected within around 5 seconds.

# Analysis of PIV in counterflow He II

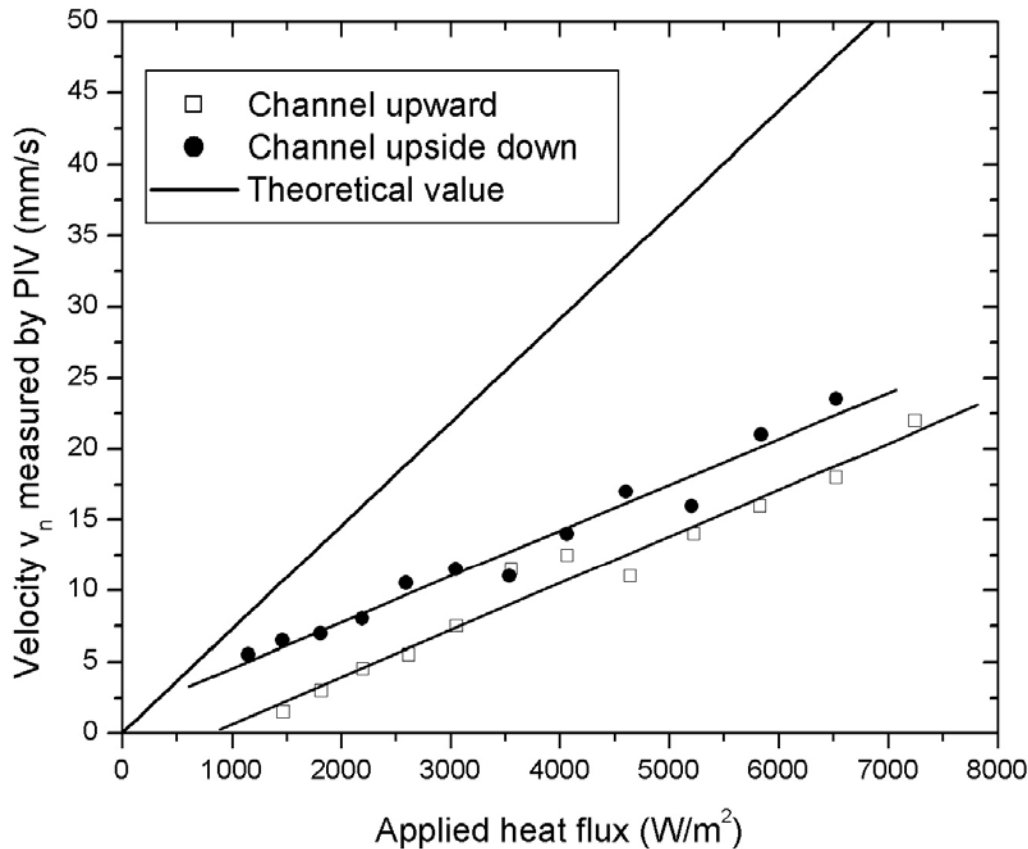


Averaged velocity field at 1.62 K and  $q = 7.24 \text{ kW/m}^2$

$\langle v \rangle \sim 40 - 45 \text{ mm/s}$  while,  $v_n = \frac{q}{\rho s T} \sim 100 \text{ mm/s}$

# Comparison with theoretical results

T=1.80 K



- PIV results represent the mean velocity of whole flow field.

- Theoretical value is calculated from

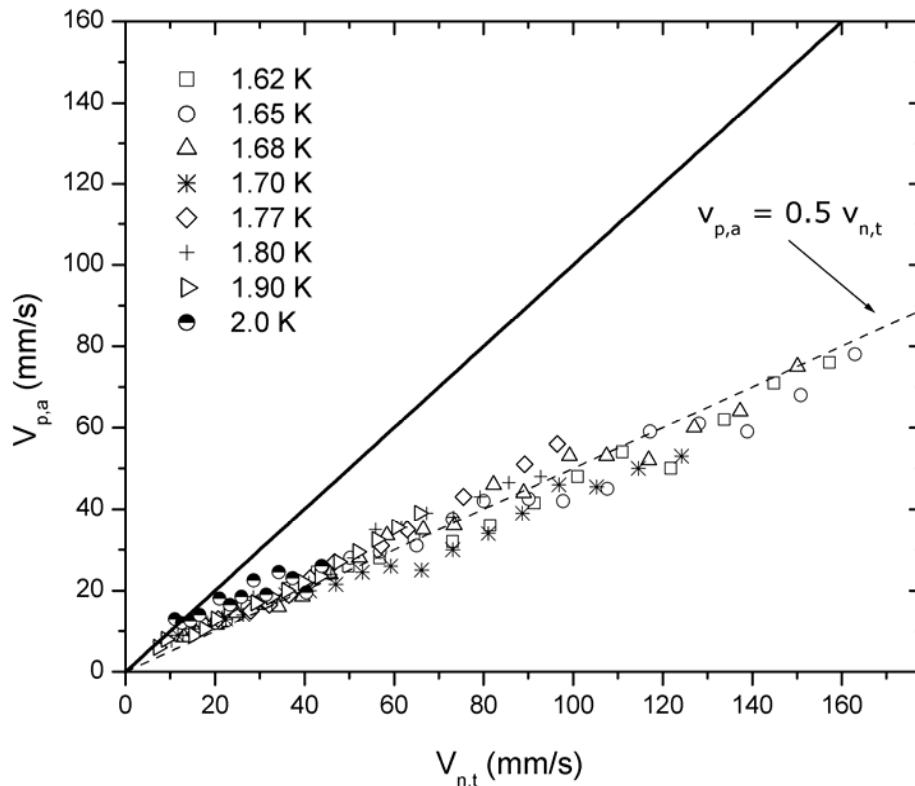
$$\bar{v}_n = \frac{q}{\rho s T}$$

- A disagreement is clearly shown.
- Slip velocity can be eliminated by averaging two configurations



# Results at various temperatures

## All temperatures

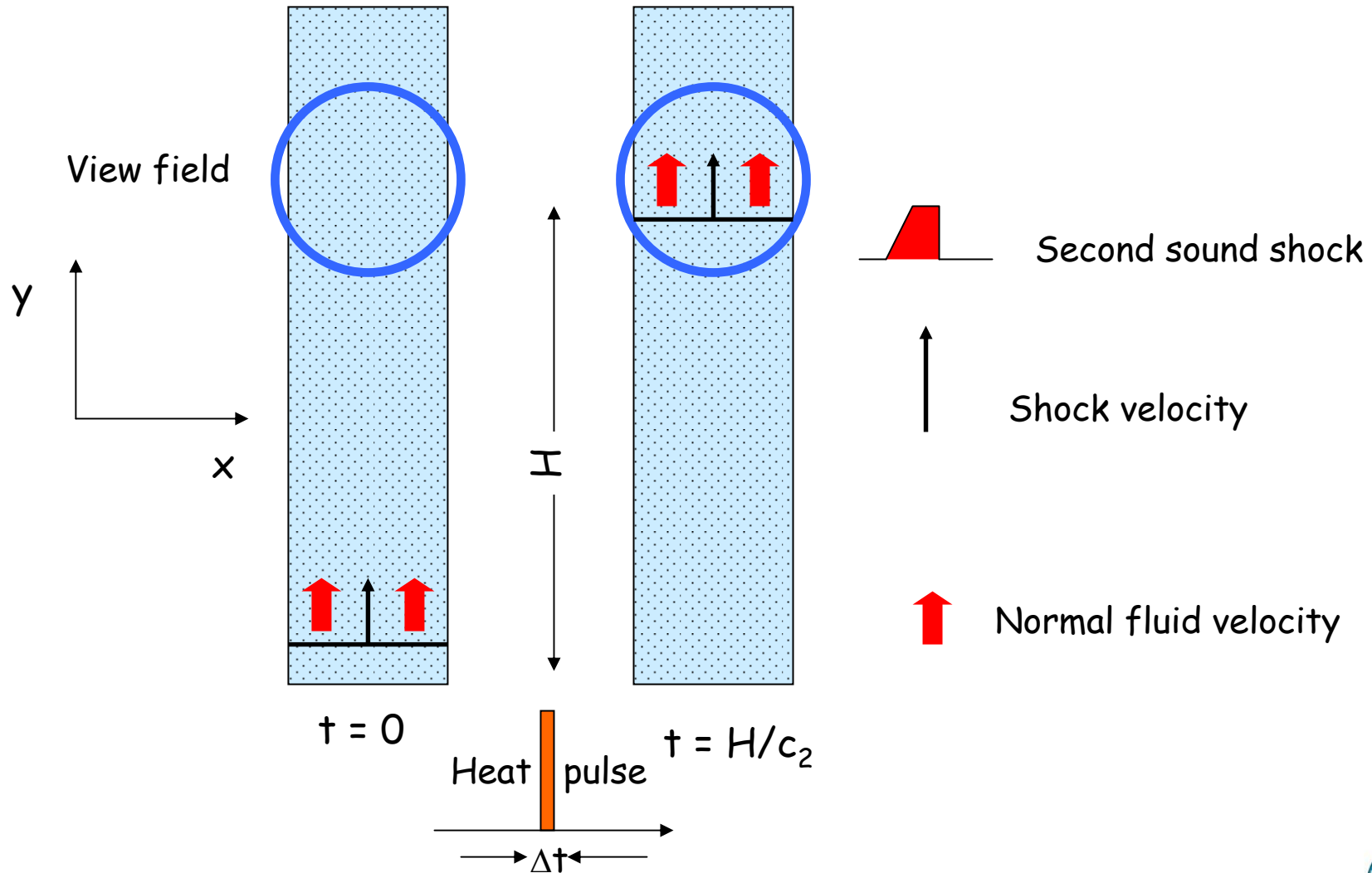


- Disagreement exists at all temperatures.  $v_p/v_n \sim 0.5$
- Different result from Nakano and Murakami (Cryogenics, 34, 179 (1994))

$$v_{n,measurement} = \frac{\rho_n}{\rho} v_{n,theory}$$

- Chung and Critchlow (PRL 14,892 (1965)) observed particle motion in pure superflow
- See T. Zhang & S. van Sciver, JLTTP Vol. 138, 865 (2005)

# PIV Measurement of Transient Counterflow



# Motion of tracer particles in the He II counterflow field



Bath temperature: 1.61 K.

Pulse width: 1 second.

Camera operating rate: 1200 Hz.

Time interval: 0.883 ms.

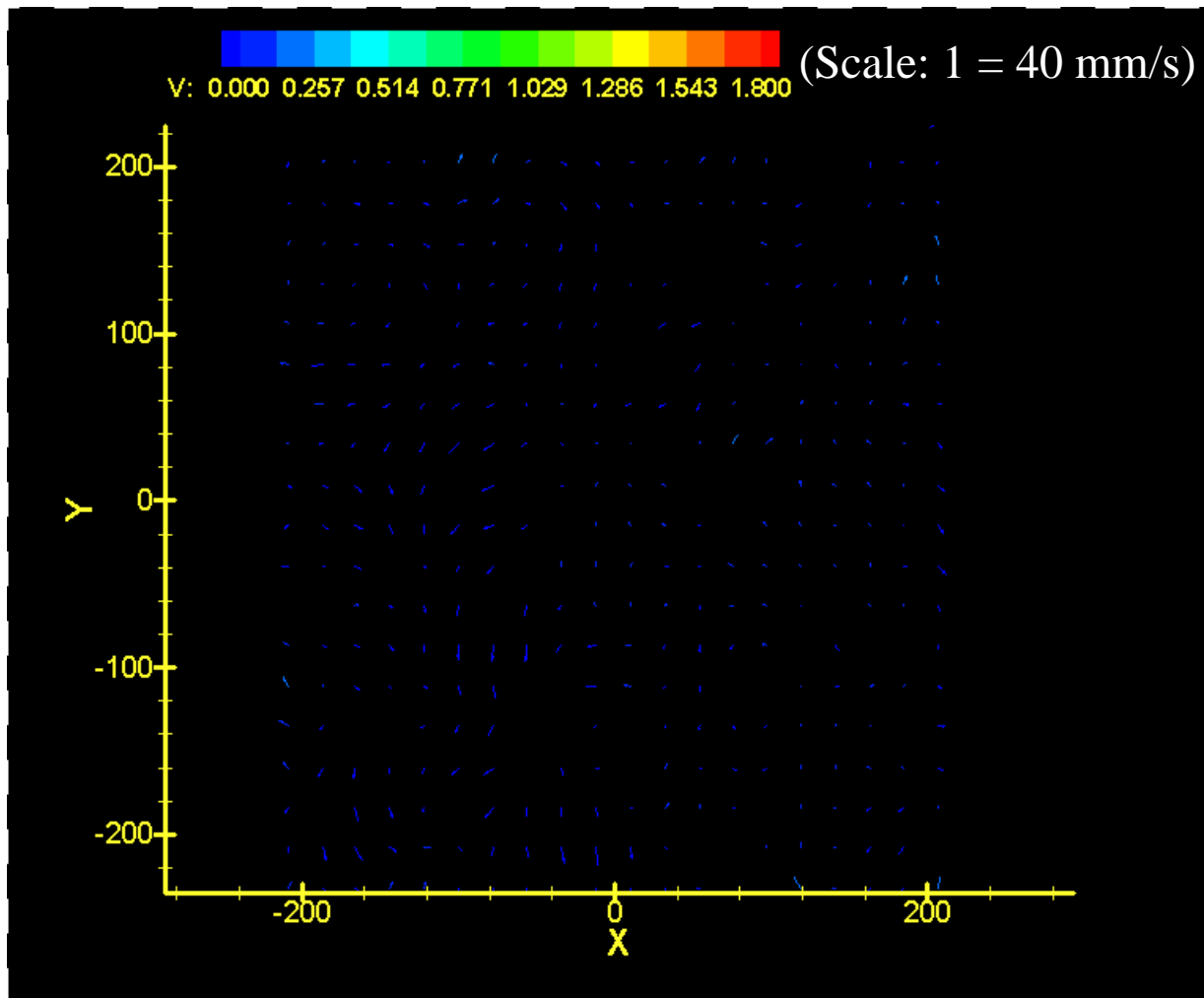
Total time: 883 ms.



$$Q = 11.2 \text{ kW/m}^2$$



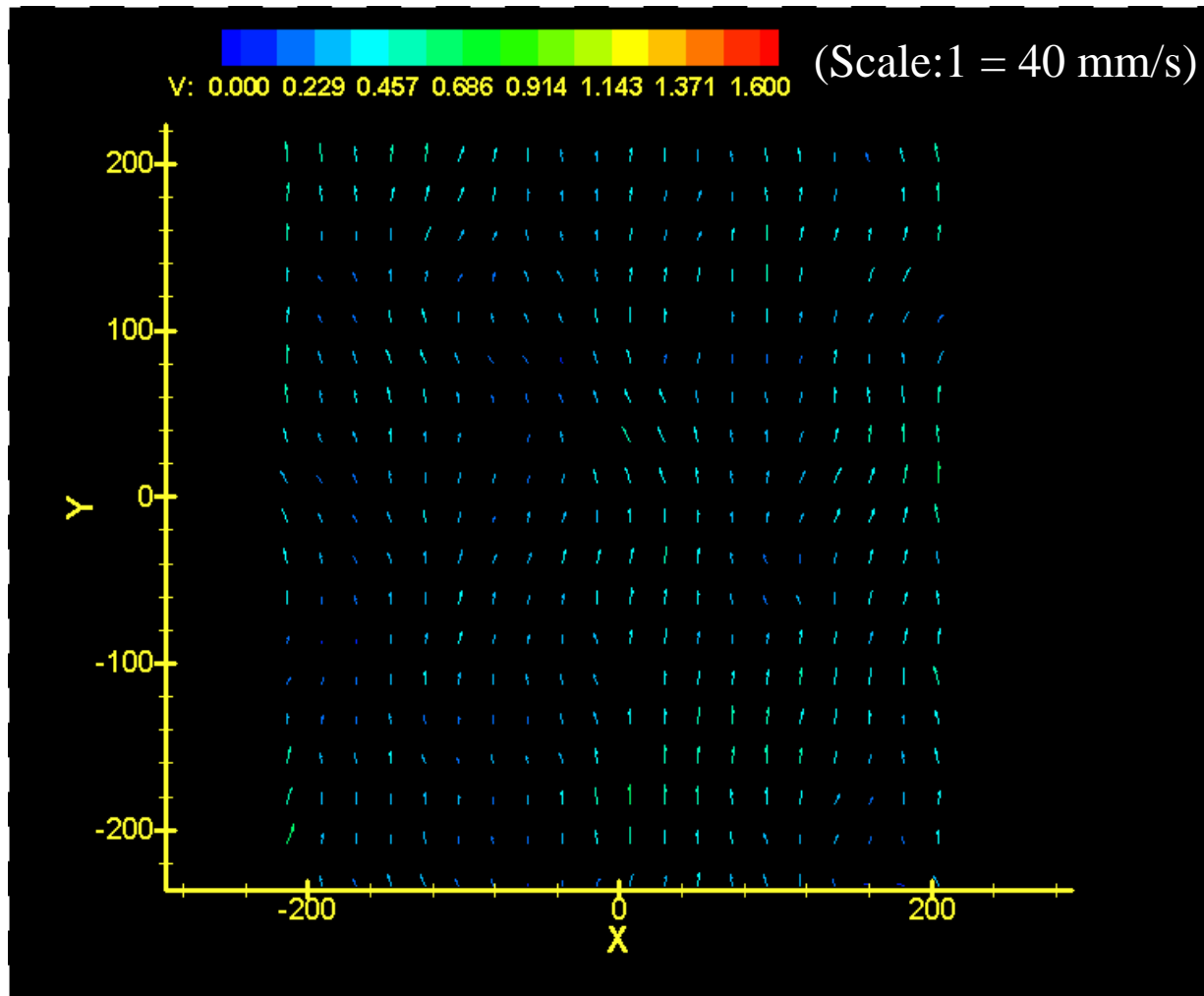
# Detection of second sound with PIV



Time under analysis: from 0 to 30 ms after heater was on.

Time interval between frames: 1.67 ms

# He II heat diffusion detection with PIV

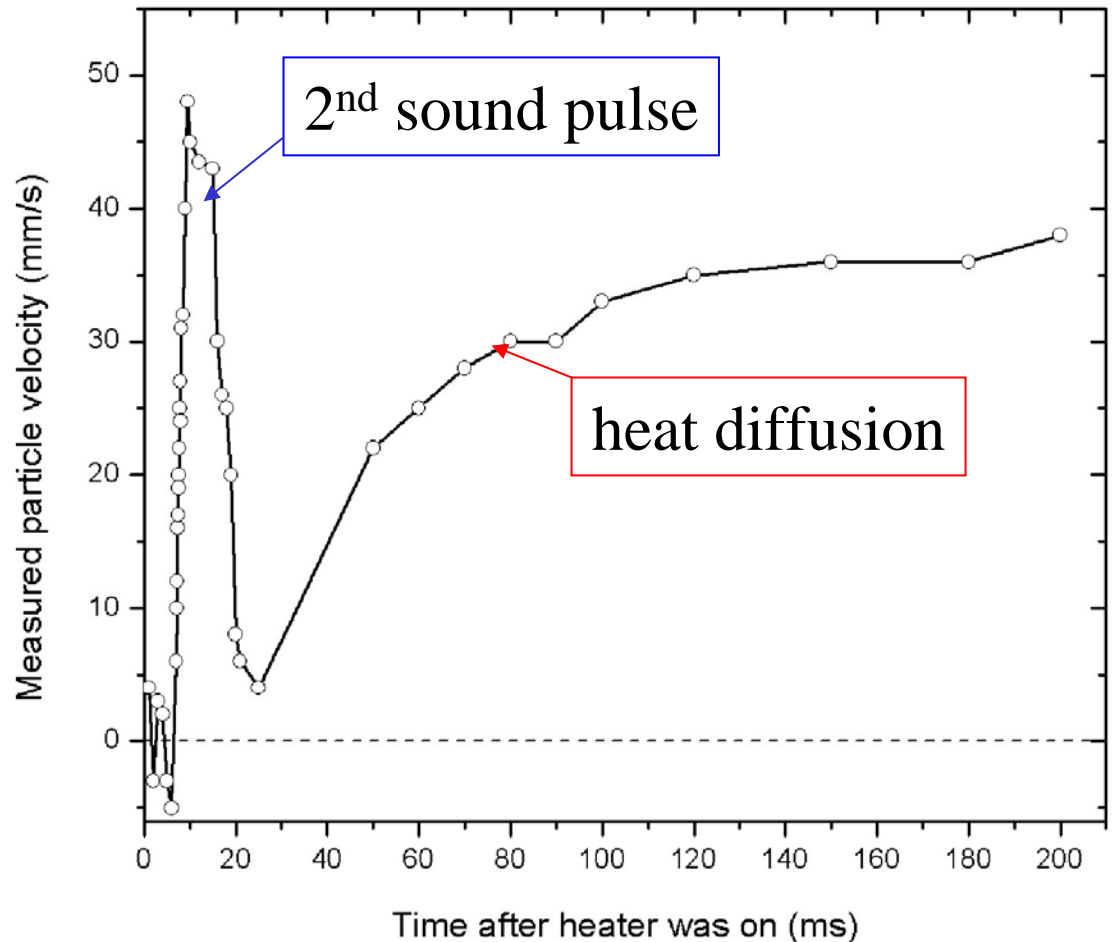


Time under analysis: from 30 to 830 ms after heater was on.

Time interval between frames: 10 ms.

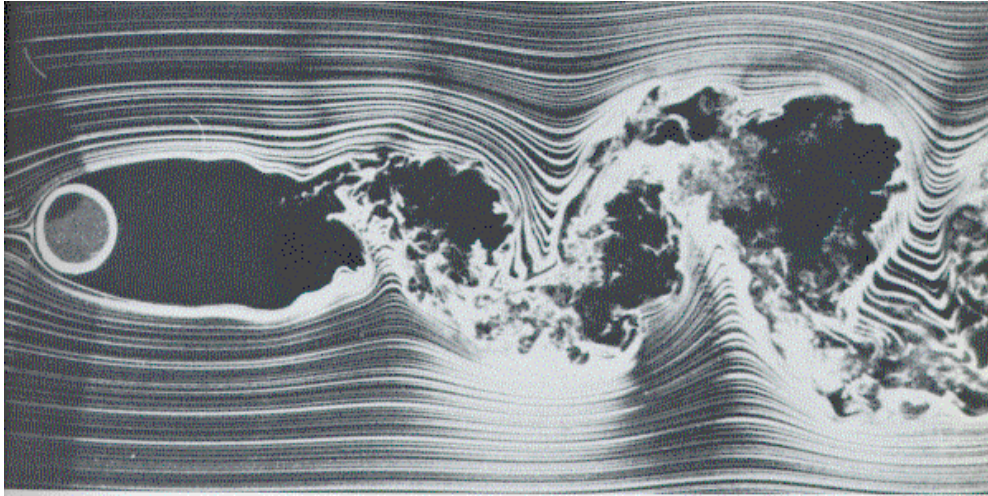
# PIV - Transient Heat Transport

- Arrival of 2<sup>nd</sup> sound pulse and heat diffusion tail are clearly visible using PIV.
- Characteristic particle velocities are less than  $v_n$  (similar result to steady state experiments)
- For details see: T. Zhang and S. Van Sciver, Phys. Fluids Vol. 16, L99 (2004)



# Counterflow around a cylinder

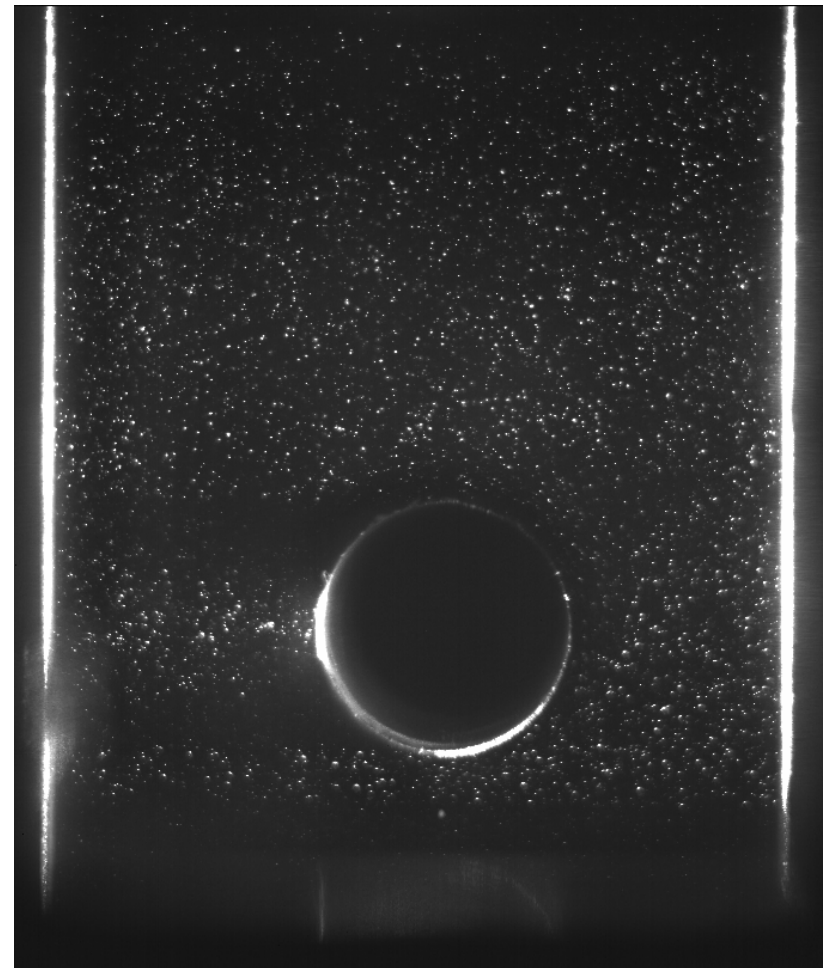
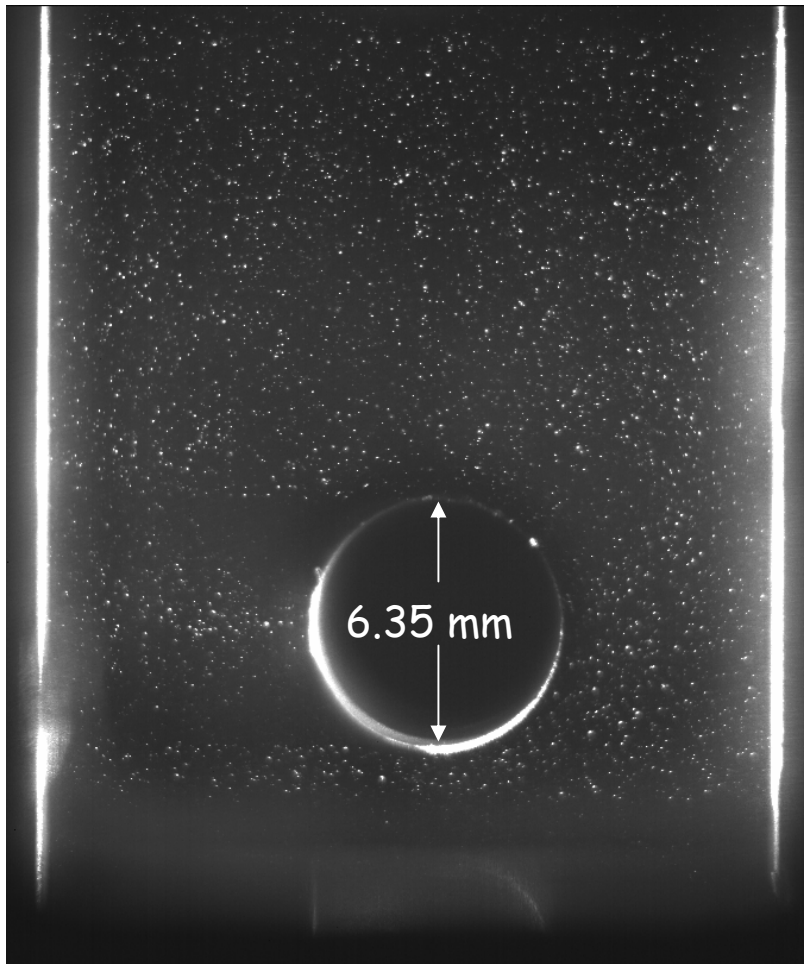
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$$\text{Re}_d = \frac{\rho v d}{\mu} \sim 10000$$

- This is a classic problem of fluid mechanics.
  - Large scale vortex shedding occurs behind the cylinder
  - Details scale with  $\text{Re}_d$
- Question: What if the flow over cylinder were counterflow He II?

# Particle motion - counterflow over cylinders



$T = 1.60 \text{ K}$ ,  $q = 8.7 \text{ kW/m}^2$ ,  $Re = 88040$

Frame rate = 5 Hz, movie is real time = 20 s

New Exp. Tech. for Quantum Turbulence

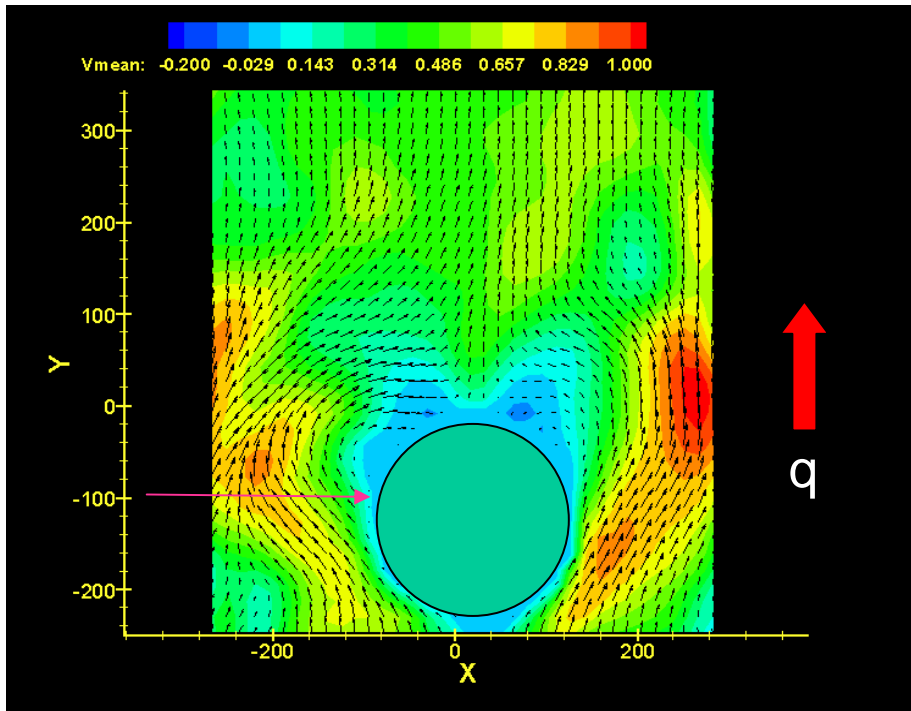
$T = 2.03 \text{ K}$ ,  $q = 14.1 \text{ kW/m}^2$ ,  $Re = 26044$

Trieste, Italy May 9-10, 2005

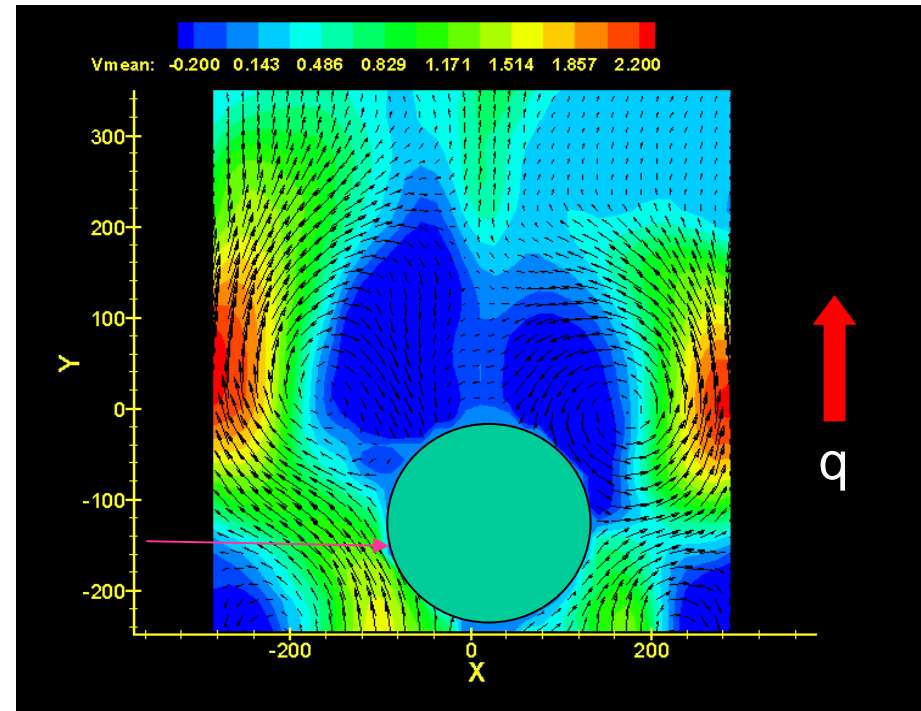


# Temperature dependence (Matching Re)

$$Re_n = \frac{\rho v_n d}{\mu_n}$$



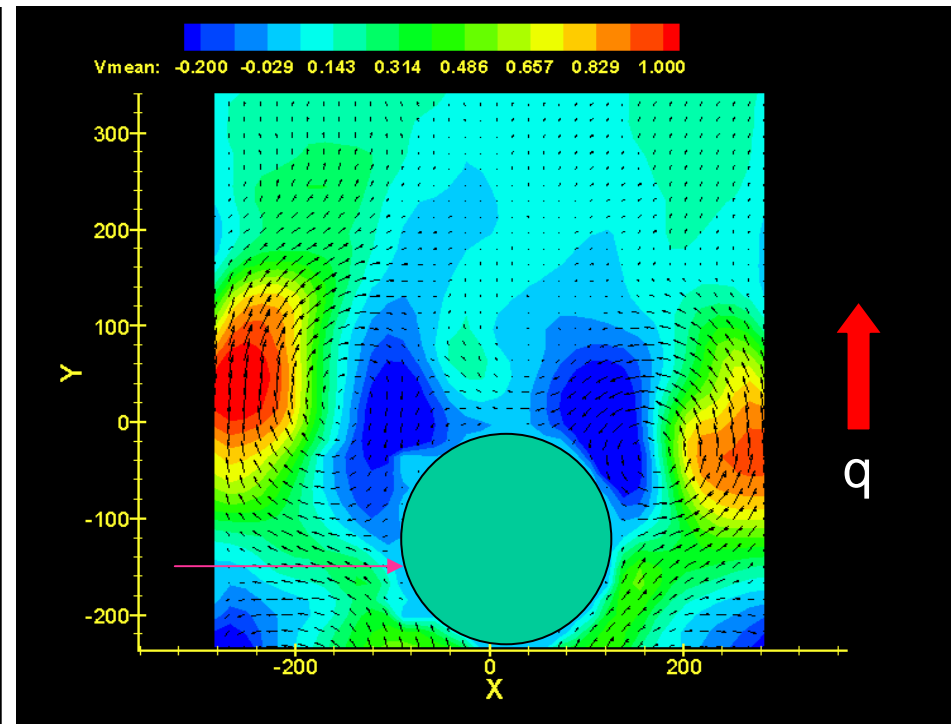
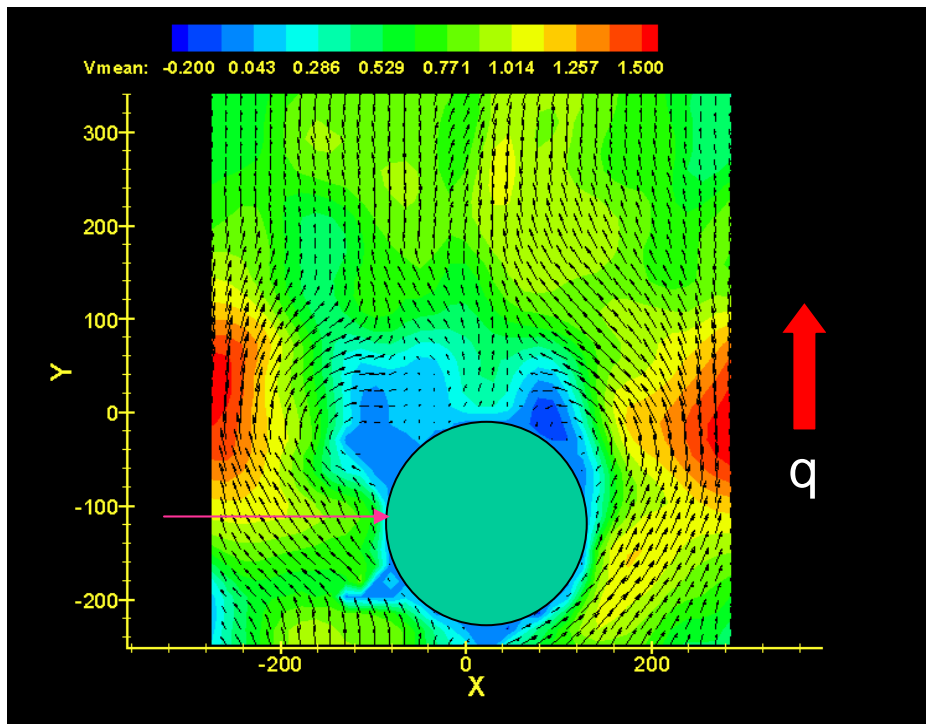
$T = 1.60 \text{ K}, I = 120 \text{ mA}, q = 2.59 \text{ kW/m}^2$   
 $Re = 26194 \quad v_n = 39.4 \text{ mm/s}$   
 $Re_n = 4828 \quad v_s = 8.9 \text{ mm/s}$



$T = 2.03 \text{ K}, I = 280 \text{ mA}, q = 14.1 \text{ kW/m}^2$   
 $Re = 26044 \quad v_n = 45.7 \text{ mm/s}$   
 $Re_n = 17775 \quad v_s = 98.2 \text{ mm/s}$

# Temperature dependence (Matching $Re_n$ )

$$Re_n = \frac{\rho_n v_n d}{\mu_n}$$



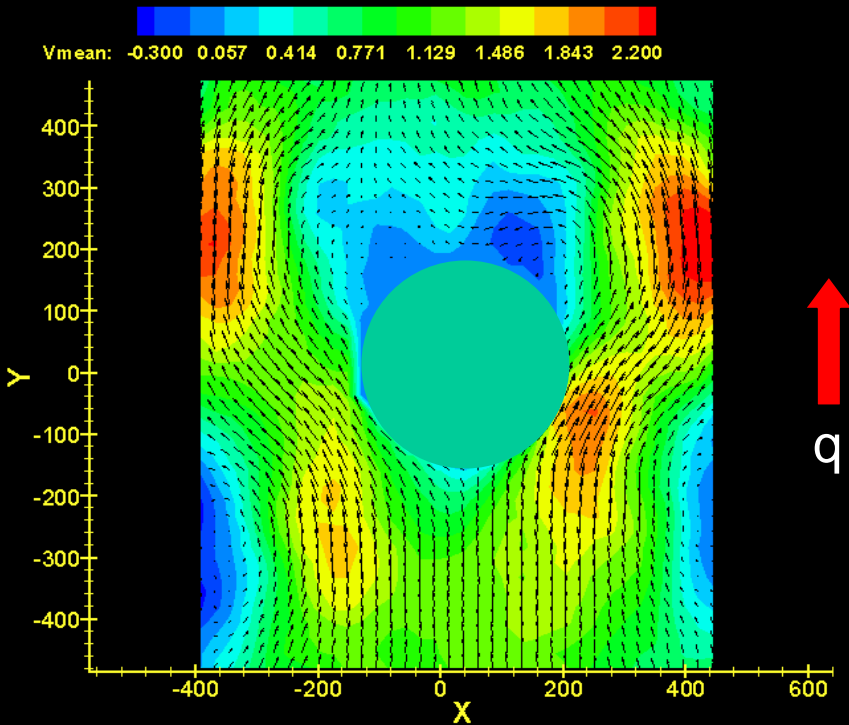
$T = 1.60 \text{ K}, I = 150 \text{ mA}, q = 4.04 \text{ kW/m}^2$   
 $Re = 40928 \quad v_n = 61.5 \text{ mm/s}$   
 $Re_n = 7543 \quad v_s = 13.9 \text{ mm/s}$

$T = 2.03 \text{ K}, I = 180 \text{ mA}, q = 5.82 \text{ kW/m}^2$   
 $Re = 10763 \quad v_n = 18.9 \text{ mm/s}$   
 $Re_n = 7346 \quad v_s = 40.6 \text{ mm/s}$

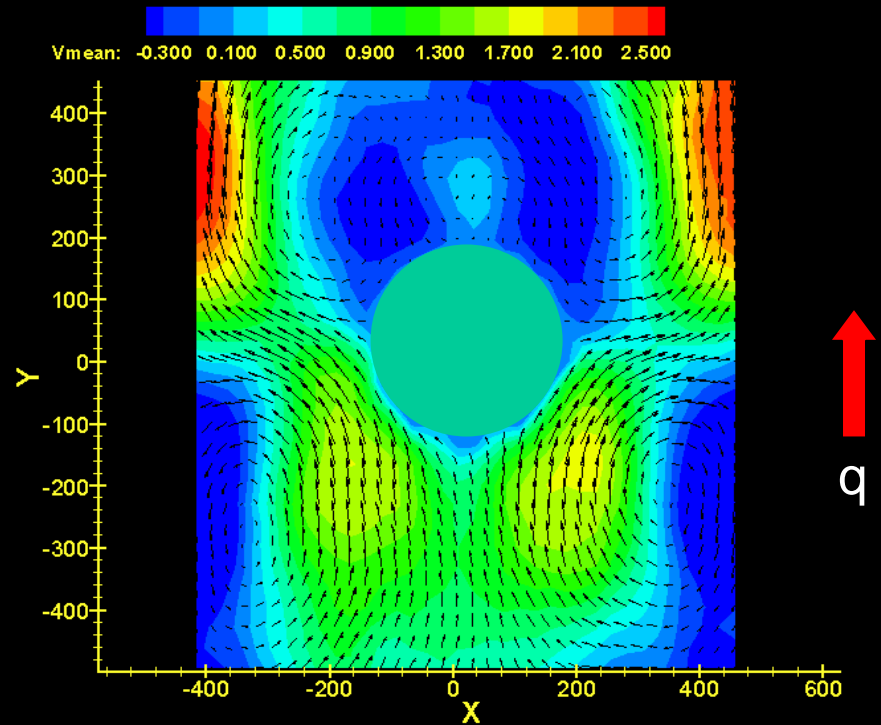


# Convection in front of cylinder

Note: 1 pixel/ms = 22 mm/s



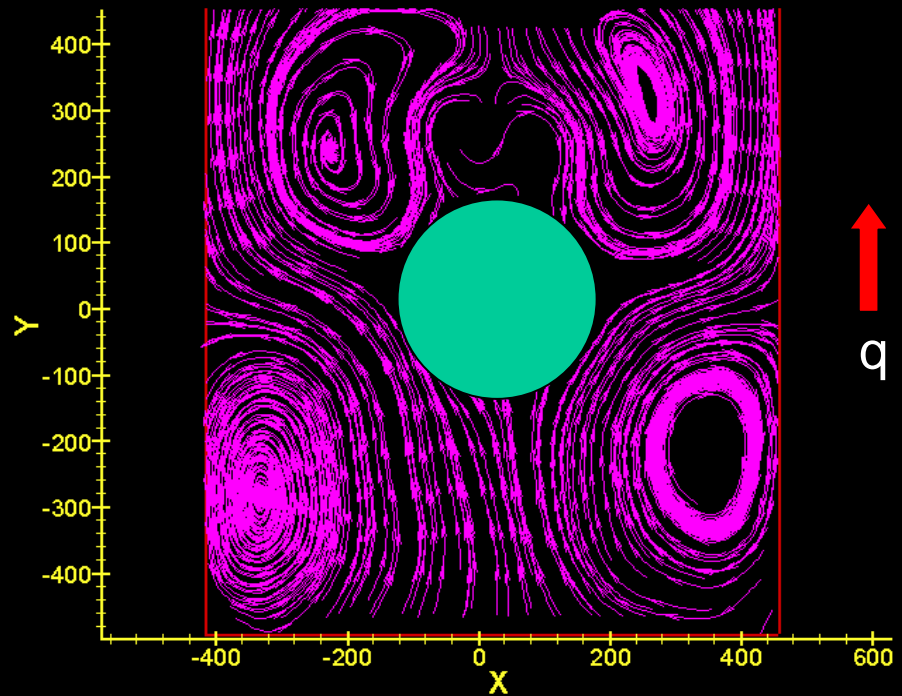
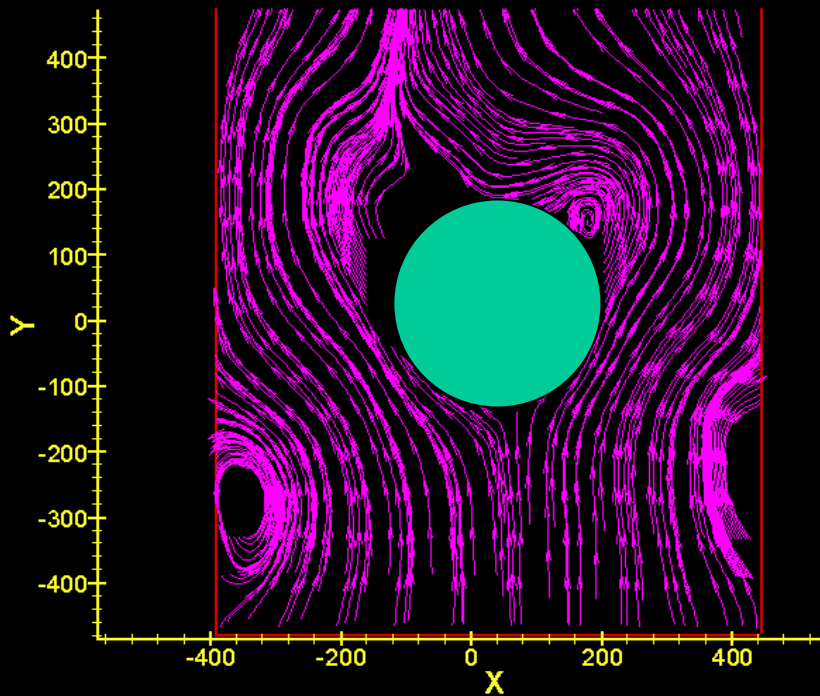
$T = 1.60 \text{ K}$ ,  $q = 4.04 \text{ kW/m}^2$ ,  $Re = 40928$



$T = 2.03 \text{ K}$ ,  $q = 11.2 \text{ kW/m}^2$ ,  $Re = 20762$



# Streamlines confirm vorticity



$T = 1.60 \text{ K}, q = 4.04 \text{ kW/m}^2, Re = 40928$      $T = 2.03 \text{ K}, q = 11.2 \text{ kW/m}^2, Re = 20762$

# Smaller cylinder (2 mm) to reduce wall interference

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Parameters:

$$q = 9.1 \text{ kW/m}^2$$

$$T = 2.0 \text{ K}$$

$$v_n = 32 \text{ mm/s}$$

$$v_s = 55 \text{ mm/s}$$

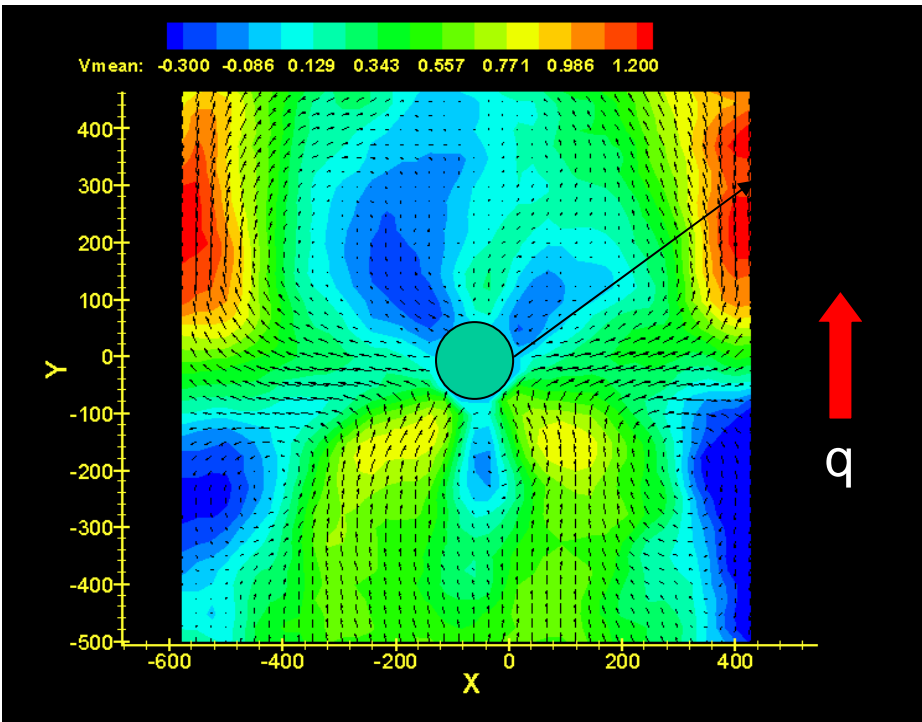
$$Re = 6150$$

$$Re_n = 3870$$

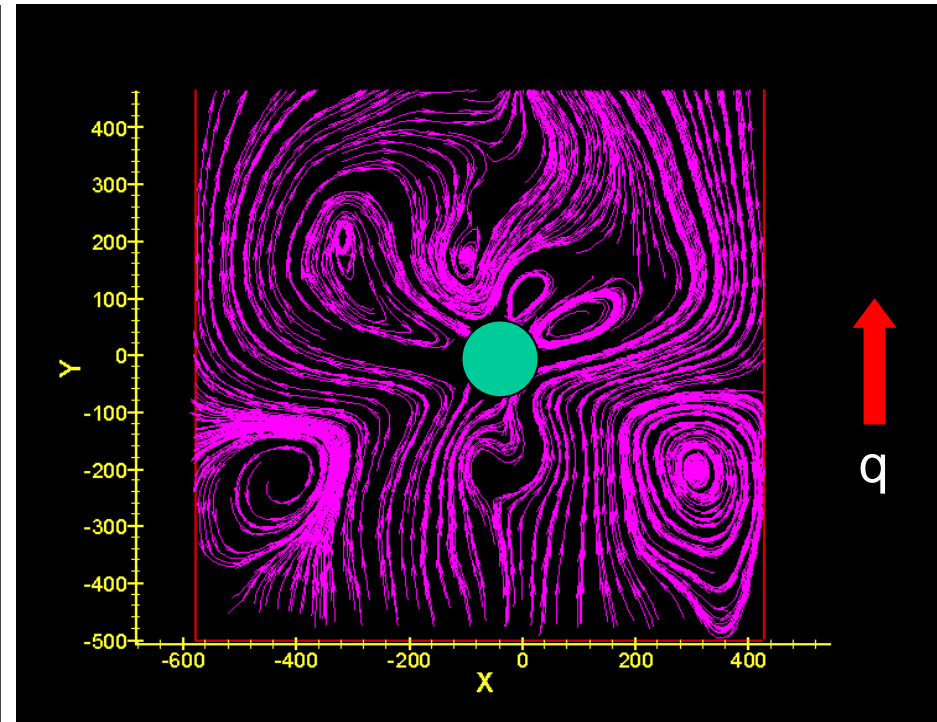
$$\text{Movie run time} = 30 \text{ s}$$

$$\text{Actual run time} = 0.48 \text{ s}$$

# Smaller cylinder (2 mm) to reduce wall interference



average velocity field

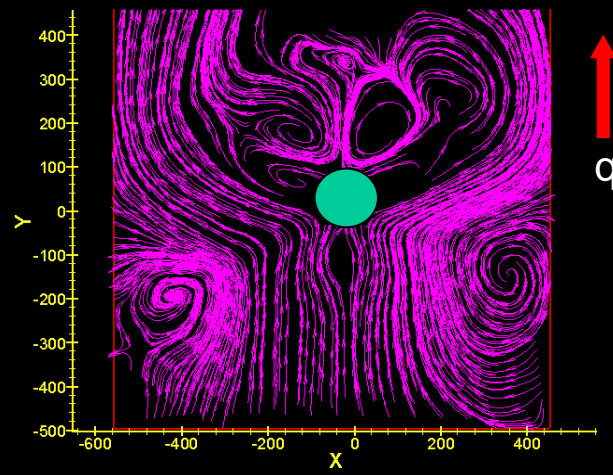
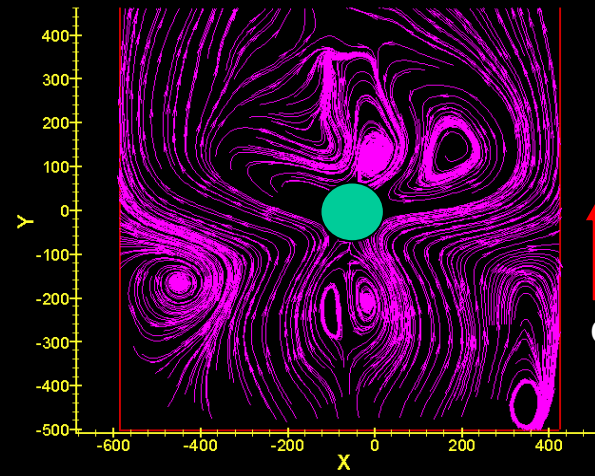
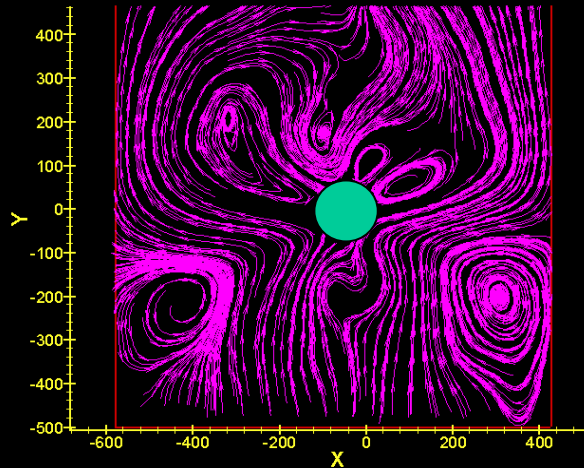


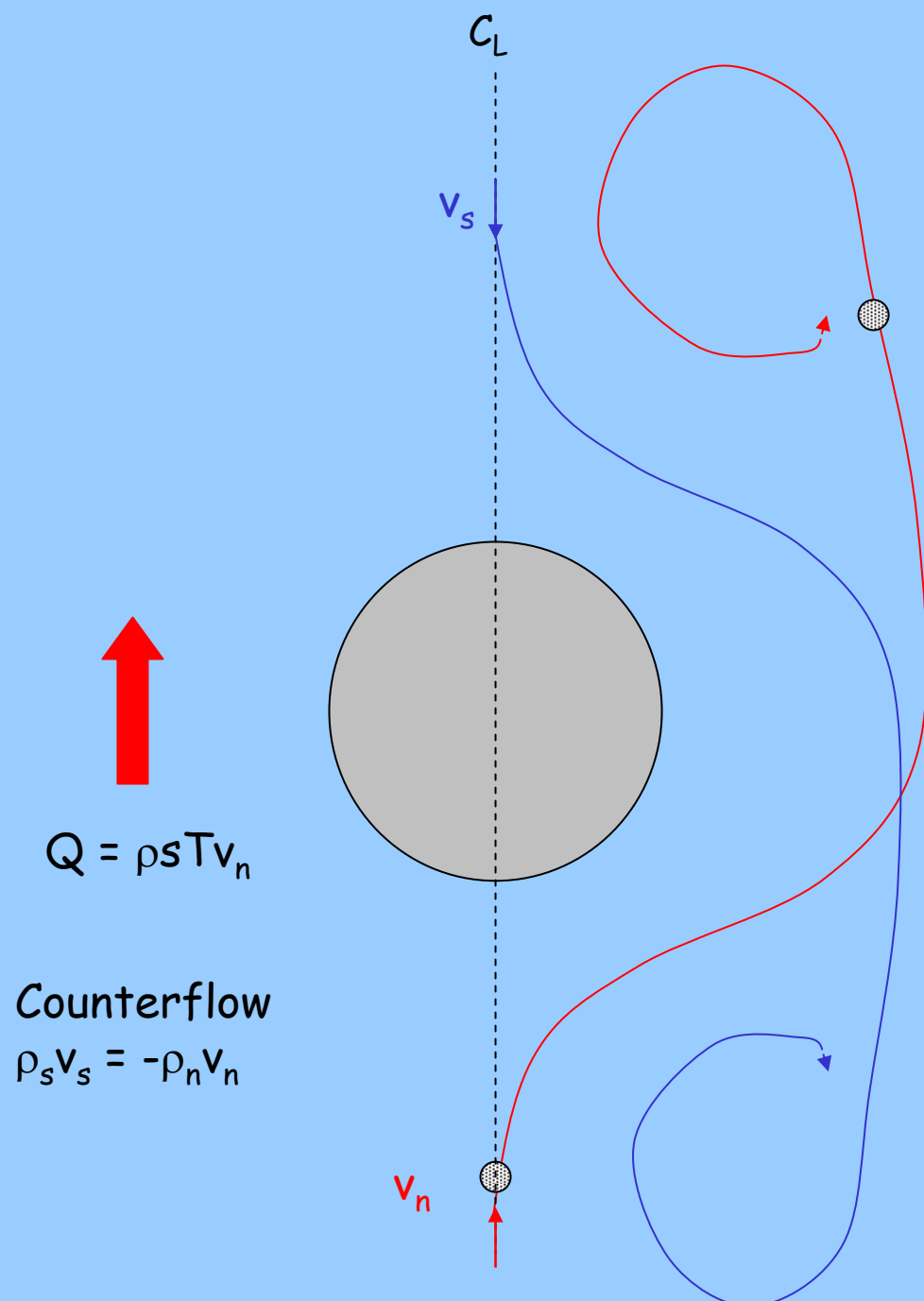
streamline profile

$T = 2.03 \text{ K}$ ,  $q = 7.2 \text{ kW/m}^2$ ,  $Re = 4185$

# Are these results repeatable?

Example:  $T = 2.03 \text{ K}$ ,  $q = 7.2 \text{ kW/m}^2$ ,  $Re = 13697$





# Discussion/Summary

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- First successful PIV experiments with He II counterflow including 2<sup>nd</sup> sound and heat diffusion
- $v_p < v_n$  for all measurements suggesting interactions in addition to the normal fluid viscous drag
- Flow over cylinder shows large scale vorticity behind and in front of cylinder
- Future work will focus on:
  - Quantitative analysis of particle velocity
  - What does the vorticity suggest about thermal fluctuations?
  - Flow over other configurations (plates, orifice, etc.)
  - Fundamental understanding in terms of two fluid model

